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# **Original Article**

# A numerical investigation of flow around octopus-like arms: near-wake vortex patterns and force development

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The fluid dynamics of cephalopods has so far received little attention in the literature, due to their complexity in structure and locomotion. The flow around octopuses, in particular, can be complicated due to their agile and dexterous arms, that frequently display some of the most diverse mechanisms of motion. Their recognized manipulative and locomotor skills could inspire the development of advanced robotic arms, able to operate in fluid environments. Our primary aim was to study the hydrodynamic characteristics of such bio-inspired robotic models and to derive the hydrodynamic force coefficients as a concise description of the vortical flow effects. Utilizing computational fluid dynamic methods, the coefficients were computed on realistic morphologies of octopus-like arm models undergoing prescribed solid-body movements; such motions occur in nature for short durations in time, e.g. during reaching movements and exploratory behaviors. Numerical simulations were performed on translating, impulsively rotating, and maneuvering arms, around which the flow field structures were investigated. The results reveal in detail the generation of complex vortical flow structures around the moving arms. Hydrodynamic forces acting on a translating arm depend on the angle of incidence; forces generated during impulsive rotations of the arms are independent of their exact morphology and the angle of rotation; periodic motions based on a slow recovery and a fast power stroke are able to produce considerable propulsive thrust, even leading to a maximum displacement of 0.39 arm-lengths per period of a self-propelling arm, while harmonic motions are not. Parts of these results have been employed in bio-inspired models of underwater robotic mechanisms. This investigation may further assist elucidating the hydrodynamics underlying aspects of octopus locomotion.

Keywords: computational fluid dynamics; hydrodynamics; biologically-inspired robots; biomimetics; aquatic locomotion; octopus

## 1. Introduction

Underwater robotic systems are increasingly being used for demanding real-world applications (e.g., search-and-rescue operations, industrial inspections, or shallowand deep-water marine explorations; Yuh 2000), that point to an intensifying need for higher efficiency, maneuverability and controllability. Inspired by marine animals with outstanding manipulative and locomotor skills—like the octopus—, underwater robot designs can employ dexterous robotic manipulators to improve their application scope. Current robotic models commonly simplify the effects of flow

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Figure 1. (a-f) Octopus reaching movement involving elongation and rotation of a single arm: a large part of the arm is stiffened and held almost straight during the motion (100ms lapse between frames; video sequence by Hanassy S, Botvinnik A, Flash T, Hochner B, unpublished results; personal communication). (g-i) Bi-pedal walking followed by head-up swimming during an exploratory behavior at which the octopus translates an extended arm towards a moving target: a portion of the arm is relatively straight and stiffened, while positioned at an angle with the direction of movement (Kazakidi et al. 2012b).

and, hence, the acting hydrodynamic forces (Ekeberg 1993; Ijspeert 2001; Yekutieli et al. 2005; Sfakiotakis and Tsakiris 2007). In fact, it has been pointed out that variations in the hydrodynamic forces may strongly affect the behavior and control of robotic models in a non-trivial manner, even leading to novel propulsion modalities (Zhu et al. 2002; Thomas et al. 2005; Boyer et al. 2008; Krieg and Mohseni 2010). A detailed characterization of the flow field around aquatic animal bodies and understanding of how the fluid dynamic forces are generated could, therefore, provide valuable information for the design and accuracy of robotic models; information that only recently started to emerge for the octopus and octopus-inspired robotic arm manipulators (Kang et al. 2011; Sfakiotakis et al. 2012; Kazakidi et al. 2012a,b), but is still largely unclear, and is the subject of the current paper. Within our group's ongoing efforts to develop robotic artefacts inspired by the octopus, the purpose of this study is to derive hydrodynamic force characteristics applicable to their dynamical models.

The octopus arms are highly agile and dexterous, despite their lack of skeletal support, and, therefore, of a particular interest to the design of robotic manipulators (octopus arms are muscular hydrostats according to Kier and Smith (1985); other examples of muscular hydrostats include squid tentacles, elephant trunks, and mammalian tongues). They can perform a variety of highly complex movements, like reaching and fetching, by shortening, elongating, bending, and twisting the arm muscles (Sumbre et al. 2001; Yekutieli et al. 2005; Huffard 2006). Octopuses exhibit unique cognitive skills and show a predilection of arm usage for executing specific tasks (Zullo and Hochner 2011; Gutnick et al. 2011). They may swim or crawl along the seabed, use jet propulsion during burst swimming and exert large forces by varying their arm stiffness. Commonly, reaching movements are achieved by bend propagation along the arm, during which a large portion of the arm is stiffened and held straight (Yekutieli et al. 2005). In addition, reaching motions may often involve rotation of the arm around its base, as is shown in figures 1a-f. In these figures, the arm appears to be rigidly stiffened and rotate in a more or less planar configuration. During the exploratory behavior of figures 1g-i, the octopus translates a rigidly extended arm towards a moving target. Such solid-body-like motions do occur in nature, even though only for short durations in time (see Section 4.1 for further discussion on these figures).

However, little appears to be known about the magnitude and direction of the forces exerted on the surrounding fluid medium by the octopus body and arms, and how these forces are conveyed in order to achieve a movement. In general, aquatic Table 1. From biological organisms (*Octopus vulgaris* and *Octopus bimaculoides*) to Computational Fluid Dynamic (CFD) modeling (D is the arm base diameter).

	Biological data	CFD modeling
Arm base diameter, D	$15 \text{ mm}^{a}$	D = 10  mm
Arm length, L	$240 \text{ mm}^{\mathrm{a}}, 150200 \text{ mm}^{\mathrm{b}}$	10D = 100  mm
Arm tip diameter	$1 \text{ mm}^{a}$	0.1D = 1  mm
Reynolds number	$Re=1000-3000^{a}$	Re=1000 (for steady motion)
Arm translational velocity	$20-60 \text{ cm/sec}^{a}, 5-50 \text{ cm/sec}^{c}$	10  cm/sec (for steady motion)
Arm rotational velocity	$\sim 1.05 \text{ sec for } 90^o \text{ rotation}^s$	$\sim 1.57 \text{ sec} (1 \text{ rad/sec}) \text{ for } 90^{\circ} \text{ rotation}$
Number of octopus suckers	$150-300^{b,e}$	$38 (2 \ge 19 \text{ pairs})$
Sucker arrangement	two staggered rows <sup>b</sup>	two staggered rows of cylindrical protrusions

<sup>a</sup>Yekutieli et al. (2005). <sup>b</sup>Grasso (2008). <sup>c</sup>Gutfreund et al. (1996).

 $^{\rm d}{\rm Estimation}$  from video sequence of figure 1a-f.  $^{\rm e}{\rm Including}$  those  $<\!\!1$  mm in diameter.

organisms differ from their aerial counterparts in that the medium (water) in which they move has a much higher density than air, and is always incompressible. For animals with a body density comparable to that of water, gravity is nearly balanced out and the animal can remain suspended at a stationary depth, with little or no energy consumption (neutral buoyancy). Nevertheless, many marine organisms including species of cephalopods—are slightly denser than seawater and sinking is avoided by regulating their buoyancy; for example, by swimming continuously or hovering actively, or by using other buoyancy mechanisms (swimbladders, vacuum systems, body fluids, etc.) (Alexander 1990; Webber et al. 2000). In doing so, they may expend a considerable amount of metabolic energy to generate hydrodynamic lift (buoyancy augmentation). For swimming at a steady speed, marine animals must produce enough directional thrust to balance vertical and lateral forces and overcome the induced hydrodynamic drag (Sfakiotakis et al. 1999; Lauder and Drucker 2002; Borazjani and Sotiropoulos 2008).

There is accumulating evidence, emerging primarily from studies of fish swimming (but also of bird flight), that some fish and other marine animals (as well as birds) have developed control mechanisms to exploit perturbations and unsteady flow features (e.g., vortices and vortex sheets), considerably reducing, as a result, the cost of locomotion and remarkably increasing their swimming (or flying) performance (Lauder 2010). For example, common fish species, of the order *Perciformes*, appear to shed vortex rings in the wake of their body during locomotion, which, according to interspecies variations, could be paired or single, linked or discrete, and could reorient downstream or laterally with increasing speed to enhance or reduce, respectively, the hydrodynamic thrust (Lauder and Drucker 2002). Species of trout (of the order Salmoniformes) are able to further adapt their locomotion mechanism behind vortical structures that pre-exist in the flow stream (generated either artificially or as part of a schooling formation), by extracting energy from the vortices (Kármán gait) (Liao et al. 2003a,b) and reducing, therefore, their locomotion cost. Eels (of the order Anguilliformes), which have a more elongated and slender body shape than other fish, and have no separated caudal fin (caudal peduncle), generate jets of fluid directed only laterally, with no substantial downstream flow, when swimming steadily (Tytell and Lauder 2004; Tytell 2004). Furthermore, hydromedusan jellyfish (of the order *Filifera*) generate, during jet propulsion, a single vortex per jet pulse, the size of which could be exploited by the animal via velar diameter changes to augment thrust and reduce locomotion costs (Dabiri et al. 2006). Finally, squid (of the order *Teuthida*) produce several vortical structures as a result of both jet propulsion and fin undulations; the latter showing a notable contribution to increased thrust and propulsive efficiency (Anderson and Demont 2000; Bartol et al. 2008).

A hydrodynamic analysis around octopus arms, using either experimental, the-

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oretical, or numerical methods, is not trivial, primarily due to their flexibility and agility. Furthermore, the estimation of the relationship between thrust and velocity for use in robotic models can be ambiguous due to its dependence on multiple factors, such as the shape, the texture and the velocity of the arm. Hydrodynamic loads can also be naturally unsteady in time, due to fluctuations in the flow field, even if the arm is assumed to move at a constant velocity. A first attempt in developing mathematical models of octopus arm motions was made by Yekutieli et al. (2005); they performed force measurements on free-falling amputated arms, reinforced with various gravitational loads in order to maintain a fixed shape, and demonstrated the strong influence of hydrodynamic forces on a planar arm-reaching movement. A theoretical approach could assume each of the eight arms as a right circular cone or tapered circular cylinder, due to the general shape of the arm, which is very long as compared to the lateral dimensions (the length to base-diameter ratio is estimated between 10 to 16, table 1) (Yekutieli et al. 2005; Grasso 2008). There are indeed numerous previous works on flows around circular and tapered cylinders which could provide some insight into this direction (e.g., Taylor 1952; Vallès et al. 2002; Parnaudeau et al. 2007; Narasimhamurthy et al. 2009). However, about one third of the lateral sides of the arm is covered by suckers of various sizes and numbers (about 150-300, table 1; Grasso 2008), and although in general they are arranged in a structured way (in two staggered rows along the length of the arm), the suckers make the surface rough. It is particularly hard to suggest a theory of hydrodynamics for rough tapered geometries, since the generated loads would depend on the precise geometrical features of the roughness.

In an approach to address these issues, we used computational fluid dynamics (CFD) simulations to study and quantify the flow over realistic morphologies of octopus-like arms undergoing prescribed solid-body movements. Examples of such type of movements are displayed in Figure 1, involving straight-arm tasks. Our aim was to derive hydrodynamic loads related to such movements that could be useful in robotic arm models and therefore, at first approximation, the octopus-like arms were assumed to be rigid in our numerical simulations. The study considers various distinct arm geometries and movements. It was divided into three parts: the hydrodynamic forces were calculated, firstly, for an arm positioned at various angles of incidence to a steady flow stream (this case corresponds to an arm moving steadily at various inclined positions through a quiescent fluid); secondly, for impulsively rotating arm movements; and lastly, for periodic arm movements to examine thrust generation. For the second part, different arm morphologies were also compared. This study is a continuation of a preliminary work focusing on the effects of octopus-like suckers in straight-arm configurations and multi-segmented models (Kazakidi et al. 2012a). The results within the present work show how the hydrodynamic forces acting on moving octopus-like arms depend on the different types of prescribed motions, applicable to robotic arm models.

The remainder of the paper is organized as follows: Section 2 presents the computational modeling framework; Section 3 discusses the CFD results obtained for steadily moving, impulsively rotating, and maneuvering octopus-like arms; Section 4 discusses the biological implications and robotic applications related to this work; and Section 5 summarizes the results of this study and concludes with suggestions for further work.



Figure 2. (a) Simplified octopus-like arm. (b-d) Octopus-like arm models, including pairs of cylindrical protrusions at the upstream face, positioned in a staggered configuration (oblique projection); (b) straight arm, (c) partially bent (convex) arm, (d) fully bent (concave) arm. Insets (b-i,c-i) show a close-up view of the three top- and bottom-most pairs of protrusions (frontal view). The inset (d-i) shows a detail of the surface mesh around an octopus sucker.

### 2. Methods

### 2.1 Octopus-like arm geometries

Four octopus-like arm geometries were considered in this study: a simplified arm (figure 2a); a straight arm with a staggered array of cylindrical protrusions representing octopus suckers (figure 2b); an arm with a partial bend at the tip (convex configuration, figure 2c); and an arm with almost ninety degrees bend (concave configuration, figure 2d). The scope was not to replicate the exact anatomy of real octopus arms, as there are vast variations among octopus species (Huffard 2006), but to develop realistic arms, inspired primarily from the *Octopus vulgaris* and *Octopus bimaculoides* (Table 1), for use in biomimetic robotic-arm models.

The straight arms were approximated as right circular conical frustums with a base diameter of D, tip diameter of 0.1D and length of 10D, which gave a taper ratio of approximately 11:1, defined as length/(base diameter – tip diameter), and an aspect ratio of approximately 18:1, defined as length/mean diameter. The relative ratios of these dimensions are within the physiological scales of real octopus arm sizes (table 1; Yekutieli et al. 2005; Grasso 2008). The partially bent arm (convex configuration) was the same as the straight arm with protrusions near the base, but near the tip, approximately 33% of its length was made to follow a segment of circle of 33.6D in radius (6 last pairs of protrusions, figure 2c). The fully bent arm (concave configuration) formed a smoothed ninety-degree bend with the longitudinal arm axis, near the middle.

For the geometries with protrusions, the arm was rounded at the tip with a halfsphere of diameter 0.1D and included 19 pairs of cylindrical protrusions (a total of 38) located at the upstream face of the arm, in a staggered configuration (table 1). This is in agreement with biological observations of octopus suckers arrangement (Grasso 2008). The center-to-center distance for each column of protrusions was considered equal to 0.5D (where D is the diameter of the arm base). Starting from the base and moving towards the tip of the arm, the radius of each pair of

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protrusions decreased by 0.01D, the first pair having a radius of 0.2D and the last 0.02D (insets in figure 2b-i, 2c-i). The first sucker had a height (in the negative x-direction) of 0.04D, while the height of each subsequent sucker decreased towards the arm tip by 0.01D. The center-to-center lateral separation of each sucker within a pair (in the y-direction) was equal to their respective diameter; that is, the inner lateral lips of all protrusions were aligned to the midline of the upstream face of the arm. The protrusions were staggered in the negative z-direction, with a center-to-center separation of 0.25D.

For all geometries, the computational domain was 46D long (in the x-direction), 31D wide (in the y-direction) and 20D high (in the z-direction). The upstream and lateral lips of the arm base were located 15D away from the upstream and lateral edges of the domain, respectively.

### 2.2 Governing equations and boundary conditions

The flow field induced by the arm motion in the water is governed by the incompressible Navier-Stokes equations (Batchelor 2000; Tritton 1988). A Newtonian fluid was considered (that is, its viscosity was assumed to be constant), described by the following formulation of the Navier-Stokes equations

$$\frac{\partial \mathbf{u}}{\partial t} + (\mathbf{u} \cdot \nabla) \mathbf{u} = -\frac{1}{\rho} \nabla \mathbf{p} + \nu \nabla^2 \mathbf{u}, \tag{1}$$

$$\nabla \cdot \mathbf{u} = \mathbf{0},\tag{2}$$

where  $\mathbf{u}=[\mathbf{u}, \mathbf{v}, \mathbf{w}]$  is the velocity vector,  $\rho$  is the fluid density, p is the pressure, and  $\nu$  is the kinematic viscosity. The flow is characterized by the Reynolds number (Re<sub>D</sub>), a non-dimensional parameter, defined as

$$Re_D = \frac{UD}{\nu},\tag{3}$$

where U is the free stream velocity of the fluid, and D is the characteristic length (here, D is the base diameter of the octopus arm). The Reynolds number is a measure of the ratio of inertial forces ( $\rho U^2$ ) to viscous forces ( $\mu U/D$ ). At high Reynolds numbers, inertial forces dominate, whereas viscous forces prevail at very low Reynolds numbers. Octopus arm movements in seawater are performed with a Reynolds number of the order of 1000 (Gutfreund et al. 1996; Yekutieli et al. 2005), and therefore, inertial forces dominate (Re<sub>D</sub> >> 1).

For the first part of the study, we considered an arm positioned at various angles of incidence to a steady flow stream, where  $\text{Re}_D$  was taken equal to 1000 (based on the base diameter) and the upstream, bottom and side faces of the computational domain were assigned as inflow boundary conditions, using a uniform inlet velocity profile. The top and downstream faces were considered as pressure-outlets, while the no-slip condition was applied on the arm surface.

### 2.3 Hydrodynamic forces

The total hydrodynamic force (F) acting on the arm, when positioned at an angle with respect to the direction of the oncoming flow (figure 3a), can be decomposed into components parallel and perpendicular to either the flow direction (drag,  $F_D$ and lift,  $F_L$ , respectively) or the arm axis (axial force,  $F_A$  and normal force,  $F_N$ , Computer Methods in Biomechanics and Biomedical Engineering



Figure 3. Hydrodynamic loads acting on: (a) an octopus-like arm with protrusions and (b) an arm including an infinitesimal segment. In (a) the total hydrodynamic force (F) is decomposed into components parallel and perpendicular to either the flow direction (drag,  $F_D$  and lift,  $F_L$ , respectively) or the arm axis (axial force,  $F_A$  and normal force,  $F_N$ , respectively). The inset illustrates decomposition into components ascribable to pressure drag and skin friction drag, in the direction normal to the axis of the arm (top row, with coefficients  $C_{p1}$ ,  $C_{f1}$ , respectively) and parallel to the axis of the arm (lower row, with coefficients  $C_{p2}$ ,  $C_{f2}$ , respectively).

respectively). These are defined as

$$F = \sqrt{F_D^2 + F_L^2} = \sqrt{F_N^2 + F_T^2}$$
(4)

$$F_D = \frac{1}{2}\rho U^2 C_D A, \quad F_L = \frac{1}{2}\rho U^2 C_L A$$
(5)

$$F_N = \frac{1}{2}\rho U^2 A \left( C_D \sin\alpha + C_L \cos\alpha \right) \tag{6}$$

$$F_A = \frac{1}{2}\rho U^2 A \left( C_D \cos\alpha - C_L \sin\alpha \right) \tag{7}$$

where  $C_D$  and  $C_L$  are the drag and lift coefficients, respectively, A is the reference area, which here is taken as the projected frontal area of the straight arm, and  $\alpha$  is the angle between the axis of the arm and the direction of the flow stream (angle of incidence).

When the arm includes a complicated array of protrusions representing octopus suckers, a more detailed description of the force development could be as follows (equivalent models for smooth and rough circular cylinders can be found in the literature, e.g., Taylor 1952): Consider an infinitesimal segment of diameter 2R and length dl on the arm (figure 3b), where  $R = R_t + (R_b - R_t)l/L$ , and where  $R_b$ ,  $R_t$  are the base and tip radii of the arm, respectively, and l the position of the segment along the arm of length L. Due to the roughness of the arm with the protrusions, the acting forces could be divided into components ascribable to pressure drag and skin friction drag. Therefore, the normal force acting on the arm segment, can be expressed as:

$$dF_N = \frac{1}{2} \rho U_N^2 C_{p1} dA_p + \frac{1}{2} \rho U^2 C_{f1} \sin \alpha \, dA_s \tag{8}$$

where  $dA_p = 2R dl$  is the projected area of the arm segment in the normal direction and  $dA_f = 2\pi R dl$  is the lateral surface area of the infinitesimal segment;  $U_N$  is the normal component of the free stream velocity.  $C_{p1}$  and  $C_{f1}$  are the pressure drag and skin friction drag coefficients, respectively, in the direction normal to the axis of the arm (inset of figure 3b, top row). Integrating over the entire length of

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the arm, the above equation becomes:

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$$F_N = \frac{1}{2} \rho U^2 (R_b + R_t) L (C_{p1} \sin^2 \alpha + \pi C_{f1} \sin \alpha)$$
(9)

Similarly, the axial force acting on the arm, can be expressed as:

$$F_A = \frac{1}{2} \pi \rho U^2 \left[ R_b^2 \ C_{p2} \ \cos^2 \alpha + (R_b + R_t) \ L \ C_{f2} \ \cos \alpha \right]$$
(10)

where  $\pi R_b^2$  is the projected area of the segment in the axial direction.  $C_{p2}$  and  $C_{f2}$  are the pressure drag and skin friction coefficients, respectively, in the direction parallel to the axis of the arm (inset of figure 3b, lower row).

In the previous description, the coefficient values could possibly be approximated by empirically known values (c.f., e.g., Taylor 1952; Potter and Wiggert 2002) for flow around circular cylinders and cones, respectively, and for the appropriate Reynolds number. Instead, for a steadily-moving straight octopus-like arm (see Section 3.1), we used the theoretical formulae of Eqs. 9, 10 to fit the CFD-obtained data, and find those values of the  $C_{p1}$ ,  $C_{p2}$  and  $C_{f1}$ ,  $C_{f2}$  coefficients that give the minimum root-mean-square (RMS) error with the CFD results. This approach is likely to give rise to a prediction model for the hydrodynamic forces generated also under other characteristics of the movement, e.g. for all intermediate angles of incidence, or for other aspect and taper ratios of the arm, and/or velocity values, which are useful for robotic arm models.

### 2.4 Computational tools

The ANSA (BETA CAE Systems S.A., http://www.beta-cae.gr) grid generation software package, which offers high accuracy and flexibility in generating user-specified element sizes, was used for the generation of hybrid-type meshes and body-fitted elements on the octopus-like arms. The bent arms were constructed with the use of a morphing technique available in ANSA.

The number of triangular elements used for the surface discretization of the cylindrical protrusions ranged between 1054 and 3155, depending on their individual size, in diameter and height, and their location along the arm (cf. Section 2.1). Figure 2d-i illustrates an example of the triangular surface mesh used for one of the cylindrical protrusions, at approximately one quarter away from the arm base. The number of discretization points was found more than sufficient to numerically capture the geometrical details of the protrusions. Having ensured a well-defined surface mesh, a mesh consisting only from prismatic elements was constructed at the near wall region of each octopus arm. This prismatic volume mesh was extended outwards of the arm, up to a distance that encapsulates the boundary layer growth during the arm movement, enableing accurate capturing of viscous flow effects. The near wall region was composed of 10 layers of non-uniform thickness. The first element height was approximately 0.001D, where D is the arm base diameter, and a growth rate of 1.1 was used for the extra layers.

The rest of the computational domain was discretized with unstructured volume elements. Their size was controlled with the use of size boxes in the near volume region around the prismatic mesh, which prevented the elements to grow more than a specific value. For example, the mesh used for the periodic motions was controlled by a 12D x 2D x 15D size box around the arm (in the xyz-directions, respectively), constraining the size of the volume elements at values smaller than 0.05D. In this way, for the simplified arm geometry without protrusions (figure 2a),

a first mesh of  $0.63 \times 10^6$  prismatic and  $2.18 \times 10^6$  unstructured tetrahedral elements was generated, in addition to a much finer one of a total of  $24 \times 10^6$  elements. For the arm geometry with protrusions (figure 2b) three meshes were tested, one with  $4.85 \times 10^6$  unstructured tetrahedral elements, another with  $6.46 \times 10^6$  hybrid elements and a much finer one with  $18.76 \times 10^6$  tetrahedral elements, to capture vortex generation during periodic motions. The partially bent arm geometry (figure 2c) was solved with a mesh of  $6.17 \times 10^6$  hybrid elements, whereas the fully bent arm geometry (figure 2d) had a mesh including a total of  $17.55 \times 10^6$  elements.

Numerical simulations were carried out in Fluent (ANSYS, Canonsburg, PA), which implements the finite-volume method to solve the governing equations (Eqs. 1,2). Solution was obtained by utilizing the SIMPLE algorithm for pressure-velocity coupling, a second-order pressure discretization scheme and a second-order upwind discretization scheme for momentum. For the simulations of impulsively rotating and periodic arm movements, a moving mesh strategy was adopted, available in the flow solver. The computational mesh was made to follow exactly the arm for the entire duration of a movement, thus ensuring that the high-resolution volume mesh constructed around the arm was maintained at all instances. The meshes used for all cases examined in this paper are, therefore, considered to provide high resolution in the near-wall region, sufficient in capturing the viscous flow effects. The mesh quality and density in the near-wake field, guarantees good preservation of the generated vortex patterns, with little diffusion, thus ensuring accuracy and high fidelity in our computations.

### 2.5 Sensitivity analysis

Sensitivity analysis for the straight-arm geometry with cylindrical protrusions was carried out, by comparing results between a coarser and a finer mesh, as well as a double-sized arm. In the former case, an insignificant mesh dependency of about 0.6% for the drag coefficient was measured for the calculation of the flow at an angle of incidence  $20^{\circ}$ .

A validation study was also performed for viscous flow past a two-dimensional circular cylinder, to determine the mesh requirements for adequate vortex capturing, the accuracy in obtaining shedding frequency and unsteady loads, and the boundary conditions. The two-dimensional mesh for the validation study consisted of 65554 elements and the computational domain was 46 cylinder-diameters long (in the stream-wise direction) and 31 cylinder-diameters wide (in the cross-flow direction), which is identical to the xy-plane of the three-dimensional domain of the cases presented in this paerer, i.e. identical to a plane that is perpendicular to the longitudinal axis of the straight arm. The numerically evaluated lift and drag coefficients for a Reynolds number of 100 (which corresponds to the Reynolds number at the tip of the 3D octopus-like arm, from where flow separation primarily originates) were found in very close agreement with previous published results (Braza et al. 1986; Williamson and Brown 1998).

A second validation study was performed for flow past an impulsively starting circular cylinder. Patterns of vorticity magnitude and time evolution of the drag coefficient at Reynolds number 550, using the same 2D mesh, were found in very good agreement (both quantitatively and qualitatively) with previous published results (Bouard and Coutanceau 1980; Koumoutsakos and Leonard 1995). In all cases, the steep gradients of the near wall flow were resolved with sufficient number of elements.

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## 2.6 Data analysis

For visualization purposes, the results of the next section are presented using contours of vorticity magnitude, streamtraces of velocity colored by helicity, and isocontours of the  $\lambda_2$  criterion (see definition below) showing the existence of vortices. The vorticity,  $\omega_v$ , is a vector quantity, representing the local rotation of a fluid element, and is defined as  $\omega_v = \nabla \times \mathbf{u}$ . The helicity, H, is a measure of the angle between the velocity and vorticity vectors, defined as  $H = \int \mathbf{u} \cdot (\nabla \times \mathbf{u}) \mathbf{d}^3 \mathbf{r}$  (Moffatt and Tsinober 1992), and describes the topological linking and direction of rotation of vortex lines.

Wake vortex topology was determined with the  $\lambda_2$  criterion, as proposed by Jeong and Hussain (1995), according to which, a vortex core is located at regions where the second eigenvalue  $\lambda_2$  of the  $S^2 + \Omega^2$  tensor is negative (S and  $\Omega$  correspond, respectively, to the symmetric and antisymmetric parts of the velocity gradient). Vortex breakdown, according to Leibovich (1983), was identified by substantial perturbations in the structure of the generated vortices and transition to turbulence (Robinson et al. 1994; Ekaterinaris and Schifft 1993).

In our analysis, we also consider the integrated (unsteady) forces acting on different arm morphologies, during various motions. Variation of these forces along a specific direction can yield a further insight into the different flow effects on each arm morphology and motion profile. Unless specified otherwise, force coefficients presented in this work were based on the projected frontal area of the straight arm  $(A = 5.5D^2)$ , using the reference values of  $\rho=1022 \text{ kg/m}^3$  and U=10 cm/sec.

## 3. Results

The results of our study are presented in the following order: near-wake vortex patterns and force development are analyzed, initially, for an octopus-like arm positioned at angles spanning from zero to  $90^{\circ}$ , at intervals of  $10^{\circ}$ , with the oncoming flow (figures 4-7; this case corresponds to an arm moving steadily at inclined positions through a quiescent fluid); then, different geometries of the arms are considered and examined under impulsively rotating movements (figures 8-9); and finally, the effects of periodic movements on thrust generation are presented (figures 10-13). In the first two cases, the arms moved in the ventral direction (protrusions facing the flow), reflecting realistic octopus arm reaching movements, during which the ventral face of the arm with the protrusions is exposed in order to reach an object (figure 1a-f).

### 3.1 Arm translation at constant speed

Figure 4 displays the arm of figure 2b positioned at various angles of incidence  $\alpha$ , with respect to the oncoming flow (from  $10^{\circ}$  to  $90^{\circ}$ , at intervals of  $10^{\circ}$ ), and demonstrates how the characteristics of the flow and the pattern of the generated near-wake vortices depend on  $\alpha$ . In general, although at small angles of incidence the flow was steady in the downstream region, at a critical angle (of about  $50^{\circ}$ ) the flow exhibited natural fluctuations in time, above which full separation occurred and the flow remained unsteady.

For angles of incidence up to  $30^{\circ}$  (figures 4a-c), two counter-rotating vortices were formed simultaneously on the lee side of the octopus-like arm, along its length. The two vortices were slightly asymmetric, due to the staggering of the protrusions. Increase of  $\alpha$ , increased the magnitude of vorticity and the extent of the generated vortices: at  $\alpha = 10^{\circ}$  (figure 4a), the vortices were small in magnitude; at  $\alpha = 20^{\circ}$  November 8, 2013 16:38 Manuscript cfd revised final

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Figure 4. Near-wake flow patterns for an octopus-like arm performing steady translational motions, at various angles of incidence. Contours of instantaneous vorticity magnitude, normalized by U/D, at five horizontal slices along the straight octopus-like arm with protrusions, overlaid by instantaneous velocity streamtraces (colored by helicity). Angles of incidence  $\alpha$ : (a) 10°, (b) 20°, (c) 30°, (d) 40°, (e) 50°, (f) 60°, (g) 70°, (h) 80°, (i) 90° ( $\nu = \mu/\rho = 10^{-6} \text{ m}^2/\text{sec}$ , U= 10 cm/sec, Re<sub>D</sub>=1000).



Figure 5. Details of instantaneous velocity streamtraces, colored by helicity, showing pairs of counterrotating vortices on the lee side of steadily translating arms (only a part of the arm close to the tip is shown). (a-b) xz-plane. (c-d) yz-plane. (a,c)  $\alpha=40^{\circ}$ . (b,d)  $\alpha=50^{\circ}$ . The insets display iso-contours of  $\lambda_2$ , showing wake vortex topology ( $\lambda_2 = -50$ , Re=1000).

(figure 4b), they were strengthened and more extended; at  $\alpha = 30^{\circ}$  (figure 4c), they further enlarged within a distinct separated region in the downstream area. In this range of angles, the vortices remained attached to the lee surface, for the entire length of the arm.

For mid-range values of the angle of incidence (40°-60°, figures 4d-f), the separated region downstream of the arm progressively enlarged with increasing  $\alpha$ . At  $\alpha=40^{\circ}$  (figure 4d), pairs of counter-rotating vortices in the near-wake region were tightly attached to the lee surface of the arm, for most of its length; however, near



Figure 6. Time evolution of the drag coefficient for (a)  $\alpha = 40^{\circ}$  and (b)  $\alpha = 50^{\circ}$ , respectively, for the last stages of the simulations.

the base, an abrupt change in the structure of the vortices led to vortex breakdown. At  $\alpha=50^{\circ}$  (figure 4e), drastic variations in the vortices structure and detachment occurred even near the middle of the arm; whereas, at  $\alpha=60^{\circ}$  (figure 4f), the flow detached almost near the tip. These features are more visible in figures 5a-d, which display close-up views of the instantaneous velocity streamtraces, colored by helicity, that strike the arm almost at the tip, for  $\alpha=40^{\circ}$  and  $50^{\circ}$ . According to figure 4, these streamtraces pass approximately through the centers of the vortices (vortex cores). They helped evaluate the angular displacement of the vortex cores with respect to the arm axis (z-axis of the computational domain), as well as estimate the location along the arm where vortex breakdown occurred, as presented in Table 2. For  $\alpha=50^{\circ}$ , the vortex cores up to the detachment point were found at about 8° with the arm axis (in the xz-plane, figure 5b) and  $\pm 2^{\circ}$  with the arm axis (in the xz-plane, figure 5b) and  $\pm 2^{\circ}$  with the arm axis (in the xz-plane, figure 5b) and  $\pm 2^{\circ}$  with the arm axis (of the arm length (measured from the base).

For large angles of incidence  $(70^{\circ}-90^{\circ}, \text{ figures 4g-i})$ , flow perturbations were extended further towards the tip of the arm, where complete detachment occurred. For these angles of incidence, an extended noncanonical structure of vortices was seen at the downstream region of the arm.

## 3.1.1 Hydrodynamic loads

The forces generated by the flow on the octopus-like arm were calculated by means of CFD methods for each of the cases described in the previous paragraphs. For  $\alpha=10^{\circ}-30^{\circ}$ , the forces were steady in all directions. For  $\alpha=40^{\circ}$ , they reached a steady-state solution, although initially they were unsteady (figure 6a shows the time evolution of the drag coefficient). At the critical value of  $\alpha=50^{\circ}$ , the forces exhibited an oscillatory behavior (self-excited unsteadiness); however, the amplitudes of the oscillations were very small (figure 6b). The forces maintained their unsteady nature also for larger angles.

For each case of  $\alpha$ , both the drag and lift coefficients showed similar temporal behavior. Across all cases, however, the coefficients differed substantially with  $\alpha$ (figure 7a). There was approximately a 15-fold difference between  $C_D$  and  $C_L$ , for  $\alpha=0^{\circ}$  and  $90^{\circ}$ . However, for most of the other angles, the coefficients had comparable values (1.5- to 2.8-fold difference), indicating the importance of both forces. Overall, the drag force coefficient increased with increasing angle of incidence. The

Table 2. Approximate vortex core angular displacements and vortex breakdown locations for angles of incidence  $\alpha = 40^{\circ}$ ,  $50^{\circ}$ , and  $60^{\circ}$ , with respect to the arm axis (z-axis of the computational domain), for steadily moving arms in the x-direction (figure 4).

	$\alpha = 40^o$	$\alpha=50^o$	$\alpha=60^o$
Angular displacement with respect to the arm axis (in the $xz$ -plane) Angular displacement with respect to the arm axis (in the $yz$ -plane) Vortex breakdown location along the arm length (from the arm base)	$6.5^{o} \pm 1.7^{o} \ 3.7D$	$8^{o} \pm 2^{o} \\ 6.6D$	$9^{o} \pm 2^{o} \\ 9.8D$



Figure 7. Mean values of (a) the drag ( $C_D$ ) and lift ( $C_F$ ), and (b) the total force ( $C_F$ ) coefficients at ten angles of incidence ( $0^o$ -90<sup>o</sup>), for the steadily moving straight arm at inclined positions (the coefficients are based on the projected frontal area of the arm).

lift coefficient, on the other hand, increased almost linearly with  $\alpha$  for angles up to 30°, dropped slightly for mid-range values of  $\alpha$ , and then, after a small increase, it decreased to almost zero. The reduction in the lift coefficient for angles between 40° and 60° is concomitant with the onset of natural fluctuations in the flow that led to vortex breakdown. This also explains the delay in the growth of the drag coefficient with  $\alpha$ , at the same range of angles. It further explains the lowered values in the total force coefficient,  $C_F$ , for the same angles of incidence, as displayed in figure 7b.

In the same figure, the theoretical formulae suggested in Eqs. 9, 10 were used to find the best fit to the CFD-calculated values (solid line in figure 7b), as described in the Methods section (see Section 2). This model can fit sufficiently well with the CFD results for  $C_{p1}=0.27$ ,  $C_{p2}=-0.39$ ,  $C_{f1}=0.12$ ,  $C_{f2}=-0.01$  (the RMS error was  $1.1 \times 10^{-4}$ ; Table 3). However, the three lowered values of  $C_F$  between 40° and 60°, at which self-excited flow-induced instabilities initiated, made the model to slightly underestimate the values of  $C_F$  for the other angles of incidence. Recalculating the RMS error after exclusion of these three lowered values (between 40° and 60°), gave a better estimate for angles between 0°-30° and 70°-90° (dashed line in figure 7b), for which the RMS error was then  $2.4 \times 10^{-5}$  ( $C_{p1}=0.07$ ,  $C_{p2}=-0.15$ ,  $C_{f1}=0.20$ ,  $C_{f2}=0.03$ ; table 3). With these models, the total force coefficient can, hence, be predicted also for all other intermediate angles of incidence, or for different aspect and taper ratios of the arm.

Table 3. Total force generated on a steadily translating octopus-like arm at various angles of incidence in a quiescent fluid: value of  $C_F$  (column II) and direction of the total force vector (col. III), where  $\tan\theta_F = F_L/F_D = C_L/C_D$ .  $C_F$  was fitted by Eqs. 9, 10 (col. IV; RMS=1.1x10<sup>-4</sup>;  $C_{p1}=0.27$ ,  $C_{p2}=-0.39$ ,  $C_{f1}=0.12$ ,  $C_{f2}=-0.01$ ). Column V gives the adjusted theoretical formulae after excluding three  $C_F$  values at which flow-induced instabilities initiated (RMS=2.4x10<sup>-5</sup>:  $C_{p1}=0.07$ ,  $C_{p2}=-0.15$ ,  $C_{f1}=0.20$ ,  $C_{f2}=-0.03$ )

$\alpha^o$	$C_F^{\ CFD}$	$ an \theta_F$	$C_F^{\ (Eqs.9,10)}$	$C_{F}^{\ adjusted}$
0 10	$0.0660 \\ 0.1209$	$\begin{array}{c} 0.0729 \\ 0.4219 \end{array}$	$0.0881 \\ 0.1143$	$0.0609 \\ 0.1239$
$\begin{array}{c} 20\\ 30 \end{array}$	$0.2135 \\ 0.3181$	$0.6435 \\ 0.6842$	$0.1824 \\ 0.2706$	$\begin{array}{c} 0.2247 \\ 0.3271 \end{array}$
40 50	$0.3556 \\ 0.4062 \\ 0.4648 $	$0.5535 \\ 0.4564 \\ 0.5526$	$0.3655 \\ 0.4579 \\ 0.5002$	$0.4235 \\ 0.5092 \\ 0.5002$
60 70	0.4643 0.6291 0.6622	$0.3526 \\ 0.3061 \\ 0.1754$	0.5396 0.6036	0.5802 0.6334
90 90	0.6929	$0.1754 \\ 0.0583$	0.6584	0.6004 0.6775



Figure 8. Impulsive rotations of octopus-like arms around the y-axis, at  $\omega=1$  rad/sec. Each subfigure depicts three distinct instances in time, at different angles of rotation. The near-wake flow is illustrated with contours of instantaneous vorticity magnitude, normalized by  $\omega L/D$ , at four slices along (a) the simplified arm, (b) the straight arm with protrusions, (c) the partially bent arm (convex configuration), and (d) the fully bent arm (concave configuration), for  $10^{\circ}$ ,  $30^{\circ}$  and  $50^{\circ}$  angles of rotation.

## 3.2 Impulsive rotation of the arm

The transient rotational movements of four distinct arm morphologies within an incompressible fluid, which is initially at rest, are shown in figure 8; all arms rotated around the y-axis with a constant angular speed of 1 rad/sec, which was estimated from the video sequence of figure 1a-f (Table 1). The arms moved in the ventral direction (protrusions facing the flow) from an initial position at zero angle of rotation between the longitudinal axis of the arm and the z-axis of the computational domain. Impulsive rotation of the simplified arm from rest produced a separated region on the lee side of the arm, which enlarged with increasing angle (figure 8a). Rotation of the straight arm with protrusions resulted in more extended and asymmetric counter-rotating vortices, generated within the separated region (figure 8b). Movement of the partially bent (convex) arm led to the diffusion of the vortices near and around the bent (figure 8c), with the development of a stronger shear layer that produced thinner and more elongated vortices. Finally, impulsively starting, rotational movement of the fully bent (concave) arm produced near-wake vortices that were substantially different from those of the other arm morphologies performing the same motion. The normalized magnitude of vorticity had lower values at the corresponding slices along the arm, and the vortices were less extended (figure 8d).

To account for the different arm morphologies, the total force coefficients  $C_F$  generated on the arms during the impulsive rotations, were scaled by the nondimensional ratio of total length to projected length of each arm (figure 9). With the exception of the initial steps (due to the impulsive start), the coefficients showed similar angular evolution for each arm morphology:  $C_F$  slightly decreased up to Computer Methods in Biomechanics and Biomedical Engineering



Figure 9. Total force coefficient during impulsive rotations of four arm morphologies around the y-axis, at 1 rad/sec, scaled with the ratio of total length to projected length of each arm morphology. Dashed line indicates the average curve.

 $35^{\circ}-45^{\circ}$  angle of rotation, then moderately increased up to about  $70^{\circ}$ , after which it faintly rose (up to  $90^{\circ}$ ). In addition, the variation in the values of  $C_F$  with increasing angle of rotation was quantitatively small, for each case. Hence, it could be postulated that  $C_F$  is nearly independent of the angular position of the individual arm. It is also apparent that the values of  $C_F$  were almost independent of the exact arm morphology, for all angular positions. It is, therefore, reasonable to calculate an average of the angular evolution of the total force coefficient, as representative for impulsively rotating arms (dashed line in figure 9), that could be applied to robotic models. This averaging can further be seen as a derived curve of  $C_F$  for this type of movement. For the values of the average curve of  $C_F$  between  $10^{\circ}$  and  $90^{\circ}$ , the mean  $C_F$  was approximately 0.225.

## 3.3 Periodic motion of the arm

Inspired by the octopus arm swimming patterns (Huffard 2006), which in general involve movements of the arms in synchrony with a power (closing) and a recovery (opening) stroke of the arms (Kazakidi et al. 2012b; Sfakiotakis et al. 2012), we examined two different periodic profiles: a harmonic oscillation (figure 10a, solid line) and a "sculling" movement (figure 11, solid line). In both cases, the arm was prescribed to move periodically around the -40° angular position, in the xz-plane, with an angular span of  $\pm 20^{\circ}$ . These values were rather arbitrary, but served as a pattern of motion for a one-arm system, in which we wished to investigate the possible generation of forward thrust (in the positive z-direction). A third motion was achieved by allowing the arm to slide freely along the z-axis as a result of the propulsive force generated by the sculling motion (self-propelling arm).

1) A harmonic oscillatory motion was examined first, at which the straight arm with protrusions was made to rotate with a time-varying angular velocity of  $\omega(t) = -0.35 \sin(t) \operatorname{rad/sec}$  (figure 10a, solid line). For half of the period, the arm moved in the dorsal direction (from  $-20^{\circ}$  to  $-60^{\circ}$ ) with the protrusions trailing and, for the other half, the arm moved from  $-60^{\circ}$  to  $-20^{\circ}$  in the ventral direction (with the protrusions leading). Due to the small values of the angular velocity prescribed during the harmonic oscillation, the vorticity magnitude was small as compared to the purely rotational motions of the previous paragraphs. Nevertheless, two counter-rotating vortices were still visible on the trailing sides of the arm, for each

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Figure 10. Straight octopus-like arm performing harmonic oscillatory motion around the  $-40^{\circ}$  angular position (in the xz-plane). (a) Motion profile. (b) Instantaneous iso-contours of the  $\lambda_2$  criterion showing vortex topology for the two extreme angular positions ( $\lambda_2 = -5$ ): movement in the ventral direction (protrusions leading) is performed from  $-60^{\circ}$  to  $-20^{\circ}$ , while movement in the dorsal direction (protrusions trailing) is from  $-20^{\circ}$  to  $-60^{\circ}$ . (c)  $C_{Fx}$  and  $C_{Fz}$  force coefficients.

stroke. For both extreme angular positions, the appearance of well-formed vortices was apparent (figure 10b displays instantaneous vortical structures at the end of the two strokes).

The  $C_{Fx}$  and  $C_{Fz}$  force coefficients are displayed in figure 10c, as a function of the angle of rotation. Time-periodicity was achieved after about 1.5 periods. The results show that the force components varied with the angle of rotation and with the orientation of the oscillation (leading versus trailing faces): for  $C_{Fz}$ , the motion produced positive thrust during most of the protrusions-leading stroke and negative thrust during most of the protrusions-trailing stroke, resulting in an effectively almost zero forward thrust in the positive z-direction (integral of the  $C_{Fz}$  curve with respect to the angle of rotation for one complete period).

2) The second periodic motion was a "sculling" movement (Sfakiotakis et al. 2012), composed of a fast power and a slow recovery strokes. It involved a constant angular velocity of  $\omega_{recovery} = 0.87$  rad/sec for the recovery stroke, and a five-times larger maximum velocity of  $\omega_{power} = 4.36$  rad/sec for the power stroke, at a ratio between recovery-stroke duration and power-stroke duration of approximately 3.6. This sculling motion was prescribed in the CFD code by a polynomial expression for  $\omega(t)$  (solid line in figure 11), that was C<sup>2</sup> continuous at all points and the area integral over the entire time period was zero. Like in the previous case, the arm rotated around the -40° angular position, in the xz-plane, with an angular span of  $\pm 20^{\circ}$ . This motion profile was tested on both the simplified arm (see figures 12a,d) and the straight arm with protrusions (see figures 12b,e), in order to further examine the effects of protrusions on the generated forces and vortex evolution.

During the one-arm fixed-axis propulsive motion, the numerical computations revealed in detail the generation of complex structures of vortices around both the simplified arm and the arm with protrusions. Figures 12a,b depict instantaneous vortical patterns as observed at the beginning of the power stroke (at t = 1.84T), while figures 12d,e show instantaneous vortical patterns existing in the surrounding field at the beginning of the recovery stroke (t = 2.08T). For clarity, both time instances are marked in the plot of figure 11, over the extend of one complete period,

even though they are captured at the end of the second period and beginning of the third period of the simulations, respectively. Regions of similar vortical structures are indicated with latin numerals, using the subscripts of p and r to mark the power and recovery strokes, respectively.

Concentrating first on the arm with protrusions and observing figures 12b and 12e together, it appears that vortical structures in regions  $I_p$  and  $I_r$  were remnants of a previous transition between power and recovery strokes; flow structures in regions  $II_p$  and  $II_r$  were formed during the (slow) recovery stroke; and structures in regions  $III_p$  and  $III_r$  were generated during the (fast) power stroke. Overall, during the transition from the recovery to the early stages of the power stroke (until the maximum value of  $\omega_{power}$  was reached), a concentrated region of separated flow was generated on the protrusions-free side of the arm (figure 12b, region  $III_p$ ). In the short duration of the power stroke, the arm moved at a very large angular velocity with the protrusions-face leading the movement, resulting in large separation of the flow past the staggered protrusions (figure 12e, region  $III_r$ ). At the end of the power stroke and the transition to the recovery stroke, the arm decelerated heavily, resulting in the development of complex three-dimensional, counter-rotating vortical structures as flow separated in the opposite direction past the protrusions (figure 12e, region  $I_r$ ).

Comparing these regions of vortical structures with those obtained at the same instances in time for the smooth, simplified conical arm (figures 12a,d), it is clear that the protrusions have a notable impact on the fine details of the vortex evolution around the sculling arm. For example, a large part of fluid structures in region  $III_r$  of figure 12e is undoubtedly a result of the protrusions, since they are absent in the equivalent region of figure 12d. Nevertheless, the gross characteristics of the observed structures exist for both the simplified arm and the arm with protrusions (see, for example, region  $III_p$ ), and are therefore ascribable to the sculling motion profile. This is also evident from the resulting hydrodynamic forces, which appear to be almost identical between the simplified arm and the arm with protrusions (figures 13a,b). It is, hence, the particular sculling motion profile that has a much higher impact on the evolution of the near-wake vortex patterns and force development, than the existence or not of protrusions, although undeniably they contribute to a more realistic flow simulation around an octopus-like arm by highlighting the fine details of the flow structures.

The hydrodynamic force coefficients  $C_{Fz}$  along the z-axis (figure 13a), and  $C_{Fx}$  along the x-axis (figure 13b), generated under this periodic motion, were both



Figure 11. Motion profile of straight octopus-like arm performing sculling motion around the  $-40^{\circ}$  angular position. The recovery stroke is performed with constant angular velocity  $\omega_{recovery}$  (from  $-20^{\circ}$  to  $-60^{\circ}$ ), while the power stroke is performed with a five-times larger maximum angular velocity  $\omega_{power}$  (from  $-60^{\circ}$  to  $-20^{\circ}$ ).



Figure 12. Composite figure depicting instantaneous near-wake vortex patterns at (a-c) the early power stroke (t = 1.84T) and (d-f) the early recovery stroke (t = 2.08T), for three separate cases of sculling motion: (a), (d) Simplified arm performing sculling motion around a fixed axis. (b), (e) Straight arm with protrusions performing the same fixed-axis sculling motion as, respectively, in (a) and (d). (c), (f) Arm being free to propel forward, along the z-axis, due to the propulsive thrust ( $F_z$ ) generated by the sculling motion (self-propelling arm). Although the time-instances shown correspond to the early power or recovery strokes, the images essentially depict vortices produced during the previous strokes (recovery or power, respectively). Vortex topology in all subfigures is illustrated with instantaneous iso-contours of the  $\lambda_2$  criterion, for  $\lambda_2 = -5$ . Regions of similar vortical structures are indicated with latin numerals, using the subscripts of p and r to mark the power and recovery strokes, respectively. Dashed lines in (a) and (d) indicate the angular span of the sculling motion, which is the same for all subfigures. The profile of  $\omega(t)$  is shown in Fig. 11.  $v_z(t)$  and  $S_z(t)$  in (c) and (f) are, respectively, the propulsive speed and the resulting displacement due to the thrust generated by the sculling motion.

slightly negative during the recovery stroke, however they acquired large positive values during the power stroke. The integral of the  $C_{Fz}$  curve with respect to the angle of rotation (expressed in rads/sec) was clearly positive (with a value of 1.366 for the arm with protrusions—and 1.377 for the simplified arm without protrusions, accounting for a negligible increase of 0.8%—), suggesting the generation of considerable forward thrust for this one-arm system (in the positive z-direction). The  $C_{Fx}$  curve (figure 13b) indicates the generation of a significant force also in the lateral (x-) direction. It is not unreasonable here to speculate that the existence of a second, matching arm performing a mirrored sculling movement at some distance laterally to the first (i.e. resulting in a two-arm system, like the one in



Figure 13. (a)  $C_{Fz}$  and (b)  $C_{Fx}$  force coefficients for the three sculling cases of Figure 12. (c) Arm positioning and displacement during the sliding-axis sculling motion (self-propulsion). The numerals correspond to successive time-instances during the 4<sup>th</sup> period of the simulation, as also shown also in (d). Shading was used to facilitate visualization. (d) Time evolution of the propulsive speed  $v_z(t)$  due to the propulsive thrust  $F_z$  generated by the sculling motion. The plot also presents the resulting arm displacement  $S_z(t)$ .

figure 14a), would cancel the lateral forces, whereas the forward driving forces of each arm would be added.

3) The forward driving force  $F_z$  calculated in the previous paragraph for a fixedaxis sculling arm can, in principle, lead to the propulsion of the arm along the z-axis (this can also be stated for the lateral force  $F_x$  along the x-axis but, based on the previous speculation for a two-arm system, this force is of less interest as regards propulsion in the z direction). To investigate this aspect, in addition to the prescribed sculling motion, the arm with protrusions was allowed to slide freely along the z-axis under the influence of the propulsive force  $F_z(t)$  generated by the sculling motion (self-propelling arm). The resulting propulsive speed  $v_z(t)$ and displacement  $S_z(t)$  were calculated at each time-step during the simulation as follows:

$$\mathbf{v}_{z}(t) = \int_{0}^{t} \frac{F_{z}(t)}{m} dt, \qquad S_{z}(t) = \int_{0}^{t} \mathbf{v}_{z}(t) dt,$$
 (11)

where m is the mass of the arm, taken equal to 3.24 gr, based on the density for a single octopus arm, 1042 kg/m<sup>3</sup> (Yekutieli et al. 2005), and the volume of the arm with protrusions, approximately  $3.11 \cdot 10^{-6}$  m<sup>3</sup>. Figures 12c and 12f display vortical structures around the self-propelling arm with protrusions at the same time instances during the simulation as in the cases of figures 12b and 12e, respectively, which concern an identical arm. Despite the apparent displacement of the arm due to free-sliding along the z-axis and the associated displacement of some vortical structures, it can be observed that the main features of the vortex evolution in the near-wake field of the sliding-axis arm are adequately similar to the fixed-axis arm.

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Specifically, regions  $I_p$  and  $I_r$ , although diminished in extent, continued to appear in the field. Furthermore, regions  $II_p$ ,  $III_p$  and  $II_r$  were sufficiently similar to those of the fixed-axis arm. The largest discrepancies were found in region  $III_r$  where the flow separated during the previous recovery-to-power stroke transition (region  $III_p$ ) remained in the field (as opposed to the corresponding region of figure 12e where this structure is absent). In the same region  $III_r$ , the large separation of flow, produced during the fast power stroke past the staggered protrusions in the fixed-axis arm, was absent in the sliding-axis arm.

Both angular evolutions of the  $C_{Fz}$  and  $C_{Fx}$  force coefficients calculated for the sliding-axis arm had similar trends with the fixed-axis arm (figures 13a,b), but the positive values during the power stroke were reduced, which is expected since part of the produced force was consumed in the displacement of the arm. The integral of  $C_{Fz}$  with the respect to the angle of rotation was reduced by about 2.5 times, to the value of 0.556. It is evident that the z-component of the propulsive force generated by the sculling motion resulted in substantial displacements of the arm along the z-axis ( $S_z$  wasequaltoo.264 and 0.489, respectively, for the cases shown in figures 12c,f, expressed in arm-lengths, L). Figure 13c illustrates the positioning and displacement of the arm at six instances throughout a complete period of the simulation. Figure 13d shows the time evolution of the resulting propulsive speed  $v_z(t)$  and displacement  $S_z(t)$  of the self-propelling arm, for which periodicity was reached after 1.5 periods. The average velocity at the steady-state part (three last periods of figure 13d) was found to be approximately 0.37 L/sec, with an average maximum displacement per period of 0.39L.

### 4. Discussion

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### 4.1 Biological implications

The fluid mechanics of cephalopod movements is an intriguing subject, that has not received adequate attention, although it may offer some of the most innovative mechanisms for locomotion. It is recognized that marine organisms, in general, experience vortices in the surrounding flow or produce them to achieve specific tasks (Vogel 1996), as described in the introduction. The present work focused on studying the flow around octopus-like arms, inspired by real octopus arms (primarily by Octopus vulgaris and Octopus bimaculoides species) that commonly display a large repertoire of movements (such as reaching, fetching, grasping, swimming or crawling). Primitive motions of the arms, such as translational, rotational and periodic motions, may seem, at first instance, unrelated to real octopus arm movements, especially when very fast and agile behaviors come first to mind. Yet, octopuses spend most of their time in a den near the sea floor and, for animals kept in the aquarium, translational and rotational movements are not uncommon as part of a curiosity and exploration behavior. During such a behavior, the animal is curious about a target object and extends one (or two) arms in an attempt to reach it (Yekutieli et al. 2005). Arm extension involves activation of the arm muscles and stiffening to almost rigid conditions. If the distance between the octopus and the target object is small, the octopus elongates and rotates the extended arm (as figures 1a-f show) or uses bend propagation to unfold the arm tip, with negligible movement of its body. Otherwise, the animal may use bi-pedal walking (Huffard et al. 2005) and/or head-up swimming (Kazakidi et al. 2012b) to reach the target, while maintaining the arm extended (figures 1g-i). The extended arm preserves its relative position and shape for the duration of the octopus walking and/or swimming, and is therefore translated (moves steadily) towards the target by the octopus itself. Assuming no surrounding flow current (e.g., within the aquarium environment), such instances, although short, are equivalent to translational arm motions, as examined in this paper.

The results of this study indicate that real octopus arms experience hydrodynamic forces that support or hinder a smooth motion, which in turn promotes the dexterity of the animal's arms in performing particular underwater tasks. An example case is shown in figures 1a-f. For the overall duration of the almost-planar, rotational movement, the arm must experience a relatively moderate total hydrodynamic force, independently of the instantaneous arm morphology and angular position (cf. figures 8, 9 of Section 3.2); here, fluid forces appear to advocate a smooth motion. Another example is found in the exploratory behavior of figures 1g-i, in which the octopus translates an arm towards a moving target: While the angle of incidence between the longitudinal axis of the translating arm and the direction of movement is below a critical value (found around  $50^{\circ}$  in the simulations, cf. Section 3.1), the acting fluid forces are steady and, hence, support a smooth translational movement of the extended arm, during the relatively slow bi-pedal motion (figures 1g,h). Once the angle of incidence grows larger than the critical value (figure 1i), the generated hydrodynamic loads become unsteady, interrupting the smooth motion of the arm and forcing it to oscillate. Such induced fluctuations impair the animal's control on its arms and trigger it to switch its locomotion mode. Indeed, at the time-instance immediately following that of figure 1i (not shown), the octopus lowered the arm and swam swiftly away. Such interactions of the octopus arms with the surrounding fluid are possible and can be partly explained via CFD calculations, such as those presented in this paper. They require, however, further rigorous investigations which are not straightforward, primarily due to the inherent difficulty of working with this animal for flow visualization purposes. The siphon, the mantle, the head and the web between the arms must also play an important role in the characteristics of the flow field surrounding the octopus, as well as in the animal's ability to exploit and manipulate this flow field to enable propulsion or increase its efficiency.

### 4.2 Bio-inspired robotic applications

Much like the real octopus, an underwater, multi-arm robotic device (such as the two- and eight-arm robotic swimmers shown in figure 14) encounters hydrodynamic drag forces induced by the flow field on the arms. For a robotic arm that comprises part of such a multi-arm propulsive device, the fundamental motions of translation, rotation and sculling, as studied here, are, therefore, of a particular importance that relates to the investigation of the propulsive capabilities of the device. Further evidence was provided by the self-propelling arm presented. Additionally, the use of simplified tasks can be employed to validate primitive control strategies of the robotic device, before dealing with more sophisticated ones. For these reasons, we calculated computationally the drag coefficients of the fundamental motions and, where possible, found prediction models for forces generated under several characteristics of the arm movements (e.g. by using the equations 9 and 10). Knowing the induced drag for a single arm, we can then estimate the total drag for the multi-arm robotic system.

In this direction, the hydrodynamic loads computed in Section 3.1 were effectively applied as external loads to the dynamic model of a multi-segmented robotic arm (Kang et al. 2011). According to this model, incorporation of the hydrodynamic effects resulted in more realistic arm movements, as compared to a model that neglected the fluid effects. The computed forces have, further, assisted the A. Kazakidi et al.



Figure 14. (a) Two-arm and (b) eight-arm biomimetic robotic systems based on the arm geometry of figure 2b (Sfakiotakis et al. 2012).

development of the multi-arm robotic swimmers presented in figure 14 (Sfakiotakis et al. 2012). The CFD-computed coefficients of Sections 3.2 and 3.3 were employed in the dynamic models of these systems. The geometrical characteristics of the robotic arms used in the two- and eight-arm robotic swimmers are identical to the straight arm of figure 2b. Incorporation of the computed hydrodynamic drag coefficients resulted in a more accurate description of the robot movement. The CFD results provided, also, an assessment tool, for the validity of existing fluid drag models that are common in the robotic literature (Taylor 1952; Ekeberg 1993; Ijspeert 2001). Furthermore, the results of Section 3.3, showed that with the motion profile of figure 11, it is possible to generate considerable forward thrust for the one-arm system. The simulations also revealed in detail a complex vortical flow structure around the moving arm, which would otherwise be difficult to observe. Similar swimming gaits were tested in the multi-arm robotic prototypes and produced forward movements with substantial propulsive efficiency (Sfakiotakis et al. 2012).

This work is part of a larger research effort to develop octopus-inspired robotic arm artefacts that are able to perform various underwater tasks (Kang et al. 2011; Margheri et al. 2012; Sfakiotakis et al. 2012; Serchi et al. 2012; Vavourakis et al. 2012a). Such combined efforts could be extended to other variants of aquatic locomotion that may lead to novel propulsion mechanisms for robotic applications.

### 5. Conclusions and future work

This study considered flow around octopus-like arm models undergoing prescribed solid-body movements. A steadily moving octopus-like arm at various inclined positions was investigated, first, in detail; different arm morphologies were considered, next, under impulsively rotating movements; and, lastly, an arm performing periodic movements was presented. This range of cases was chosen in order to derive hydrodynamic loads that could be useful for robotic applications, such as those of figure 14. The CFD simulations revealed in detail the generation of complex vortical flow structures around the moving arms. It was displayed that the hydrodynamic forces acting on a steadily moving arm depend on the angle of incidence. Furthermore, forces generated during impulsive rotations of the arms were shown to be independent of the exact arm morphology and the angle of rotation. Finally, it was demonstrated that periodic motions based on a slow recovery and a fast power stroke are able to produce considerable forward thrust for a one-arm system, while harmonic motions are not. The results provide an insight into the mechanisms underlying some simplified movements of the arms within a fluid environment and identify models that could be transferred into the design and control of octopus-inspired robotic analogues.

Future work will include simulations of flow around octopus-like arms that de-

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form under prescribed motions. For this purpose, a new computational framework to address grid deformations is under development. Such investigations will also be enhanced by ongoing studies focusing on the kinematics of the octopus arm swimming patterns (Kazakidi et al. 2012b) and on the elastodynamics of octopus-like arms (Vavourakis et al. 2012b), in order to collectively provide better prediction models for robotic analogues (Sfakiotakis et al. 2012). In a greater perspective, the fluid mechanics of cephalopods appears to be quite complicated and a detailed analysis of their interaction with the aquatic environment, significantly challenging. This study aims to contribute towards this effort, and demonstrate directions where further investigation may be required.

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