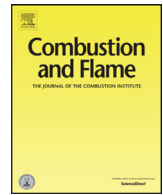




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Effective Lewis number and burning speed for flames propagating in small-scale spatio-temporal periodic flows

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ABSTRACT

Propagation of premixed flames having thick reaction zones in rapidly-varying, small-scale, zero-mean, spatio-temporal periodic flows is considered. Techniques of large activation energy asymptotics and homogenization theory are used to determine the effective Lewis number Le_{eff} and the effective burning speed ratio S_T/S_L , which are influenced by the flow through flow-enhanced diffusion. The resultant effective diffusivity matrix is, in general, neither a scalar nor a diagonal matrix and therefore induces anisotropic effects on the propagation of multi-dimensional flames. As the flow Peclet number Pe becomes large, the flow-enhanced fuel diffusion coefficient and the thermal diffusivity behave respectively like $(PeLe)^\sigma$ and Pe^σ , where Le is the Lewis number and $\sigma \leq 2$ is a constant which depends on the flow and the direction of flame propagation. The maximal value $\sigma = 2$ is achieved for steady, unidirectional, spatially periodic shear flows, while for steady two-dimensional square vortices, we have $\sigma = 1/2$. In general, the constant σ is determined by solving a linear partial differential equation. The scaling laws for the diffusion coefficients lead to corresponding scaling laws for the effective Lewis number and the effective burning speed ratio of the form $Le_{\text{eff}} \simeq Le^{1-\sigma}$ and $S_T/S_L \sim (Pe/Le)^{\sigma/2}$. Effects of thermal expansion and volumetric heat loss on the flame are also briefly discussed. In particular, it is shown that the quenching limit is enlarged by a factor $1/Le^\sigma$ for $Le < 1$ and diminished by the same factor for $Le > 1$, due to the flow-enhanced diffusion. The potential implications of the results to better understand turbulent combustion are discussed. A special emphasis is placed on the dependence of the flame on Le in the presence of high-intensity, small-scale flows. In particular, it is shown that this dependence is intimately linked to the flow through Taylor-dispersion like enhanced diffusion, rather than through the traditional molecular diffusion coupled with curvature effects. The flow-dependent effective Lewis number identified may also provide an explanation to the peculiar experimental observation that turbulence appears to facilitate ignition in $Le > 1$ mixtures and to inhibit it in $Le < 1$ mixtures.

Novelty and significance statement

An original study, combining asymptotic analysis and homogenization theory, is applied to describe flame propagation in small-scale, spatio-temporal periodic flow fields. Scaling laws are derived for the effective burning speed and the effective Lewis number for high-intensity small-scale flows, which are useful to better understand the behaviour of turbulent premixed flames in the distributed reaction zone regime. The formula for the flow-dependent effective Lewis number identified herein may explain the peculiar experimental observation that turbulence appears to facilitate ignition in $Le > 1$ mixtures and to inhibit it in $Le < 1$ mixtures. The high-intensity small-scale flows are shown to increase the quenching limit due to volumetric heat losses in $Le < 1$ mixtures and decrease it in $Le > 1$ mixtures.

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1. Introduction

In this paper, we study the problem of flame propagation in spatially or more generally spatio-temporally periodic flows having single length and time scales. A number of theoretical [1–4] and computational [5–10] investigations have been devoted to this problem in the past and more recently in [11,12]. Experimental studies have also addressed this problem, notably in order to gain insight into flame propagation in the presence of Taylor-Couette vortices [13–15]. One of the main motivations of these and similar studies has been to improve our understanding of premixed turbulent combustion. The reader is referred to specialized reviews such as [16,17] for an overview of the main issues in the vast field of turbulent combustion. Here, we simply note that most theoretical works have focused primarily on the thin flame or thin reaction zone regimes, describing flame propagation in a large-scale flow field. On the other hand, the effect of small-scale flows is ubiquitous in turbulent combustion, notably in the distributed reaction zone regime [16,18] where some flow scales can become smaller than the size of the reaction zone.

The current paper focuses on the thick reaction-zone limit [19] to elucidate the influence of small-scale periodic flows on flame propagation. Particular attention is devoted to characterising the effective Lewis number and burning speed in such flows. The focus of the study is partly motivated by apparent disagreement revived recently regarding the effective Lewis number, Le_{eff} in strongly turbulent flows. Specifically, whereas according to common views [16,20–22], Le_{eff} should be unity in such conditions, recent studies [23–26] suggest otherwise. Notably, it is argued in [23] that the molecular Lewis number effects are still important in strongly turbulent flows and are most active at scales small compared with the flame thickness. Although our laminar periodic flow model cannot fully settle the disagreement regarding Le_{eff} in turbulent combustion, it can provide a helpful insight by determining Le_{eff} for the small-scale laminar flows considered. This problem is treated analytically in the limit of large activation energy, with the reaction zone thickness being larger than the flow length scale.

The paper is organized as follows. The characteristic scales involved in thick reaction-zone flames propagating in small-scale periodic flows are introduced in Section 2. The problem governing equations and boundary conditions are then formulated in Section 3, and these are the basis of an asymptotic analysis carried out in Section 4. The scaling laws for the effective burning speed and the effective Lewis number are obtained in Section 5 for large values of the Peclet number. The results are illustrated for two classes of prototypical flow fields, namely for unsteady unidirectional flows and for the so-called Childress-Soward flows which are steady and two dimensional. Potential implications for premixed turbulent combustion are discussed in Section 5.3. The results of Section 5 are complemented by selected illustrative results for arbitrary Peclet numbers in Section 6. Further extensions of the work including the effect of thermal expansion, heat loss are briefly discussed in Section 7 and Section 8 and followed by conclusions in Section 9.

2. Scalings for thick reaction-zone flames

Consider in a reactive mixture a spatio-temporal periodic flow field $\mathbf{v}(\mathbf{x}, t)$, where \mathbf{x} is the dimensional position vector and t the dimensional time. Let the characteristic flow amplitude, spatial period and temporal period be, U , l_{cell} and t_{cell} , respectively. Further, let us also assume that the mean value of $\mathbf{v}(\mathbf{x}, t)$ is zero in a suitable frame of reference. Then, the heat transport process may be characterised by two dimensionless numbers, namely, the Peclet

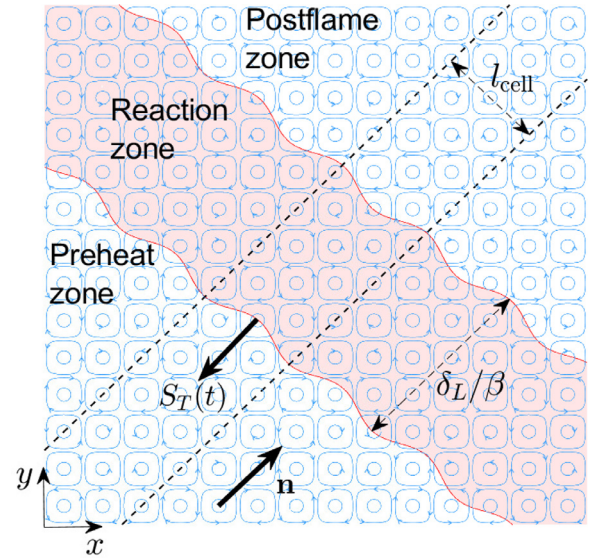


Fig. 1. Schematic illustration of the flame structure in the thick reaction-zone limit, $\delta_L/\beta \sim l_{\text{cell}}$ [19]. The flame is assumed to be periodic (or independent of) directions perpendicular to \mathbf{n} . Here, we shall consider the ultra-thick reaction-zone regime wherein $\delta_L/\beta \gg l_{\text{cell}}$.

number Pe and the Stokes number St , defined by

$$Pe = \frac{U l_{\text{cell}}}{D_T}, \quad St = \frac{l_{\text{cell}}^2/D_T}{t_{\text{cell}}},$$

where D_T denotes the thermal diffusivity of the gas mixture.

The thermal and chemical properties of the reactive mixture define a laminar burning speed S_L and a laminar flame thickness $\delta_L = D_T/S_L$. The thickness of the reaction zone is then given by δ_L/β , where β is the Zeldovich number; these quantities are defined below in (3). By comparing the flow scale with the reaction-zone thickness, combustion modes can be classified into three regimes [19], namely a thin reaction-zone ($\delta_L/\beta \ll l_{\text{cell}}$), a thick reaction-zone ($\delta_L/\beta \sim l_{\text{cell}}$) and an ultra-thick reaction-zone ($\delta_L/\beta \gg l_{\text{cell}}$) regimes. The thin reaction-zone regime includes both thin flames ($\delta_L \ll l_{\text{cell}}$) and moderately thick flames ($\delta_L \sim l_{\text{cell}}$) [27]. A schematic illustration of the flame structure in the thick reaction-zone limit is shown in Fig. 1. In the current paper, we shall focus on the ultra-thick reaction-zone regime where $\delta_L \gg \delta_L/\beta \gg l_{\text{cell}}$.

Assume that the flame structure propagates in the periodic flow field in a definite direction, say $-\mathbf{n}$, with some propagation speed. This is justified if the structure is periodic in (or, independent of) directions perpendicular to \mathbf{n} , as we shall assume. Furthermore, since we consider zero-mean flows, the flame propagation speed is also the effective burning speed $S_T(t)$, in the first approximation. Specifically, the function $S_T(t)$ (in general, periodic in t) can be defined as the total instantaneous burning rate per unit cross-sectional area normal to \mathbf{n} of the infinite strip depicted in Fig. 1. Numerical computations of flame propagation are typically performed only for such infinitely long strips [5,7–9].

As mentioned above, the present study deals with the limit $\delta_L \gg \delta_L/\beta \gg l_{\text{cell}}$. This requirement implies that for fixed Peclet number

$$\frac{S_T}{U} \sim \frac{S_L}{U} = \frac{\epsilon}{Pe} \ll 1 \quad \text{since} \quad \epsilon \equiv \frac{l_{\text{cell}}}{\delta_L} \ll 1.$$

That is to say, the burning speed is small when compared to the flow amplitude, as it is the case for sufficiently thick flames. In addition, in the case of time-dependent flows, the homogenization analysis below requires that $\delta_L/\beta S_L \gg t_{\text{cell}}$, where $\delta_L/\beta S_L$ is the residence time in the reaction zone.

Under the assumptions above, the flame can be regarded in the first approximation as being planar on the flame (large) scale $\mathbf{x} \sim \delta_L$, whereas it will be non-planar on the flow (small) scale $\mathbf{x} \sim l_{\text{cell}}$. The burning speed S_T will be a time-independent constant in the first approximation and involve corrections of order ϵ in the following approximation.

The effect of a rapidly-varying small-scale motion is known to manifest as a diffusion process on the large scale. The homogenization theory can be used to quantify the flow-enhanced diffusion analytically by taking advantage of the separation of scales between the flame and the flow. An excellent review of this technique, applied to a non-reactive scalar field, has been provided by Majda and Kramer [28]. We shall employ this technique to our flame propagation problem, following [28] closely.

3. Governing equations

It is advantageous to adopt a reference frame that is moving with the flame. The time and space coordinates are non-dimensionalized using the flow scales,

$$\tau = \frac{tU}{l_{\text{cell}}}, \quad \xi = \frac{1}{l_{\text{cell}}} \left(\mathbf{x} + \mathbf{n} \int_0^t S_T dt \right) = \frac{\mathbf{x}}{l_{\text{cell}}} + \frac{\epsilon \mathbf{n}}{Pe} \int_0^\tau S d\tau, \quad (1)$$

where $S \equiv S_T/S_L$ is the ratio of effective burning speed to the laminar flame speed. Therefore (ξ, τ) are appropriate independent variables for the small-scale flow field, but not for the thick flame. From a small-scale viewpoint, the coordinate shift $\xi - \mathbf{x}/l_{\text{cell}}$ between the laboratory frame and the flame-fixed frame is negligible at leading order according to (1) since $\epsilon \ll 1$. In the frame attached to the flame, the laboratory-frame vector field $\mathbf{v}(\mathbf{x}/l_{\text{cell}}, t/t_{\text{cell}})$ transforms into

$$\mathbf{u}(\xi, \tau) \equiv \frac{1}{U} \mathbf{v} \left(\frac{\mathbf{x}}{l_{\text{cell}}}(\xi, \tau), \frac{\tau S t}{Pe} \right)$$

after scaling with U . In the limit $\epsilon \rightarrow 0$, the right-hand side of this equation may be expanded in a Taylor series as

$$\mathbf{u}(\xi, \tau) = \frac{\mathbf{v}}{U} - \frac{\epsilon S_0 \tau}{UPe} \mathbf{n} \cdot \nabla_\xi \mathbf{v} + \dots \quad (2)$$

where $\mathbf{v}, \nabla_\xi \mathbf{v}, \dots$ are evaluated at (ξ, τ) and S_0 denotes the leading-order value of S .

The unburnt reacting mixture is assumed to be fuel lean, whose combustion chemistry is modeled by a single-step irreversible Arrhenius reaction with the fuel burning rate (in mass units) per unit volume given by $\rho B Y_F e^{-E/RT}$, that involves the pre-exponential factor B , the gas density ρ , the fuel mass fraction Y_F , the temperature T , the activation energy E and the universal gas constant R . Further, for simplicity, we shall adopt the thermo-diffusive approximation in which density and molecular diffusivities are constant and briefly discuss the effect of variable density later. Also, we introduce the Zeldovich number β , heat release parameter α and the laminar flame speed S_L (for $\beta \gg 1$) by

$$\beta = \frac{E(T_{ad} - T_u)}{RT_{ad}^2}, \quad \alpha = \frac{T_{ad} - T_u}{T_{ad}}, \quad S_L = (2Le\beta^{-2}BD_T e^{-E/RT_{ad}})^{1/2}. \quad (3)$$

In these expressions, Le denotes the Lewis number, T_u the unburnt gas temperature and $T_{ad} = T_u + qY_{F,u}/c_p$ the adiabatic flame temperature, where q is the heat release rate per unit mass of fuel burnt, $Y_{F,u}$ is the fuel mass fraction in the unburnt mixture and c_p is the constant-pressure specific heat.

The scaled fuel mass fraction and temperature are defined by

$$y_F = \frac{Y_F}{Y_{F,u}}, \quad \theta = \frac{T - T_u}{T_{ad} - T_u}.$$

The non-dimensional governing equations in a frame attached to the flame are given by

$$Pe \frac{\partial y_F}{\partial \tau} + (\epsilon S \mathbf{n} + Pe \mathbf{u}) \cdot \nabla_\xi y_F = \frac{1}{Le} \nabla_\xi^2 y_F - \epsilon^2 \omega(y_F, \theta), \quad (4)$$

$$Pe \frac{\partial \theta}{\partial \tau} + (\epsilon S \mathbf{n} + Pe \mathbf{u}) \cdot \nabla_\xi \theta = \nabla_\xi^2 \theta + \epsilon^2 \omega(y_F, \theta) \quad (5)$$

where

$$\omega(y_F, \theta) = \frac{\beta^2 y_F}{2Le} \exp \left[\frac{\beta(\theta - 1)}{1 + \alpha(\theta - 1)} \right].$$

The boundary conditions for y_F and θ need to be prescribed in terms of the large scale variable \mathbf{x}/δ_L , i.e., $\epsilon \xi$. They are given by

$$\epsilon \xi \cdot \mathbf{n} \rightarrow -\infty : \quad y_F = 1, \quad \theta = 0, \quad (6)$$

$$\epsilon \xi \cdot \mathbf{n} \rightarrow +\infty : \quad y_F = 0, \quad \epsilon^{-1} \nabla_\xi \theta \cdot \mathbf{n} = 0 \quad (7)$$

in the direction of \mathbf{n} . Periodicity conditions are imposed in other spatial directions and in time.

4. Asymptotic analysis in the double limit $\epsilon \rightarrow 0, \beta\epsilon \rightarrow 0$

In this section, we carry out an asymptotic analysis of the problem (4)–(7) in the double limit $\epsilon \rightarrow 0, \beta\epsilon \rightarrow 0$. At leading order, the flame structure is steady and one dimensional on the large scale $\mathbf{X} \equiv \epsilon \xi \sim 1$. To describe this structure on this scale, we use the multiple-scale technique, involving the small-scale coordinates (ξ, τ) and the large-scale coordinate \mathbf{X} . Consequently, derivatives transform according to

$$\nabla_\xi \rightarrow \nabla_\xi + \epsilon \nabla_X$$

where ∇_ξ and ∇_X are the gradient operators in the small-scale and large-scale coordinates. The appropriate expansion for the solution can be written as

$$y_F = F_0(\mathbf{X}) + \epsilon F_1(\mathbf{X}, \xi, \tau) + \epsilon^2 F_2(\mathbf{X}, \xi, \tau) + \dots,$$

$$\theta = \Theta_0(\mathbf{X}) + \epsilon \Theta_1(\mathbf{X}, \xi, \tau) + \epsilon^2 \Theta_2(\mathbf{X}, \xi, \tau) + \dots,$$

$$S = S_0 + \epsilon S_1(\tau) + \epsilon^2 S_2(\tau) + \dots.$$

Substituting these expansions into (4)–(7) and collecting terms of different orders of ϵ , we obtain a series of equations for F_i, Θ_i and S_i . The equations that arise at leading order are identically satisfied as we have already anticipated that F_0 and Θ_0 are independent of small-scale variables. The equations at the next two orders are found to be

$$\mathcal{L}_F F_1 = -PeLe \mathbf{v} \cdot \nabla_X F_0, \quad (8)$$

$$\mathcal{L}_T \Theta_1 = -Pe \mathbf{v} \cdot \nabla_X \Theta_0, \quad (9)$$

$$\begin{aligned} \mathcal{L}_F F_2 = & -S_0 Le [\mathbf{n} - \tau \mathbf{n} \cdot \nabla_\xi \mathbf{v}] \cdot (\nabla_X F_0 + \nabla_\xi F_1) - PeLe \mathbf{v} \cdot \nabla_X F_1 \\ & + \nabla_X^2 F_0 + 2\nabla_\xi \cdot \nabla_X F_1 - Le \omega(F_0, \Theta_0), \end{aligned} \quad (10)$$

$$\begin{aligned} \mathcal{L}_T \Theta_2 = & -S_0 [\mathbf{n} - \tau \mathbf{n} \cdot \nabla_\xi \mathbf{v}] \cdot (\nabla_X \Theta_0 + \nabla_\xi \Theta_1) - Pe \mathbf{v} \cdot \nabla_X \Theta_1 \\ & + \nabla_X^2 \Theta_0 + 2\nabla_\xi \cdot \nabla_X \Theta_1 + \omega(F_0, \Theta_0) \end{aligned} \quad (11)$$

where

$$\mathcal{L}_F \equiv PeLe \frac{\partial}{\partial \tau} + PeLe \mathbf{v} \cdot \nabla_\xi - \nabla_\xi^2, \quad \mathcal{L}_T \equiv Pe \frac{\partial}{\partial \tau} + Pe \mathbf{v} \cdot \nabla_\xi - \nabla_\xi^2 \quad (12)$$

are differential operators that act on small-scale variables.

Solutions for the first-order Eqs. (8)-(9) are obtained by assuming

$$F_1 = F_{1a}(\mathbf{X}) + PeLe \mathbf{K}_F \cdot \nabla_X F_0, \quad (13)$$

$$\Theta_1 = \Theta_{1a}(\mathbf{X}) + Pe \mathbf{K}_T \cdot \nabla_X \Theta_0 \quad (14)$$

where the vectors $\mathbf{K}_F(\xi, \tau)$ and $\mathbf{K}_T(\xi, \tau)$ are the periodic solutions of

$$\mathcal{L}_F \mathbf{K}_F = -\mathbf{v}, \quad \mathcal{L}_T \mathbf{K}_T = -\mathbf{v}. \quad (15)$$

Since \mathbf{v} is of the form $\mathbf{v} = \mathbf{v}(\xi, \tau St/Pe)$, the function \mathbf{K}_F depends only on the two parameters $PeLe$ and $StLe$ while, similarly, \mathbf{K}_T only depends on Pe and St .

The second-order non-homogeneous Eqs. (10)-(11) governing F_2 and Θ_2 are solvable only if the right-hand sides satisfy a solvability condition. Specifically, following [28], the solvability condition states that given a periodic function $f(\xi, \tau)$, the equation $\mathcal{L}_F g(\xi, \tau) = f(\xi, \tau)$ has a smooth periodic solution if and only if $f(\xi, \tau)$ has zero mean. Imposing this condition on Eq. (11), for instance, we obtain

$$\langle -S_0 [\mathbf{n} - \tau \mathbf{n} \cdot \nabla_\xi \mathbf{v}] \cdot (\nabla_X \Theta_0 + \nabla_\xi \Theta_1) - Pe \mathbf{v} \cdot \nabla_X \Theta_1 + \nabla_X^2 \Theta_0 + 2 \nabla_\xi \cdot \nabla_X \Theta_1 + \omega \rangle = 0$$

where $\langle \cdot \rangle$ denotes an average¹ over the small scale variables ξ and τ .

Clearly, terms such as $-S_0 \mathbf{n} \cdot \nabla_X \Theta_0 + \nabla_X^2 \Theta_0 + \omega$ that do not depend on the small-scale variables are unaffected by the averaging operation. On the other hand, all terms that contain ∇_ξ can be shown, using the divergence theorem and the periodicity boundary condition, to vanish identically. The only remaining contribution is due to $Pe \mathbf{v} \cdot \nabla_X \Theta_1$ in which only the second term in (14) survives upon averaging. This particular contribution $Pe \langle \mathbf{v} \cdot \nabla_X \Theta_1 \rangle = Pe^2 \langle \mathbf{v} \cdot \nabla_X (\mathbf{K}_T \cdot \nabla_X \Theta_0) \rangle$ can be written as $Pe^2 \nabla_X \cdot (\langle \mathbf{v} \mathbf{K}_T \rangle \cdot \nabla_X \Theta_0)$ or, equivalently $Pe^2 \langle \mathbf{v} \mathbf{K}_T \rangle : \nabla_X \nabla_X \Theta_0$ since \mathbf{v} is independent of \mathbf{X} and the averaging operation affects only the small-scale variables. Furthermore since the Hessian matrix $\nabla_X \nabla_X$ which represents the tensor $\partial^2 / \partial x_i \partial x_j$ is clearly symmetric, we may symmetrise the product $\mathbf{v} \mathbf{K}_T$.

Now, on applying the solvability conditions for both dependent variables and simplifying the results, we obtain

$$S_0 \mathbf{n} \cdot \nabla_X F_0 = \frac{1}{Le} \nabla_X \cdot (\mathcal{D}_F \cdot \nabla_X F_0) - \omega(F_0, \Theta_0), \quad (16)$$

$$S_0 \mathbf{n} \cdot \nabla_X \Theta_0 = \nabla_X \cdot (\mathcal{D}_T \cdot \nabla_X \Theta_0) + \omega(F_0, \Theta_0) \quad (17)$$

where the effective diffusion matrices $\mathcal{D}_F = \mathcal{D}_F(PeLe, StLe)$ and $\mathcal{D}_T = \mathcal{D}_F(Pe, St)$ are given² by

$$\mathcal{D}_F = \mathbf{I} - \frac{1}{2} Pe^2 Le^2 \langle \mathbf{v} \mathbf{K}_F + (\mathbf{v} \mathbf{K}_F)^T \rangle = \mathbf{I} + Pe^2 Le^2 \langle \nabla_\xi \mathbf{K}_F \circ (\nabla_\xi \mathbf{K}_F)^T \rangle \quad (18)$$

$$\mathcal{D}_T = \mathbf{I} - \frac{1}{2} Pe^2 \langle \mathbf{v} \mathbf{K}_T + (\mathbf{v} \mathbf{K}_T)^T \rangle = \mathbf{I} + Pe^2 \langle \nabla_\xi \mathbf{K}_T \circ (\nabla_\xi \mathbf{K}_T)^T \rangle \quad (19)$$

in which \mathbf{I} denotes the identity matrix and the symbol \circ represents element-wise matrix multiplication such that, for example, $D_{T,ij} = \delta_{ij} + Pe^2 \langle \nabla_\xi K_{T,i} \cdot \nabla_\xi K_{T,j} \rangle$.

While formulas (18)-(19) for the effective diffusion matrices are of general use in a variety of problems such as problems involving

propagation and stability, they are more transparent if an axis of the coordinate system is chosen to be along \mathbf{n} . Let \mathcal{R} be the rotation matrix which transforms the original coordinate vector \mathbf{X} into a new coordinate vector $\mathbf{X}' = \mathcal{R}\mathbf{X}$ such that the X'_1 -axis is directed along \mathbf{n} . In the rotated coordinate system, $F_0(X'_1)$ and $\Theta_0(X'_1)$ are function only of X'_1 and therefore Eqs. (16)-(17) simplify to

$$S_0 \frac{dF_0}{dX'_1} = \frac{D'_{F,11}}{Le} \frac{d^2 F_0}{dX'^2_1} - \omega(F_0, \Theta_0), \quad (20)$$

$$S_0 \frac{d\Theta_0}{dX'_1} = D'_{T,11} \frac{d^2 \Theta_0}{dX'^2_1} + \omega(F_0, \Theta_0) \quad (21)$$

where $D'_F = \mathcal{R} \mathcal{D}_F \mathcal{R}^T$ and $D'_T = \mathcal{R} \mathcal{D}_T \mathcal{R}^T$. Also, the boundary conditions (6)-(7) reduce to

$$X'_1 \rightarrow -\infty : F_0 - 1 = \Theta_0 = 0 \quad \text{and} \quad X'_1 \rightarrow +\infty : F_0 = \frac{d\Theta_0}{dX'_1} = 0. \quad (22)$$

The solution of problem (20)-(22) for large β , as done e.g. in [29], provides the following formulas for the effective burning speed and the effective Lewis number

$$\frac{S_T}{S_L} \simeq S_0 = \frac{D'_{T,11}}{\sqrt{D'_{F,11}}} = \sqrt{\frac{Le_{\text{eff}}}{Le} D'_{T,11}} \quad \text{and} \quad Le_{\text{eff}} = Le \frac{D'_{T,11}}{D'_{F,11}}. \quad (23)$$

Note that the dependence of S_0 on $D'_{T,11}$ and $D'_{F,11}$ is intuitively correct as it extends the dependence of the laminar speed S_L on D_T and D_F in (3), namely $S_L \propto D_T / \sqrt{D_F}$, by replacing D_T and D_F with corresponding flow-enhanced values.

5. Scaling laws for large Peclet numbers

The dependence of the effective burning speed ratio S_T/S_L and the effective Lewis number Le_{eff} on Pe is determined by the enhanced diffusion coefficients, as indicated in (23). For large values of Pe , we may assume that the asymptotic behaviour of the effective diffusion coefficients is of the form

$$D'_{F,11} - 1 \sim (PeLe)^\sigma, \quad D'_{T,11} - 1 \sim Pe^\sigma \quad \text{for} \quad Pe \gg 1, \quad (24)$$

and similar behaviours (with different exponents σ) for other elements of the matrix $\mathcal{D}_F - \mathbf{I}$. Then (23) implies that

$$Le_{\text{eff}} \simeq Le^{1-\sigma}, \quad \frac{S_T}{S_L} \sim \left(\frac{Pe}{Le} \right)^{\sigma/2}, \quad (25)$$

provided the exponent σ , which needs to be computed as done below, is positive. Negative values of σ indicate that there is no enhancement of diffusion with respect to molecular diffusion for $Pe \gg 1$ and therefore

$$Le_{\text{eff}} = Le, \quad \frac{S_T}{S_L} = 1.$$

In fact, as demonstrated in [28], σ is bounded from above, namely $\sigma \leq 2$. This upper bound indicates that maximal enhancement of diffusion is achieved when $\sigma = 2$. The maximal enhancement in fact occurs for steady, unidirectional periodic (or confined) shear flows and is associated with the Taylor's dispersion mechanism [29]. It is instructive to consider two classes of flow fields, namely, unsteady unidirectional flows for which the exponent σ can be determined explicitly and the two-dimensional so-called Childress-Soward flows for which σ will be computed numerically.

¹ The average is defined by $\langle \varphi \rangle \equiv \frac{St}{Pe} \int_0^{Pe/St} \int_0^1 \int_0^1 \int_0^1 \varphi d\xi_1 d\xi_2 d\xi_3 d\tau$.
² The equality of the second relation to the first is shown readily in index notation [28]: substitute $v_i = \mathcal{L}_F K_{F,i}$ to obtain $\frac{1}{2} \langle v_i K_{F,j} + v_j K_{F,i} \rangle = \frac{1}{2} \langle K_{F,j} \mathcal{L}_F K_{F,i} + K_{F,i} \mathcal{L}_F K_{F,j} \rangle$, which can be re-written, using (12), as $\frac{1}{2} \langle \mathcal{L}_F (K_{F,i} K_{F,j}) + 2 \nabla_\xi K_{F,i} \cdot \nabla_\xi K_{F,j} \rangle$ and then impose the solvability condition for the first term $\langle \mathcal{L}_F (K_{F,i} K_{F,j}) \rangle = 0$.

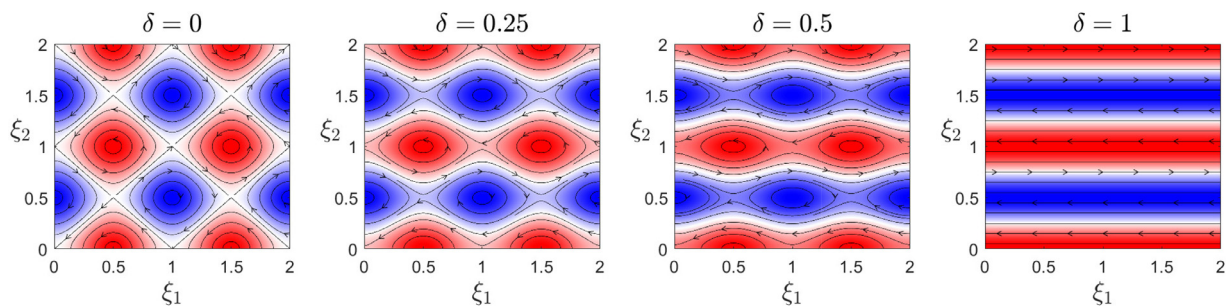


Fig. 2. Streamline plots for the Childress-Soward flows (27) for selected values of δ . The colours describe the scalar vorticity field (normalized by its maximum value with red indicating a positive or counter-clockwise vorticity and blue a negative or clockwise vorticity). (For interpretation of the references to colour in this figure legend, the reader is referred to the web version of this article.)

5.1. Unsteady unidirectional flows

Diffusion enhancement has been studied in the context of unsteady unidirectional periodic shear flows by Zeldovich [30], who provided an exact solution for the effective diffusion coefficient. The similar diffusion problem in confined geometries has been studied by Watson [31]. The general unidirectional shear flow periodic in time and space with zero mean may be written in the form of a double Fourier series as

$$v_1(\xi_2, \tau) = \sum_{(k,m) \neq (0,0)} \hat{v}_{k,m} e^{2\pi i(k\xi_2 + m\tau/Pe)}, \quad v_2 = v_3 = 0.$$

Using this expression in (15) to determine \mathbf{K}_F and then using (18) to determine \mathcal{D}'_F , we find that the only non-zero element of $\mathcal{D}'_F - \mathbf{I}$ is $\mathcal{D}'_{F,11} - 1$. This term is given by

$$\mathcal{D}'_{F,11} - 1 = \sum_{(k,m) \neq (0,0)} \frac{(kPeLe)^2 |\hat{v}_{k,m}|^2}{4\pi^2 k^4 + (mStLe)^2}, \quad (26)$$

a result which is equivalent to formula (55) of [28], which generalizes an earlier formula originally derived by Zeldovich [30]. It is worth noting that when $St \sim Pe$, i.e., when the time scale t_{cell} corresponds to l_{cell}/U , $\mathcal{D}'_{F,11} - 1$ tends to a constant independent of Pe as $Pe \rightarrow \infty$. This indicates that the exponent $\sigma = 0$ and that the enhancement of diffusion remains bounded as noted by Zeldovich. On the other hand, when $St = 0$ corresponding to a steady flow, or more generally when $St \ll Pe$ corresponding to a quasi-steady flow, formula (26) indicates that $\mathcal{D}'_{F,11} - 1 \sim (PeLe)^2$, which is the maximal enhancement aforementioned. For $St \gg Pe$, diffusion enhancement is negligible according to (26).

5.2. Steady two-dimensional Childress-Soward flows

In addition to the unidirectional flows, another prototypical flow which has been used in theoretical studies such as [1–4] on flame-flow interaction is the so-called vortical (or cellular flow). A useful class of simple steady flows depending on a parameter $0 \leq \delta \leq 1$ which encompasses both the shear and cellular flows is the so-called Childress-Soward flows [32]. The velocity components of the Childress-Soward flow in a suitable coordinate system are given by

$$v_1 = -(1 + \delta) \sin(2\pi \xi_2), \quad v_2 = -(1 - \delta) \sin(2\pi \xi_1), \quad v_3 = 0. \quad (27)$$

As shown by the streamline plots in Fig. 2, we have a cellular flow consisting of square vortices for $\delta = 0$ and a unidirectional shear flow directed along ξ_1 -axis for $\delta = 1$, with intermediate values of δ representing a series of cats-eye vortices with varying degrees of eddy-like/shear motion.

For the Childress-Soward flows (in the frame of reference chosen) given by (27), $\mathcal{D}'_T - \mathbf{I}$ is a diagonal matrix. Corresponding to

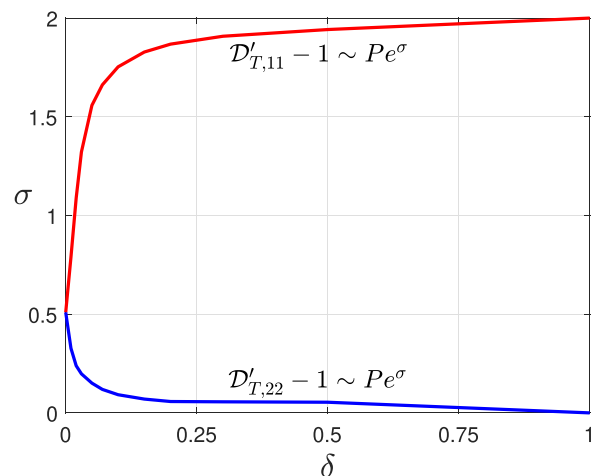


Fig. 3. The exponents σ in the formula $\mathcal{D}'_{T,11} - 1 \sim Pe^\sigma$ (red line) and in the formula $\mathcal{D}'_{T,22} - 1 \sim Pe^\sigma$ (blue line) vs. the parameter δ appearing in (27), obtained by fitting numerical computations for Pe in the range $[6 \times 10^3, 10^4]$. (For interpretation of the references to colour in this figure legend, the reader is referred to the web version of this article.)

these flows, we now calculate numerically the exponent σ for each diagonal entry of the matrix $\mathcal{D}'_T - \mathbf{I}$. For $\mathcal{D}'_{T,11} - 1$ we first solve (15) for \mathbf{K}_T , then evaluate \mathcal{D}'_T using (19) and then finally fit the data for large Pe to the profile

$$\mathcal{D}'_{T,11} - 1 \sim Pe^\sigma. \quad (28)$$

We proceed similarly to determine the exponent σ for $\mathcal{D}'_{T,22} - 1$. The numerical results are obtained by solving the inhomogeneous elliptic PDEs (15) using COMSOL Multiphysics software. The equations are solved subject to periodic boundary conditions along with an additional condition, say $\mathbf{K}_T(0, 0) = 0$. The latter condition is needed since the solution is unique only within an additive constant. For the empirical fit, we have used the numerical results corresponding to Pe in the range $[6 \times 10^3, 10^4]$. It should be cautioned that the accuracy in determining the exponent σ depends on its value; the larger the value of σ , the better is the fitting accuracy.

The computed exponents σ corresponding to the two diagonal elements are plotted in Fig. 3 as a function of δ . It can be observed from the figure that when $\delta = 0$, $\sigma = 1/2$ in agreement with the predictions of past investigations [33,34] on square vortices. Similarly for $\delta = 1$ corresponding to a unidirectional shear flow, it is seen that the enhancement of diffusion is in the ξ_1 -direction only, in line with the conclusions of the previous subsection.

We note that as δ is increased, the enhancement of diffusion increases in the ξ_1 -direction and decreases in the other direction. This translates into the effective diffusion process becoming more anisotropic as δ is increased. For $\delta = 0$, there is no anisotropy in

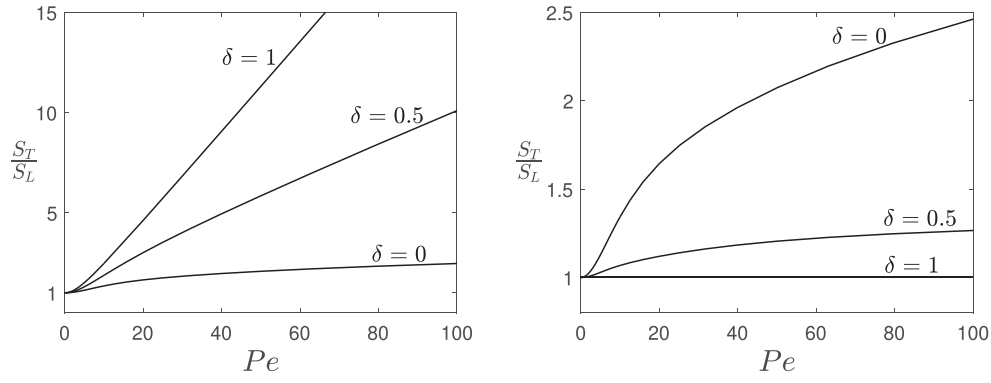


Fig. 4. The effective burning speed ratio S_T/S_L vs. the Peclet number Pe for the Childress-Soward flows (27) with $Le = 1$. The left figure corresponds to flame propagation along the ξ_1 -axis and the right figure to the ξ_2 -axis.

diffusion as both exponents in the figure are equal, while maximum anisotropy is achieved when $\delta = 1$. This latter case for which the maximum value of σ is achieved and corresponds to the ξ_1 direction, the diffusion enhancement is attributable to the well-known Taylor dispersion mechanism [35]. The relatively smaller diffusion enhancement when $\delta = 0$ may be explained by the fact that although convective transport of a scalar can be quick within a given eddy, the transport to an adjacent eddy is still predominantly controlled by the slow molecular diffusion.

Using in (25) the exponents computed in Fig. 3, we can determine the asymptotic behaviours of the effective Lewis number and the effective burning speed ratio. In the case of square vortices ($\delta = 0$), these are given by

$$Le_{\text{eff}} \simeq Le^{1/2}, \quad \frac{S_T}{S_L} \sim \left(\frac{Pe}{Le}\right)^{1/4} = \frac{1}{Le^{1/4}} \left(\frac{U}{S_L}\right)^{1/4} \left(\frac{l_{\text{cell}}}{\delta_L}\right)^{1/4}, \quad (29)$$

irrespective of the direction of flame propagation due to the isotropy of the effective diffusion process. As for the case of unidirectional shear ($\delta = 1$), we have

$$Le_{\text{eff}} \simeq \frac{1}{Le}, \quad \frac{S_T}{S_L} \sim \frac{Pe}{Le} = \frac{1}{Le} \frac{U}{S_L} \frac{l_{\text{cell}}}{\delta_L} \quad (30)$$

when the flame propagates in the direction of the shear flow (ξ_1 -direction). Of course, if we consider flame propagation in the ξ_2 -direction where there is no diffusion enhancement, then

$$Le_{\text{eff}} = Le, \quad \frac{S_T}{S_L} = 1.$$

The behaviours of other values of δ , lie between the above two limiting cases $\delta = 0$ and $\delta = 1$ just considered. It should be noted that the $\frac{1}{4}$ th power dependence of S_T/S_L on Pe given in (29) for the case of square vortices was first identified by Audoly et al. [4] and later confirmed in [8,36,37]; see also [38, pp.186–187]. On the other hand, the dependence on Pe and Le in (30) for unidirectional shear flows was first reported in [29].

5.3. Potential implications for turbulent premixed combustion

It is worth comparing the asymptotic behaviours (29)–(30) with the corresponding trend in premixed turbulent combustion in the distributed reaction zone regime. We begin by comparing our results with the burning speed formula

$$\frac{S_T}{S_L} \sim \frac{\sqrt{Re}}{Le} \sim \frac{Re_\lambda}{Le} \quad (31)$$

reported in the recent experimental study [26] on highly turbulent jet flames; here Re is the turbulent Reynolds number and Re_λ is

the Taylor-scale Reynolds number. Now according to Damköhler's second hypothesis [39], the effect of small scale turbulence is to enhance the effective diffusion coefficients and hence the effective burning speed S_T , without altering the flame structure. Therefore, as we argued in [19], we may write

$$\frac{S_T}{S_L} = \sqrt{\frac{Le_{\text{tur}} D_{T,\text{tur}}}{Le D_T}} \quad (32)$$

which follows from using formula (3); here $Le_{\text{tur}} = D_{T,\text{tur}}/D_{F,\text{tur}}$ is the turbulent Lewis number and $D_{T,\text{tur}}$ and $D_{F,\text{tur}}$ are the turbulent (or effective) thermal diffusivity and fuel diffusion coefficient.

Comparing the last two relations, we find

$$Le_{\text{tur}} = \frac{1}{Le} \quad \text{and} \quad \frac{D_{T,\text{tur}}}{D_T} \sim Re. \quad (33)$$

The Lewis number dependence of the turbulent burning speed (31) and the turbulent Lewis number (33) appear to be in better agreement with the predictions of unidirectional shear flow (30) than the square vortices (29). This observation is somewhat surprising as the shear flow lacks more the isotropic aspect of turbulent diffusion coefficients than the cellular flows. Furthermore, we note that it is difficult to reconcile the dependence on the Reynolds number between the turbulent and the laminar flow cases. Yet, it is interesting to note that formulas (31) with (30) are in good agreement if Pe is identified with the Reynolds number Re_λ based on the Taylor microscale (rather than Re).

It is also instructive to examine the dependence of the effective burning speed S_T , rather than S_T/S_L , on the Lewis number Le . In particular, since $S_L \propto \sqrt{Le}$, we have

$$S_T \propto Le^{(1-\sigma)/2} \quad (34)$$

or equivalently $S_T \propto \sqrt{Le_{\text{eff}}}$. From the above relation, we can conclude that the effective burning speed S_T decreases with increasing Lewis number only when $\sigma > 1$. This trend for S_T is also observed in turbulent cases, see e.g. Fig. 3 in [17], which can therefore be explained, in part, by flow-enhanced diffusion.

Irrespective of the complications associated with turbulent combustion, our study highlights a physically important result. Specifically, the study shows that the flow plays a crucial part in determining the effective Lewis number, leading to surprising results such as $Le_{\text{eff}} \simeq 1/Le$ for parallel flows (30) and $Le_{\text{eff}} \simeq Le^{1/2}$ for square vortices (29) at large values of Pe . Such results can provide explanations for unexpected flame behaviours in turbulent or complex flow fields. An example of such unexpected behaviours is the experimental observation reported in [40,41] that turbulence appears to facilitate ignition in $Le > 1$ mixtures and to inhibit it in $Le < 1$ mixtures. Partial explanation to this observation may be

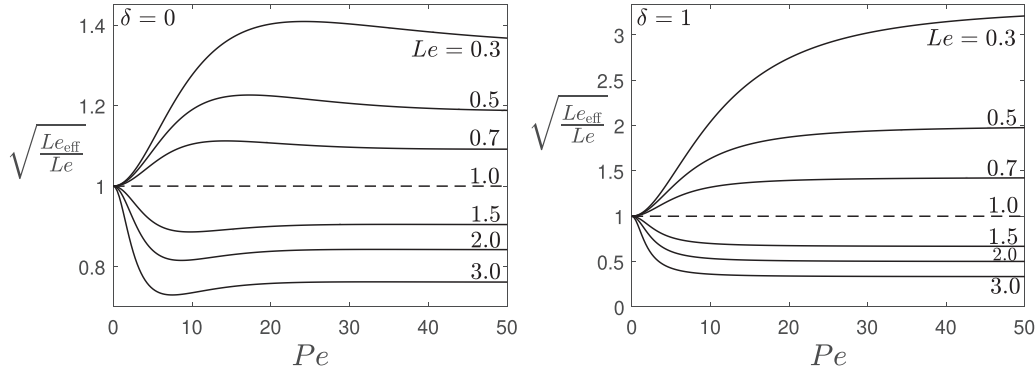


Fig. 5. The factor $\sqrt{Le_{\text{eff}}/Le}$ (36) as a function of Pe for selected values of Le . The left figure corresponds to the cellular flow ($\delta = 0$) and the right figure to the shear flow ($\delta = 1$). In both cases, flame is assumed to propagate in the ξ_1 -direction.

provided by the dependence of Le_{eff} on the flow field emphasized herein.

6. Results for Childress-Soward flows with arbitrary Peclet numbers

In the previous section, we have explored the asymptotic behaviours of the burning speed for large values of the Peclet number. Here we shall present illustrative results for arbitrary values of Pe in the case of Childress-Soward flows (27). We first consider the unity Lewis number case for which formula (23) implies that

$$\frac{S_T}{S_L} \simeq \sqrt{D'_{T,11}} \quad \text{or} \quad \frac{S_T}{S_L} \simeq \sqrt{D'_{T,22}} \quad (35)$$

depending on whether the flame propagates in the ξ_1 -direction or the ξ_2 -direction.

Figure 4 is generated by computing the effective diffusion coefficients in (18)-(19) for different values of Pe and substituting into (35). The curves of S_T/S_L reveal a quadratic dependence on Pe for small values of Pe , whereas at large values they approach the asymptotic behaviour identified in the previous section. In particular, it is worth noting at large values of Pe the linear behaviour for $\delta = 1$ and the sublinear behaviour when $\delta < 1$ which exhibits a bending effect of the curve S_T/S_L vs. Pe .

It is also worth noting that the curve for the periodic shear flow ($\delta = 1$) with flame propagation along ξ_1 -direction may be compared with the corresponding curves reported for Poiseuille flows. Specifically, our findings are consistent with the curves in Figs. 5 and 6 of [42] and figure 8 of [43]. In the case of cellular flows ($\delta = 0$), the findings are found to be consistent with the result exhibited in Fig. 4.32 of [44].

We now examine the influence of non-unity Lewis numbers. To this end, we note that the burning speed ratio S_T/S_L for non-unity Lewis numbers can be obtained according to formula (23) by multiplying the corresponding ratio S_T/S_L for unity Lewis numbers plotted in Fig. 4 by the factor $\sqrt{Le_{\text{eff}}/Le}$. In other words, the factor

$$\sqrt{\frac{Le_{\text{eff}}}{Le}} = \frac{S_T/S_L}{(S_T/S_L)|_{Le=1}} \quad (36)$$

is a convenient way to quantify the departure of the scaled burning speed from its unit Lewis number value. This factor is computed using (18)-(19) and (23) and is plotted as a function of Pe in Fig. 5 for selected values of Le . All curves in this figure are found to exhibit a quadratic behaviour near $Pe = 0$ and asymptote to the value $1/Le^{\sigma/2}$ for large Pe .

7. Effects of thermal expansion and heat loss

Influence of thermal expansion and heat loss can be taken into account in a straightforward manner because the primary change that encountered here is the enhancement of diffusion coefficients. First, let us address thermal expansion effects. It is clear that density variations associated with thermal expansion due to heat release must be in the first approximation a function of \mathbf{X} given by $\rho_0(\mathbf{X})$, as has been shown in the related confined geometry problems [19,29,45,46]. This means that density is practically constant on the small-scale variables (ξ, τ). On account of the density variation on the large scale, the effective diffusion coefficients (18) and (19) will depend on $\rho_0(\mathbf{X})$; see e.g. formulas (21)-(22) in [29]. It follows that the required change in our asymptotic formulas (23) is that the diffusion coefficients need simply to be evaluated at the burnt gas temperature. This implies, for example, that formula (24) need to be replaced with

$$D'_{F,11} - 1 \sim (\mathcal{P}Le)^\sigma \quad \text{where} \quad \mathcal{P} = Pe(1 - \alpha) \quad (37)$$

is the Peclet number involving the gas expansion parameter α defined in (3). The reader is referred to [19] for an analysis that incorporates the thermal expansion in a related simpler problem.

Turning now to the effect of heat loss, let us assume that the heat loss rate per unit volume may be written as $\rho c_p K(T - T_u)$, where K^{-1} is a characteristic cooling time. The scaling of K for flame quenching is given by $K\delta_L^2/D_T \sim 1/\beta$. Thus, we can introduce the parameter $\kappa = \beta K\delta_L^2/D_T$, which introduces on the right-hand side a term $-\epsilon^2\kappa\theta/\beta$ in (5) and correspondingly a term $-\kappa\Theta_0/\beta$ in (17). The classical asymptotic result [47] for the burning speed with account taken of diffusion enhancement then becomes

$$S_0^2 \ln \frac{S_0}{S_{0,ad}} = -\kappa D'_{T,11} \quad (38)$$

where $S_{0,ad}$ is the adiabatic flame speed given by (23). The burning speed S_0 exists for $\kappa \leq \kappa_{\text{ext}}$, where

$$\kappa_{\text{ext}} = \frac{S_{0,ad}^2}{2eD'_{T,11}} = \frac{Le_{\text{eff}}}{2eLe}. \quad (39)$$

For large Peclet numbers with $\sigma > 0$, this formula simplifies to

$$\kappa_{\text{ext}} = \frac{1}{2eLe^\sigma} \quad (40)$$

which indicates that for $Le < 1$, the extinction limit is enlarged by a factor $1/Le^\sigma$ in the presence of the periodic flow, whereas it is diminished by the same amount for $Le > 1$. This effect of diffusion enhancement on flame quenching is greatest when $\sigma = 2$, as identified for Taylor-dispersion controlled flames in [19].

8. Possible extensions of the study

Although, the results derived herein pertain to zero-mean, small-scale periodic flows, they are also applicable if the flows have a small non-zero mean which varies on the large scale. For example, the formulas for the generalized diffusion matrices derived in (18)–(19) are still applicable for the flow field $\mathbf{v}(\mathbf{x}/l_{\text{cell}}, t/t_{\text{cell}})/U + \epsilon \mathbf{V}(\mathbf{x}/\delta_L, tS_L/\delta_L)$ where \mathbf{V} denotes the large-scale weak mean flow. The convection velocity $\epsilon S \mathbf{n} \approx \epsilon S_0 \mathbf{n}$ in (4)–(5), emerging in the flame-fixed frame, is itself a weak mean flow, albeit a constant one at leading order. Excluding some peculiar cases discussed in [28, Section 2.1.3.1], there appears to be no theoretical development for the large-scale mean flow of arbitrary magnitude. The latter problem is also of considerable interest for future investigations.

To close this section, we note that findings of the present paper are applicable, strictly speaking, when $\delta_L/\beta \gg l_{\text{cell}}$. When the flow scale l_{cell} is of the order of, or slightly larger than, the reaction zone thickness δ_L/β , further progress can be made provided $l_{\text{cell}} \ll \delta_L$. This can be done by carrying out an asymptotic analysis in the distinguished limit $\delta_L/\beta \sim l_{\text{cell}}$, as done in [19] for unidirectional flows. In this distinguished limit, the theoretical approach developed here can be applied to the preheat and post flame zones, but for the reaction zone, a convective-diffusive-reactive inner problem is obtained. Specifically, consideration of this distinguished limit shows that the first corrections to the leading-order solutions obtained here are of order $\beta\epsilon = \beta l_{\text{cell}}/\delta_L$.

9. Concluding remarks

In this paper, we have carried out an asymptotic analysis of the propagation of a thick flame in small-scale, zero-mean, spatio-temporal periodic flows. Using activation energy asymptotics and homogenization theory, formulas (23) for the effective Lewis number Le_{eff} and the effective burning speed ratio S_T/S_L have been derived. The formulas quantify the dependence of the propagation and structure of the flame on the flow via flow enhanced diffusion. In particular, when the flow Peclet number Pe is large, the enhanced fuel diffusion coefficient and the enhanced thermal diffusivity are found to grow like $(PeLe)^\sigma$ and Pe^σ , respectively, where $\sigma \leq 2$ is a constant that depends on the flow and the direction of flame propagation. Consequently, this leads to $Le_{\text{eff}} \simeq Le^{1-\sigma}$ and $S_T/S_L \sim (Pe/Le)^{\sigma/2}$; the maximal diffusion enhancement ($\sigma = 2$) is achieved for steady, unidirectional flows. The result also indicates that the effective burning speed $S_T \propto Le^{(1-\sigma)/2}$ increases with decreasing Lewis numbers only when $\sigma > 1$.

We have also briefly addressed the effect of heat loss on flame propagation and quenching in a flow field in Section 7. In particular, it is worth noting that the quenching limit due to heat loss is increased by the factor $1/Le^\sigma$ due to the presence of the flow. That is to say, small-scale flows increase the quenching limit of subunity Lewis-number mixtures and decrease it for mixtures with $Le > 1$.

Finally, the potential implications of the findings to better understand turbulent combustion have been summarized in Section 5.3. A particular aspect which has been emphasised in our study is the dependence of the flame characteristics on the Lewis number in the presence of high-intensity, small-scale flows. This dependence has been shown to intimately depend on the flow through Taylor-dispersion like flow-enhanced diffusion, rather than merely on the conventional molecular diffusion coupled with curvature effects. We have also argued that the flow-dependent effective Lewis number identified herein may be useful to explain the peculiar feature that turbulence appears to facilitate ignition in $Le > 1$ mixtures and to inhibit it in $Le < 1$ mixtures, observed in experiments on ignition in a turbulent reactive flow [40,41].

Declaration of Competing Interest

The authors declare that they have no known competing financial interests or personal relationships that could have appeared to influence the work reported in this paper.

CRediT authorship contribution statement

Prabakaran Rajamanickam: Conceptualization, Formal analysis, Writing – original draft. **Joel Daou:** Conceptualization, Writing – original draft.

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