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Abstract

Since the heating furnace system has emanated it has faced the problem of high power consumption, colossal amount of time to heat the substances and the vulnerability of getting exploded thus the objective of the paper is to achieve a system for same with less power consumption, whit time to heat the substances and making it safe from explosion.

Using the mathematical way of modeling the dynamic critical systems the heating furnace is being modeled by using the damping, spring and mass elements. The integer order model of the system is being achieved by the Laplace transform and fractional order model for the same is obtained using the Grunwald-Letnikov formula. The Cohen-Coon tuning technique is being amalgamated with the Nelder-Mead, Interior-Point, Active-Set and Sequential Quadratic Programming optimization techniques respectively so as to design the FOPID controller for heating furnace.

When the feedback systems were being formed then the outputs demonstrated that the system now consists the properties of less power consumption, less time to heat the substances along with less overshoot. Earlier the integer order model had the settling time (time taken to heat the substance), steady state error (power consumption) and overshoot (explosion) of 1500 seconds, 50% and 0% respectively. When the PID controller was designed for the same using Cohen-Coon tuning technique and forming a feedback system it had setting time of around 800 sec. and also the steady state error was brought to 0% but the overshoot went up to 35%. Therefore FOPID controller is being designed using the concocted technique that is the amalgamation of tuning technique and optimization techniques and forming and feedback system with FOM of heating furnace, the system yielded steady state error as 0%, where the settling time have been reduced to 300 seconds and overshoot between 7%-12%.

Using the concocted technique that is the amalgamation of Cohen-Coon tuning technique with the optimization tuning techniques the FOPID controller was being formed for the FOM of the heating furnace which is being kept in feedback so as to form a system. Thus systems formed ameliorated the settling time i.e. time taken to heat the substance, the overshoot i.e. the vulnerability of getting exploded also remains low and the steady state error i.e. power consumption is also reduced drastically.

Keywords: FOPID Controller, Heating Furnace, Optimization Techniques, Overshoot, Settling Time, Steady State Error

1. Introduction

It is fascinating to note that almost 90% of the modern controllers being used today are the Proportional Integral Derivative (PID) controller or the modified Proportional Integral Derivative (PID) controller.

Proportion Integral Derivative (PID) controller is the controller which is of the integer order and it is being utilized widely in the field of industry. The controller appeared in the year 1939 and from that point forward it has stayed crucial in view of its execution. Since most of the Proportional Integral Derivative (PID) controllers are balanced nearby, a wide range of sorts of tuning principles have been proposed in the literature. Utilizing the tuning standards, sensitive and calibrating of Proportion Integral Derivative (PID) controllers can be made on site. Likewise,
programmed tuning techniques have been created and a percentage of the Proportional Integral Derivative (PID) controller may have online programmed tuning capacities. Adjusted types of Proportion Integral Derivative (PID) control, for example, I-PD control and multi degrees of freedom Proportion Integral Derivative (PID) control are right now being used in industry. Numerous pragmatic systems for bump less changing (from manual operation to programmed operation) and increase booking are economically accessible. The handiness of Proportion Integral Derivative (PID) controls lie in their general pertinence to most control frameworks. Specifically, when the scientific model of the plant is not known and in this way investigative outlines strategies can’t be utilized, Proportional Integral Derivative (PID) controls ends up being generally helpful. In the field of procedure control frameworks, it is surely understood that the fundamental and the modified Proportional Integral Derivative (PID) control strategies have demonstrated their helpfulness in giving palatable control, in spite of the fact that in numerous given circumstances they may not give ideal control. Mathematically the Proportional Integral Derivative (PID) controller using differential equation is

\[ u(t) = K_p e(t) + K_i \int_0^t e(\tau)d\tau + K_d \frac{de}{dt} \]  

Where,

- \( K_p \) designates the gain of Proportional,
- \( K_i \) designates the gain of Integral,
- \( K_d \) designates the gain of Derivative,
- \( e \) designates the Error,
- \( t \) designates the Instantaneous time
- \( \tau \) designates the variable of integration that takes on the values from time 0 to the present \( t \).

The transfer function of Equation (1) is

\[ L(s) = K_p + \frac{K_i}{s} + K_d s \]  

Here, we are using the FOPID controller which has been derived from the Proportional Integral Derivative (PID) controller by just converting it into fractional order from the integer order which was first being proposed in 1999 by Igor Podlubny. In addition, the incentive behind the utilization of FOPID is that it is very easy to design the controller for systems with higher order by using the techniques of modeling based on regression and also because it has the iso-damping property which makes possible the variation over wide range of operating point for a singular controller. There are numerous other grounds which are liable for the utilization of FOPID controller and they are the heftiness from the high frequency noise as well as for the gain variation of the plant, the absence of the steady state error and it contains both the phase and gain margin and also the gain and phase cross over frequency.

The FOPID controller can further be defined numerically using differential Equation as,

\[ u(t) = K_p e(t) + K_i D^{-\lambda} e(t) + K_d D^{\mu} e(t) \]  

And the transfer function of the Equation (3) is given as,

\[ L(s) = K_p + \frac{K_i}{s^{\lambda}} + K_d s^{\mu} \]  

\( \{\lambda, \mu\} \) in Equation (4) designates the Differential-integral’s order for FOPID controller.

Optimization also termed as enhancement is the system of making the things more flawless, powerful and productive in order to yield the best result. The various techniques of optimization are Nelder-Mead, Active-Set, Interior-Point, SQP (Sequential Quadratic Programming) etc. Numerically it can be explained as the system of expansion and contraction of the target capacity relying upon various decision variables under an arrangement of limitations. The optimization technique has been used so as to find and achieve the finest results so as to design the most accurate FOPID controller that gives the finest output and assists the plant to enhance its performance.

Since the classical calculus cannot solve the equations with fractional order therefore the fractional calculus has been used extensively here. Basically the fractional calculus is numerical computation that scrutinizes the likelihood of accepting the real and complex number powers of operator of differentiation.

\[ 2 \text{. Modeling of Heating Furnace Dynamically} \]

The modeling that has been approximated for the heating furnace rivets amount of input that varies with time and is in fact the fuel mass gas flow rate and also the pressure inside the furnace which is the output value.

The modeling of heating furnace dynamically rivets the mass, energy and the momentum balances. It also rivets the heat transfer from the hot flue hot gas to water, flue gas flow from the boiler model and steam model.

As we know for any physical system the total force is equal to the summation of individual forces exerted by mass (m), damping (b) and spring (k) element.

Numerically we can deduce the same as
\[ F = ma + bv + kx \]  
\[ \text{(5)} \]

In the Equation (4) acceleration is signified as \( a \), velocity is signified as \( v \) and displacement is signified as \( x \). Therefore the differential equation of Equation (4) is,

\[ F = m \frac{d^2x}{dt^2} + b \frac{dx}{dt} + kx \]  
\[ \text{(6)} \]

It must be accounted that for designing a network based PID the above equation or model is a rough process behavior description. Therefore, the differential equation of the heating furnace using the above equation becomes,

\[ F = 73043 \frac{d^2x}{dt^2} + 4893 \frac{dx}{dt} + 1.93x \]  
\[ \text{(7)} \]

The Laplace transfer function of Equation (6) which gives the Integer Order Model (IOM) as

\[ G_i(s) = \frac{1}{73043s^2 + 4893s + 1.93} \]  
\[ \text{(8)} \]

‘s’ in Equation (8) signifies the Laplace operator. Where, the mass denoted by ‘m’ is 73043, the damping denoted by ‘d’ is 4893 and the spring denoted by ‘k’ is 1.93.

From Figure 1 it can be very well said that when the heating furnace is being modeled in integer order then overshoot is 0%, the steady state error is around 50% and the settling time is 1500 seconds.

### 3. A Precis on Fractional Order Calculus

Fractional order calculus is a mathematical concept that has been in existence from 300 years ago. It is the mathematical concept that has proved itself better as compared to the integer order methods.

The definition of the fractional order calculus is as follows:

According to Lacroix,

\[ \frac{d^m}{dx^m} x^n = \frac{m!}{(m-n)!} (x)^{n-m} \]  
\[ \text{(9)} \]

According to Grunwald-Letnikov, which is being used widely is,

\[ D^\alpha f(t) = \lim_{h \to 0^+} \frac{1}{h^\alpha} \sum_{k=0}^{\infty} \frac{\Gamma(\alpha+1)}{\Gamma(\alpha+k+1)} f(t-kh) \]  
\[ \text{(12)} \]

Where,

\[ \Gamma(t) = \int_0^\infty x^{t-1} e^{-x} dx \]  
\[ \text{(13)} \]

Which is called the Euler’s gamma function.

The fractional order derivatives and integrals properties are as follows:

- If \( f(t) \) being a logical function of \( t \) then the fractional derivative of \( f(t) \) which is \( D_\alpha^\alpha f(t) \) is an analytical function of \( z \) and \( \alpha \).
- If \( \alpha = n \) (\( n \) is any integer) then \( D_\alpha^\alpha f(t) \) produces the similar result as that of the traditional differentiation having order of \( n \).
- If \( \alpha = 0 \) then \( D_\alpha^\alpha f(t) \) is an identity operator.
The differentiation and integration of fractional order are said to be linear operations\(^{13}\),
\[
\mathcal{D}_t^\alpha f(t) = \frac{d^n}{dt^n} \mathcal{D}_t^\alpha f(t) = \mathcal{D}_t^\alpha \left( \frac{d^n f(t)}{dt^n} \right) = \mathcal{D}_t^\alpha \mathcal{D}_t^n f(t)
\]
(17)

Where for \( t = a, f^{(k)}(a) = 0 \) for \( k = 0, 1, \ldots, n-1 \). The given equation shows that \( \frac{d^n}{dt^n} \) and \( \mathcal{D}_t^\alpha \) are commuted\(^{13}\).

4. Cohen-Coon Tuning Technique

Cohen-Coon tuning technique is primarily used to vanquish the slow, steady state response which takes place in the Ziegler-Nichols tuning technique\(^{44}\). This technique is generally utilized for the systems with first order or models having time delay as the controller does not spontaneously responds to the disturbances.

It is an offline method that is when it is at steady state then a step change can be introduced at the input. After this on the basis of the time constant and the time delay the output can be calculated and the initial control parameters can be found out using the response\(^{31}\).

To get minimum offset and standard decay ratio there is an arrangement of pre-decided settings for the Cohen-Coon tuning technique,

Where, \( P \) is the percentage in the input, \( N \) is percentage change of output/ \( \tau \), \( L \) is \( \tau_{\text{dead}} \) and \( R \) is \( \frac{\tau_{\text{dead}}}{\tau} \). We can use \( K_u \) in place of \( \frac{P}{(NL)} \).

The procedure of the tuning technique is as follows:

- Wait for the complete procedure till the steady state is attained.
- Step change is to be introduced at the input.
- Approximate first order constant with time constant \( \tau \) which is delayed by \( \tau_{\text{dead}} \) units which is based on the output, from the time the step input was introduced. By recording the following time instances the value of \( \tau \) and \( \tau_{\text{dead}} \) can be found, \( t_0 \) signifies the input step start up point, \( t_1 \) signifies the half point time and \( t_2 \) signifies the time at 63.2%.
- Calculate the process parameters \( \tau \), \( \tau_{\text{dead}} \) and \( K_u \) by utilizing the assessment done at \( t_0 \), \( t_1 \), \( t_2 \), \( t_3 \), A and B.
- On the basis of \( \tau \), \( \tau_{\text{dead}} \) and \( K_u \) the parameters of controller can be found.

The advantages of the Cohen – Coon method are that the time of reaction of the closed loop is quick or fast and this technique can be utilized in the systems along with time delay.

Whereas the disadvantages of this technique are that it can only be utilized for the first order systems which involve large process delay, it is an offline technique, closed loop systems are unstable and the approximated value of \( \tau \), \( \tau_{\text{dead}} \) and \( K_u \) might not be compulsorily accurate for different systems\(^{42}\).

From Table 1 it can be derived that using various formulae the tuning parameters i.e. proportional gain \( K_p \), integral gain \( K_i \) and derivative gain \( K_d \) can be obtained specifically when the Cohen-Coon technique is being used.

When the Equation (8) which is the Integer Order Model (IOM) of the heating furnace is placed in a closed loop along with the PID controller which is tuned using the Cohen-Coon tuning technique as shown in the Figure 2 then the step response that we get is shown in Figure 3.

From Figure 3 it can be said that the settling time drastically reduced and also the steady state error also reduced to 0% but at the same time the overshoot increased a lot which is certainly not acceptable and all these happened when the PID controller was used.

5. Nelder-Mead Optimization Technique

Nelder-Mead optimization technique is also called the Downhill simplex technique or the amoeba technique.
Taking a function \( f(x), x \in \mathbb{R}^n \) which is to be minimized in which the current points are \( x_i, x_2, \ldots, x_{n+1} \).

- **Order:** On the basis of values at the vertices, \( f(x_i) \leq f(x_j) \leq \ldots \leq f(x_{n+1}) \).
- **Calculate the centroid of all points** \( x_i \) except \( x_{n+1} \).
- **Reflection:** Calculate \( x_i = x_0 + \alpha (x_0 - x_{n+1}) \). If the reflected point is not better than the best and is better than the second worst, that is, \( f(x_i) \leq f(x_r) < f(x_n) \). After this by replacing the worst point \( x_{n+1} \) with reflected point \( x_i \) to get a new simplex and go to the first step.
- **Expansion:** If we have the best reflected part then \( f(x_r) < f(x_1) \), then solve the expanded point \( x_e = x_0 + \gamma (x_0 - x_{n+1}) \). If the reflected point is not better than expanded point, that is, \( f(x_e) < f(x_r) \) then either by substituting the worst point \( x_{n+1} \) by expanded point \( x_e \) to get new simplex and then go to the first step or by replacing the worst point \( x_{n+1} \) by reflected point \( x_i \) to obtain or get a new simplex and then go back to the first step.
- **Else if the reflected point is not better than second worst then move to the fifth step.**
- **Contraction:** Here we know that \( f(x_i) \geq f(x_{n+1}) \), contracted point is to be evaluated \( x_c = x_0 + \rho (x_0 - x_{n+1}) \), if \( f(x_c) < f(x_{n+1}) \) that is the contracted point is better than the worst point then by substituting the worst point \( x_{n+1} \) with contracted point \( x_c \) to procure a new simplex and then go to first step or proceed to sixth step.
- **Reduction:** Substitute the point with \( x_i = x_i + \sigma (x_i - x_1) \) for all \( i \in \{2, \ldots, n+1\} \), then go to the first step.

It must be noted that the standard values for \( \alpha, \sigma, \rho, \gamma \) are 1, \( \frac{1}{2} \), -1/2, 2 respectively. In reflection the highest valued vertex is \( x_{n+1} \) at the reflection of which a lower value can be found in the opposite face which is formed by all vertices \( x_i \) except \( x_{n+1} \). In expansion we can find fascinating values along the direction from \( x_0 \) to \( x_i \) only if the \( x_i \) which is the reflection point is new nadir along vertices. In contraction it can be expected that a superior value will be inside the simplex which is being formed by the vertices \( x_i \) only if \( f(x_i) > f(x_{n+1}) \). In reduction to find a simpler landscape we contract towards the lowest point when the case of contracting away from the largest point increases \( f \) arises and which for a minimum non-singular cannot happen properly. Indeed initial simplex is important as the Nelder-Mead can get easily stuck as too small inceptive simplex can escort to local search.

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**Figure 2.** Closed loop with PID controller and the heating furnace (plant).

**Figure 3.** Step response obtained when the Equation (8) which is the Integer Order Model (IOM) of the heating furnace is placed in a closed loop along with the PID controller.
therefore the simplex should be dependent on the type or nature of problem\textsuperscript{13}.

6. Active-Set Optimization Technique

In constrained optimization, the general point is to change the issue into a less demanding sub-problem that can then be understood and utilized as the premise of an iterative procedure. A normal for a huge class of right on time routines is the interpretation of the compelled issue to an essential unconstrained issue by utilizing a penalty function for limitations that are close or past the imperative limit. Along these lines the compelled issue is settled utilizing a grouping of parameterized unconstrained advancements, which in the (arrangement's utmost) focalize to the constrained problem. These routines are currently considered moderately wasteful and have been supplanted by systems that have concentrated on the arrangement of the Karush-Kuhn-Tucker (KKT) mathematical statements. The KKT comparisons are important conditions for optimality for a constrained improvement problem. On the off chance that the issue or problem is a purported raised programming issue, that is, \( f(x) \) and \( g_i(x), i = 1, \ldots, m \), are convex function, then the KKT mathematical statements are both important and adequate for a global solution point\textsuperscript{14}.

Taking a general problem\textsuperscript{23},

\[
\min_x f(x) \tag{18}
\]

Therefore, the KKT equation can be given as\textsuperscript{14},

\[
\nabla f(x^*) + \sum_{i=1}^{m} \lambda_i G_i(x^*) = 0 \tag{19}
\]

\[
\lambda_i G_i(x^*) = 0, i = 1, \ldots, m \tag{20}
\]

\[
\lambda_i \geq 0, i = m + 1, \ldots, m \tag{21}
\]

The Equation (19) depicts a crossing out of the gradient between the active constraints and the objective function at the point of solution\textsuperscript{24}. For the gradients to be wiped out, Lagrange multipliers (\( \lambda_i, i = 1, \ldots, m \)) are important to balance the deviations in objective function’s and constraint gradient’s magnitude\textsuperscript{23}. Since only active constraints are incorporated into this crossing out operation, inactive constraints must not be incorporated into this operation, as the given Lagrange multipliers equivalent to 0. This is expressed verifiably in the Equations (20) and (21) which are the KKT equations\textsuperscript{23}.

The solution of the KKT mathematical statements shapes the premise to non-linear programming calculations. These calculations endeavor to register the Lagrange multipliers specifically. Constrained quasi-Newton procedure ensures super-linear accumulating so as to meet second order data in regards to the KKT Equations utilizing a quasi-Newton updating method. These systems are generally alluded to as Sequential Quadratic Programming (SQP) techniques, since a quadratic programming sub-problem is solved at every major iteration (also called constrained variable metric routines, iterative quadratic programming and recursive quadratic programming). The active-set optimization technique or algorithm is not an algorithm of large-scale\textsuperscript{14}.

7. Interior Point Optimization Technique

Interior point optimization technique is the method or procedure that helps to evaluate both linear and non-linear problems of convex optimization\textsuperscript{21}. It is also termed as barrier technique or method\textsuperscript{21}. The interior point way to deal with constrained minimization is to unravel a succession of estimated minimization issues. The genuine problem is\textsuperscript{21},

\[
\min_x f(x) \text{ subjected to } h(x) = 0 \& g(x) \leq 0 \tag{22}
\]

The estimated problem for every \( \mu = 0 \) is,

\[
\min_{x,s}(x,s) f_\mu(x,s) = \min_{x,s} f(x) - \mu \sum \ln(s_i), \text{subjected to } h(x) = 0 \& g(x) = 0 \tag{23}
\]

There are same numbers of slack variables \( s_i \) as there are disparity requirements in \( g \). The \( s_i \) are limited to be positive to keep \( \ln(s_i) \) limited\textsuperscript{21}. As \( \mu \) declines to zero, the minimum of \( f_\mu \) ought to approach the minimum of \( f \). The included logarithmic term is termed as barrier function\textsuperscript{21}. The rough problem Equation (22) is an arrangement of correspondence compelled problem. These are simpler to solve than the original imbalance compelled problem Equation (23)\textsuperscript{21}.  

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To solve the rough problem, the algorithm utilizes one of two principle sorts of ventures at each and every reiteration, they are, the first one is the immediate stride in \((x, s)\). This stride endeavors to comprehend the KKT (Karush-Kun-Tucker) mathematical statements for the rough issue by means of a direct estimation. This is likewise called the Newton step and the second principle is the conjugate gradient step that utilizes a trust region. Naturally the algorithm first endeavors to a direct or immediate step. In the event that it cannot then it endeavors the conjugate gradient step. One situation where it does not take an immediate or direct step is the point at which the estimated issue is not locally convex to the current emphasize. At every reiteration the algorithm diminishes a legitimacy function, for example,

\[
f(x, z) = h(x, g(z) + z)
\]

The parameter may increment with reiteration number keeping in mind the goal to constrain the arrangement towards attainability. In the event that an endeavored step does not diminish the legitimacy work, the algorithm rejects the endeavored step and endeavors another step.

8. Sequential Quadratic Programming (SQP) Optimization Technique

SQP technique speaks to the best in class in non-linear programming techniques. For example Schittkowski has actualized and tested a form of that outlanks each other tested technique as far as proficiency, precision and rate of successful elucidations over countless issues.

In view of the work of Biggs, Han and Powell and the technique permits us to firmly imitate Newton’s strategy for compelled improvement pretty much as is finished unconstrained optimization. At every significant emphasis, a guess is made of the Hessian of the Langrangian function utilizing a semi Newton overhauling system. This is then used to create a quadratic program sub problem whose arrangement is utilized to shape the quest course for a line seek technique. A review of SQP is found in Fletcher, Gill et al., Powell and Schittkowski.

Taking the Equations (18), (19), (20) and (21), the principal initiative is the detailing of a quadratic programming sub problem in light of a quadratic estimation of the Lagrangian function.

\[
L(x, λ) = f(x) + \sum_{i=1}^{m} λ_i \cdot g_i(x)
\]

We abridge the Equation (18) by assuming that bound constraints have been communicated as inequality constraints. We achieve the quadratic programming sub problem by the non-linear constraints. The SQP is implemented using three stages which are, the updating the Hessian matrix, the quadratic programming solution and the line search and merit function.

9. Designing of FOPID for the Heating Furnace using Optimization Technique

The Integer Order Model (IOM) of heating furnace, which we already equated in section (2) and we got the Equation (8), which is,

\[
G_I(s) = \frac{1}{73043s^2 + 4893s + 1.93}
\]

Now, by using the Grunwald-Letnikov Equation (12) for fractional calculus which is given as,

\[
D^\alpha f(t) = \lim_{h \to 0} \frac{1}{\Gamma(\alpha)h^\alpha} \sum_{k=0}^{(t-\alpha)} \Gamma(\alpha + k)\Gamma(k + 1)f(t - kh)
\]

When the Integer Order Model (IOM) is being solved using the Grunwald-Letnikov equation given above then we get the Fractional Order Model (FOM) of heating furnace which comes out to be,

\[
G_F(s) = \frac{1}{14494s^{1.31} + 6009.5s^{0.97} + 1.69}
\]

When the Equation (25) which is the Fractional Order Model (FOM) of the heating furnace is placed in a closed loop along with the PID controller which is tuned using the Cohen-Coon tuning technique as shown in the Figure 2 then the step response that we get is shown in Figure 4.

When PID controller for FOM of the heating furnace was designed to improve the performance of heating furnace then of course the settling time and steady state error improved but overshoot came out to be high, which can be deduced from Figure 4.

The Equation for FOPDT (first order plus dead time) is given as,
Ameliorating the FOPID (PIλDμ) Controller Parameters for Heating Furnace using Optimization Techniques

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values came out to be 151.412, 0.803807 and 4241.46 respectively.

After the discussed process we use the Nelder-Mead, interior-point, active-set and SQP optimization techniques so as to find the values of λ and μ and also the optimized values of $K_p$, $K_i$ and $K_d$ which are being listed in the Table 2 given below.

Now by using the values of Table 2 and the Equation (4) we design the FOPID controller by forming its equation.

Using the values of Nelder-Mead (NM) optimization from Table 2 we get the FOPID controller Equation as,

$$G_{FOPDT}(s) = \frac{K}{1+Ts}e^{-Ls}$$  \hspace{1cm} (26)

Where, $K$ is referred to as the gain, $L$ is referred as time delay and $T$ is referred as the time constant\(^{[2]}\).

Then, by finding out the step response of the transfer function of the plant (heating furnace) we find out the value of $K$, $L$ and $T$.

Where, $T = \frac{3(T_2 - T_1)}{2}$, $L = (T_2 - T_1)$ and $\alpha = \frac{KL}{T}$

Where, $T_1$ and $T_2$ are the time instances in seconds taken from the step response obtained having a particular steady state gain\(^{[3]}\).

So, the FOPDT model for the plant which is the heating furnace comes out to be,

$$G_{FOPDT}(s) = \frac{0.404272}{1 + 3421.935}e^{-72.464}$$  \hspace{1cm} (27)

The comparison of Equation (27) and Equation (25) is shown in the Figure 5 and form which it can be deduced that both are almost the same in performance factor.

Where the value of $K$, $L$ and $T$ we get from Equation (27) are 0.404272, 72.464 and 3421.93 respectively.

The initial values of $K_p$, $K_i$ and $K_d$ are being found out using the Cohen-Coon tuning technique whose values came out to be 151.412, 0.803807 and 4241.46 respectively.

Figure 5. FOPDT identification, comparison between the original and the identified one which appears to be perfect.

**Table 2.** Values of $K_p$, $K_i$, $K_d$, $\lambda$ and $\mu$ found using the optimization techniques

<table>
<thead>
<tr>
<th>Parameters</th>
<th>NM</th>
<th>AS</th>
<th>IP</th>
<th>SQP</th>
</tr>
</thead>
<tbody>
<tr>
<td>$K_p$</td>
<td>100</td>
<td>88.418</td>
<td>99.999</td>
<td>100</td>
</tr>
<tr>
<td>$K_i$</td>
<td>0.61134</td>
<td>99.999</td>
<td>99.999</td>
<td>100</td>
</tr>
<tr>
<td>$\lambda$</td>
<td>0.99967</td>
<td>0.34319</td>
<td>0.34772</td>
<td>0.34865</td>
</tr>
<tr>
<td>$K_d$</td>
<td>99.998</td>
<td>99.996</td>
<td>98.41</td>
<td>100</td>
</tr>
<tr>
<td>$\mu$</td>
<td>0.010014</td>
<td>0.01</td>
<td>0.010138</td>
<td>0.01</td>
</tr>
</tbody>
</table>

Now by using the values of Table 2 and the Equation (4) we design the FOPID controller by forming its equation.

Using the values of Nelder-Mead (NM) optimization from Table 2 we get the FOPID controller Equation as,

$$G_{NM}(s) = 100 + \frac{0.61134}{s^{0.99967}} + 99.998s^{0.010014}$$  \hspace{1cm} (28)

Now, by using the values of Active-Set (AS) optimization from table we get the FOPID controller Equation as,

$$G_{AS}(s) = 88.418 + \frac{99.999}{s^{0.34319}} + 99.996s^{0.01}$$  \hspace{1cm} (29)

Now by using the values of Interior-Point (IP) optimization from Table 2 we get the FOPID controller Equation as,

$$G_{IP}(s) = 99.908 + \frac{99.888}{s^{0.34772}} + 98.415s^{0.010138}$$  \hspace{1cm} (30)
10. Result

10.1 The Different Responses when Nelder-Mead Optimization Technique Values are used

From Figure 7 the step response achieved when the Equation (28) is put into the FOPID controller block and the Equation (25) is put into the plant block of Figure 6 and it exhibits overshoot of 12%, settling time of 1000 sec. and steady state error of 0%.

In Figure 8 the root locus obtained when the Equation (28) is put into the FOPID controller block and the Equation (25) is put into the plant block of Figure 6, this has been plotted to observe the stability of the system. Since all the zeroes lie at the second quadrant thus the system is stable.

The stability obtained when the Equation (28) is put into the FOPID controller block and the Equation (25) is put into the plant block of Figure 6, where the system came out to be stable with $K = 1$, $q = 0.01$, err $= 1.1481 \times 10^{-10}$ and apol $= 0.0281$ which can be observed in Figure 9.

The bode plot achieved when the Equation (28) is put into the FOPID controller block and the Equation (25) is put into the plant block of Figure 6, bode plot being obtained both in phase and magnitude can be depicted from Figure 10.

$$100s^{0.34865} + 100$$
$$100s^{0.35865} + 101.69s^{0.34865} + 100$$

Figure 6. Closed loop with FOPID controller and the heating furnace (plant).

Figure 7. Step response for Equation (32).
10.2 The Different Responses when Active-Set Optimization Technique Values are used

Figure 11 shows the step response achieved when the Equation (29) is put into the FOPID controller block and the Equation (25) is put into the plant block of Figure 6, in which settling time came out to be 300 sec., overshoot of 15% and steady state error of 0%.

Figure 12 shows the root locus obtained when the Equation (29) is put into the FOPID controller block and the Equation (25) is put into the plant block of Figure 6 in which the zeroes lie in the second quadrant which allows the system to be stable.

Figure 8. Root locus plot for Equation (32).

Figure 9. Stability graph for Equation (32).

Figure 10. Bode plot for Equation (32).

Figure 11. The step response of Equation (33).

Figure 12. The root locus plot for Equation (33).
Figure 13 shows the stability obtained when the Equation (29) is put into the FOPID controller block and the Equation (25) is put into the plant block of Figure 6, where the system came out to be stable with $K = 1$, $q = 0.01$, $err = 838e^{-10}$ and $apol = 0.0236$.

Figure 14 is the bode plot obtained when the Equation (29) is put into the FOPID controller block and the Equation (25) is put into the plant block of Figure 6, both in magnitude and phase.

10.3 The Different Responses when Interior-Point Optimization Technique Values are used

Figure 15 is the step response achieved when the Equation (30) is put into the FOPID controller block and the Equation (25) is put into the plant block of Figure 6, in which the settling time is 300 seconds, overshoot of 15% and steady state error of 0%.

Figure 16 demonstrates the step response achieved when the Equation (30) is put into the FOPID controller block and the Equation (25) is put into the plant block of Figure 6, in which all the zeroes lie in the second quadrant and from which it can be depicted that the system is stable.

Figure 17 shows the stability obtained when the Equation (30) is put into the FOPID controller block and the Equation (25) is put into the plant block of Figure 6, where the system came out to be stable with $K = 1$, $q = 0.01$, $err = 9.9290e^{-10}$ and $apol = 0.0236$.

Figure 18 is the bode plot obtained when the Equation (30) is put into the FOPID controller block and the Equation (25) is put into the plant block of Figure 6, in both magnitude and phase.
10.4 The Different Responses when Interior-Point Optimization Technique values are used

Figure 19 shows the step response obtained when the Equation (31) is put into the FOPID controller block and the Equation (25) is put into the plant block of Figure 6, in which the settling time is 440 seconds, overshoot is 7% and steady state error is 0%.

Figure 20 shows the root locus obtained when the Equation (31) is put into the FOPID controller block and the Equation (25) is put into the plant block of Figure 6, in which all the poles lie on the second quadrant which is enough to term the system as stable.

Figure 21 is the bode plot obtained when the Equation (31) is put into the FOPID controller block and the Equation (25) is put into the plant block of Figure 6, which is both in magnitude and phase.

Figure 22 is the stability obtained when the Equation (31) is put into the FOPID controller block and the Equation (25) is put into the plant block of Figure 6, where the system came out to be stable with K = 1, q = 0.01, err = 9.5622e^-10 and apol = 0.0236

11. Discussion

Palpably the transfer function of Integer Order Model abjures very poor response with a steady state error of
around 50%. When the PID controller was used then it exhibited the response with an overshoot of around 35% although steady state error becomes zero and settling time also decreases to 400 secs. Therefore this PID is designed based on the Fractional Order Model of Transfer function. When Cohen-Coon method was applied to FOM for the tuning parameters \(K_p, K_i, \) and \(K_d\), the final system became stable with an exhibited overshoot of 16%, where as the settling time increased severely up to 1000 secs. Therefore to improvise the response above mentioned optimization algorithms were used to tune the already tuned integer parameters \(K_p, K_i, \) and \(K_d\) using Cohen-Coon method and also to optimize the fractional order parameters \(\lambda \) and \(\mu\). The commonly used Nelder-Mead optimization yielded a comparatively low overshoot of 12%, whereas the problem of high settling time still existed. Therefore a new algorithm of optimization known as Interior Point was used for designing FOPID and it yielded a very nice settling time of 300 seconds but the overshoot still remained in a level of 15%. Then Active Sets optimization was used which also exhibited the similar response ass the previous one. Finally SQP optimization technique was used which decreased the overshoot of a minimum value of 7% where as the settling time was around 440 secs.

12. Conclusion

Thus, we successfully deigned the PID controller with proper optimization of integer and fractional order elements, for heating furnace. The plots of time response attributes got to be proof that the FOM of heater gave relatively great reaction by utilizing conventional Cohen-Coon tuning techniques too. Be that as it may, it displayed a high overshoot and additionally a languid reaction. As the overshoot in heater will make sudden high weight which may jeopardize the life of laborers and properties, this strategy was maintained a strategic distance from. While when all the optimization techniques were utilized, they diminished the overshoot relatively to a lower scope of 15%. In any case, when fractional components of PID were streamlined utilizing SQP advancement, the framework showed a low overshoot furthermore a similarly low settling time. Hence, it can be stated that when the fractional elements are seemly tuned then the yield is more smooth and agile.

13. References