

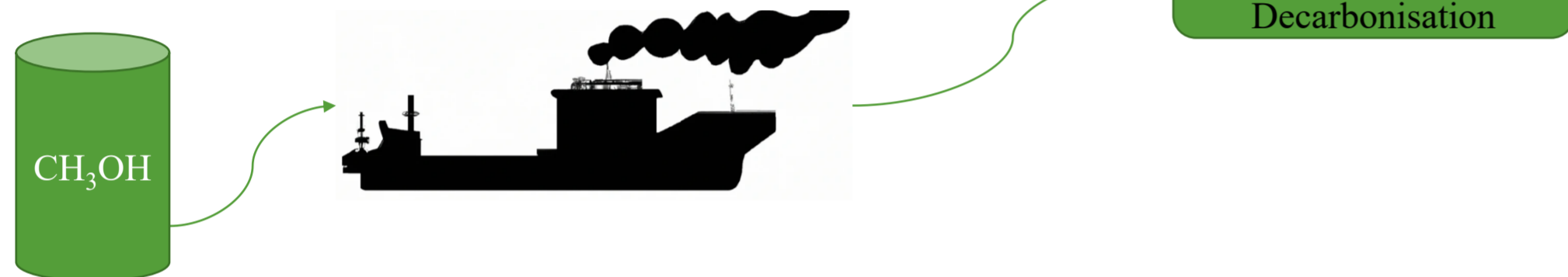
# Numerical investigation of diesel – methanol dual-fuel marine engine performance and emissions utilising high methanol fractions.

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## ABSTRACT

### BACKGROUND

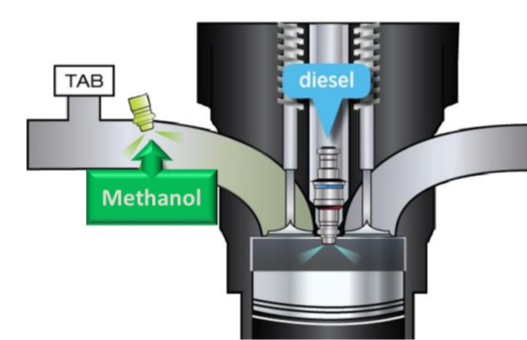


### CHALLENGES

- Lack of marine engine studies operating with high methanol fractions
- Knock combustion at high loads with methanol fuel
- Examine limitations of premixed combustion concept under diesel-methanol dual fuel operation

### AIM

- Achieving high methanol share knock-free combustion in most frequent operations of marine engines.



## METHODOLOGY

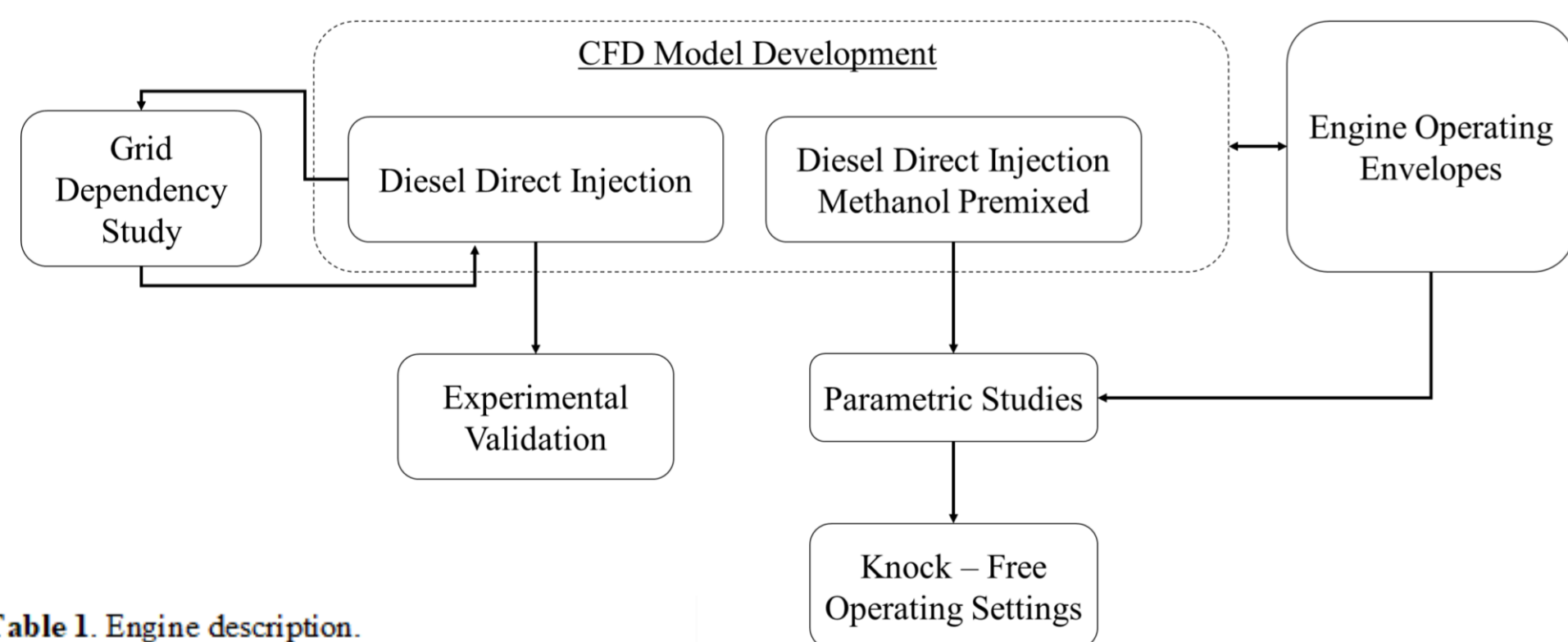


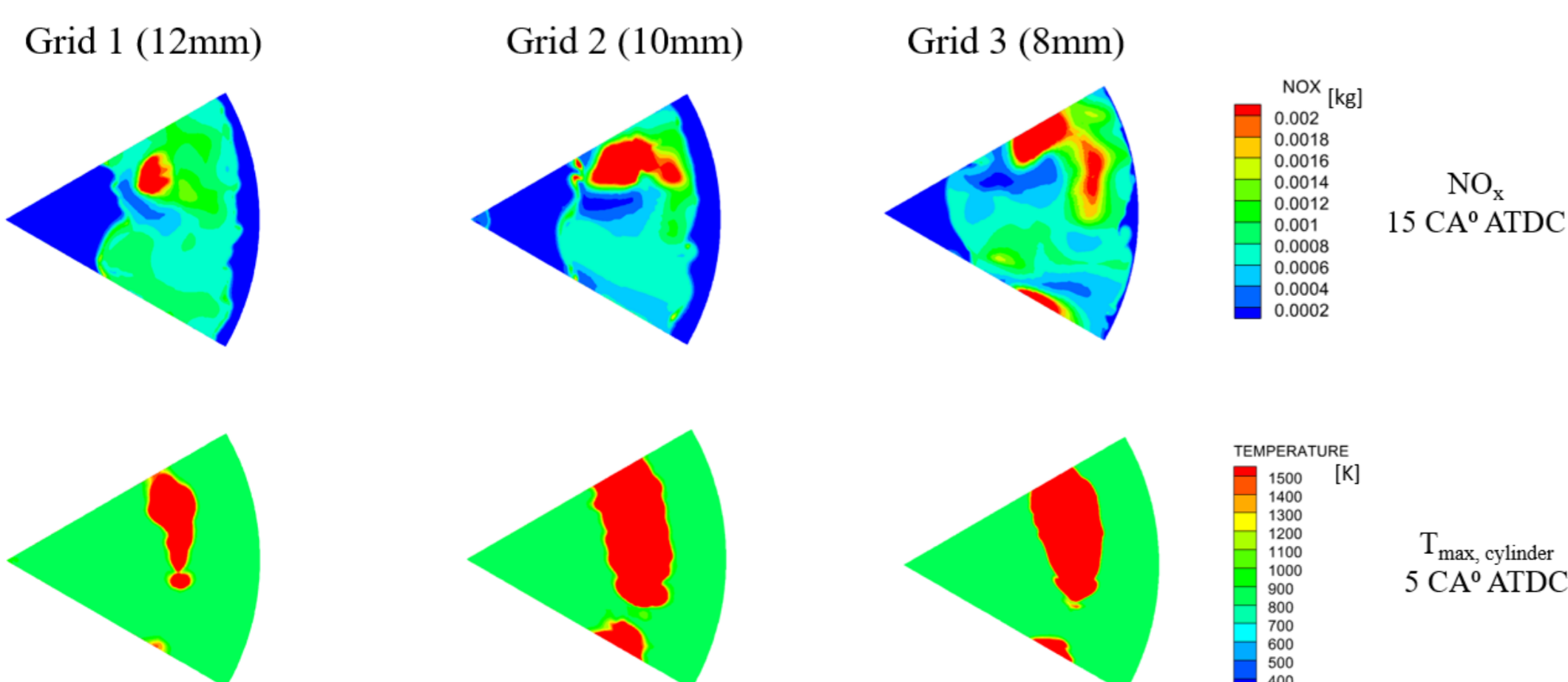
Table 1. Engine description.

Parameter	Value
Type	Wartsila 9L46C
Power Output [kW]	9450
Number of Cylinders	9
Compression Ratio	14.0:1
Bore x Stroke [mm]	460 x 580
Injection Method	Diesel: Direct Injection Methanol: Port Injection
Injection Timing	Diesel: -7 °CA BTDC

Table 2. Computational grid data

Parameter	Grid 1	Grid 2	Grid 3
Element size [mm]	12	10	08
Maximum Number of Cells*	10900	18838	36800
Adaptive mesh refinement	On	On	On
Solution duration [h]	3	4.5	9.5

\*At TDC not including embedding and mesh refining



## FINDINGS

- The CFD model matches well the experimental in-cylinder pressure and hence is considered validated.
- 50% methanol energy fraction is the upper limit of knock-free premixed combustion.
- NO<sub>x</sub> emissions are affected by the methanol high latent heat of vaporisation and the high oxygen content; the former reduces the maximum in-cylinder temperature whereas the latter, would result in increased temperature and hence NO<sub>x</sub>.
- NO<sub>x</sub> emissions reduced by 8% and 37% with methanol energy share of 10% and 50% respectively.
- For higher methanol energy fractions other combustion strategies are required, such as low pressure direct methanol injection at compression stroke.

Table 3. Engine parameters for knock – free combustion

Injection Method	Code Name	Diesel Injection	Methanol Injection	Charge Pressure [bar]	Intake Temperature [K]	Equivalence Ratio $\Phi$	EGR [mass %]
Port Injection	BL	-6 CA° BTDC	-	2.8	360	0.045	30
	1M9D-PI	-6 CA° BTDC	Port	2.8	360	0.05	30
	2M8D-PI	-6 CA° BTDC	Port	2.8	360	0.11	30
	5M5D-PI	-6 CA° BTDC	Port	2.8	360	0.27	30
	8M2D-PI	-6 CA° BTDC	Port	2	380	0.5	45

### 50% MEF

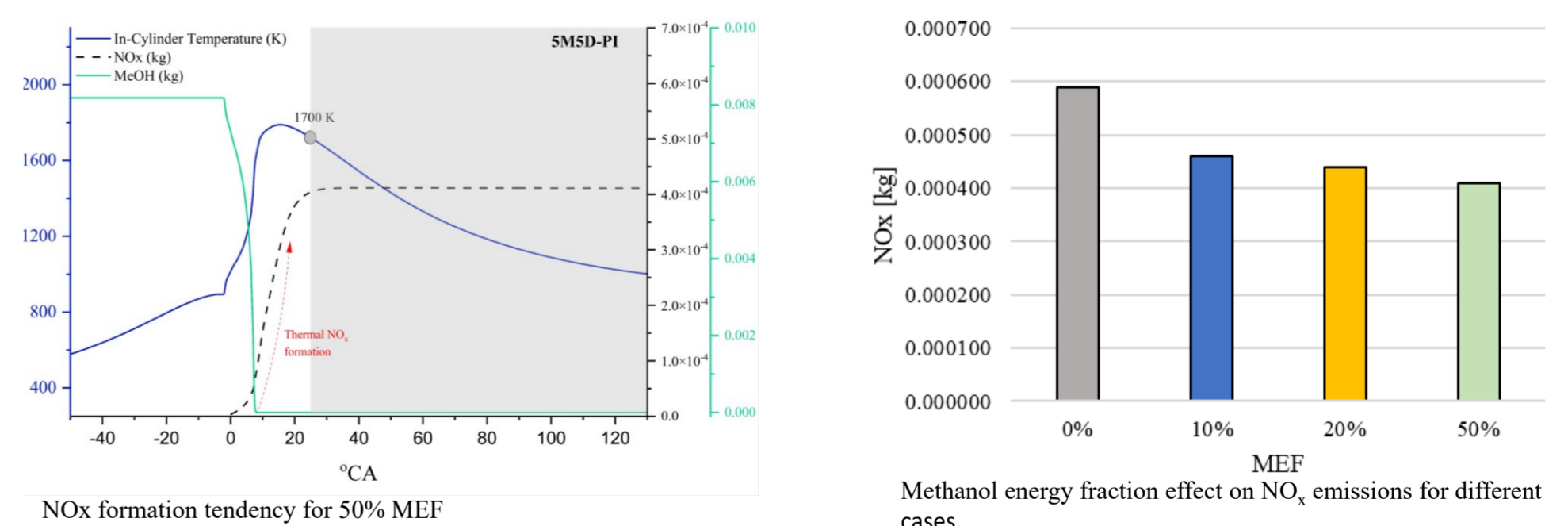
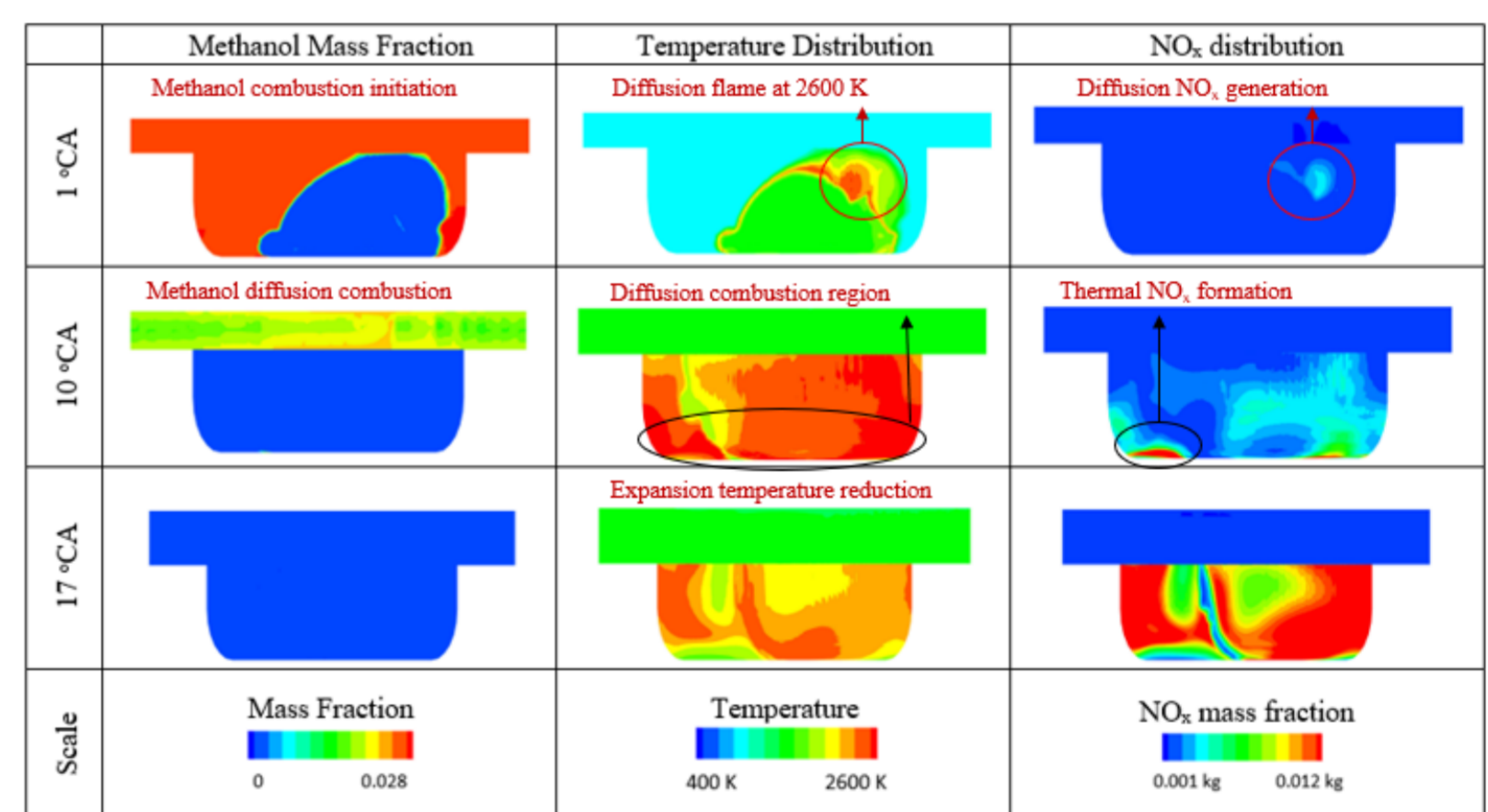


Table 4. MEF effect on engine parameters comparing to baseline case

Parameter	P <sub>max</sub>	T <sub>max</sub>	HRR <sub>peak</sub>	CA50	CA90	NO <sub>x</sub>	RI	ITE
Case	%	%	%	%	%	%	%	%
1M9D-PI	+1.6	-1.3	+5%	-4.11	+1.62	-22.03	+41	-
2M8D-PI	+8.1	-2	+23%	-19.18	-0.65	-25.42	+120	-
5M5D-PI	+35.5	-3	+390%	-51.30	-41.88	-30.51	+594	+5%

## REFERENCES

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- Datta, A., Mandal, B.K., Impact of alcohol addition to diesel on the performance of combustion and emissions of a compression ignition engine. *Applied thermal engineering*, 98, 670-682, 2016

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