1

2

3

4

01

18

Techno-Economic Comparison of Operational Aspects for Direct Drive and Gearbox-Driven Wind Turbines

David McMillan, Member, IEEE, and Graham W. Ault, Member, IEEE

5 Abstract—The majority of wind turbines currently in operation have the conventional Danish concept design-that is, the three-6 bladed rotor of such turbines is indirectly coupled with an electrical 7 generator via a gearbox. Recent technological developments have 8 enabled direct drive wind turbines to become economically feasible. 9 Potentially, direct drive wind turbines may enjoy higher levels of 10 availability due to the removal of the gearbox from the design. 11 However, this is only a theory: so far not substantiated by detailed 12 analytic calculation. By providing such a calculation, this paper 13 14 enables us to quantitatively evaluate technical and economic merits of direct drive and gearbox-driven wind turbines. 15

16 *Index Terms*—Markov chain, operational comparison, reliabil-17 ity, wind turbines.



Fig. 1. Recent growth in U.K. wind generation capacity [2].

I. INTRODUCTION

ORLDWIDE installed capacity of wind generation is 19 growing significantly and is likely to continue to in-20 crease in the future. The twin policy objectives of energy secu-21 rity and climate change mitigation have resulted in economic 22 incentives, which in turn, have driven investment in wind energy. 23 Taking the U.K. as an example, Fig. 1 shows how the installed 24 capacity has grown since 2005-by the end of 2008, the installed 25 capacity broke through the 3 GW barrier [1]. This 3 GW capac-26 ity consists of 2276 individual wind turbines (WTs) [2], the vast 27 majority of which are conventional Danish concept, gearbox-28 driven machines. However, recent technical strides have enabled 29 direct drive machines to become economically feasible. 30

Since a gearbox is not included in the direct drive concept, 31 it is clear that the reliability and availability of the WT will 32 improve-if it can be assumed that all other factors remain 33 unchanged. On the other hand, it has been reported in the paper 34 that failure rates of electrical components and generators of 35 direct drive wind turbines are significantly higher than those of 36 gearbox-driven equivalents [3], [4]. The purpose of this paper is 37 to establish if there is a technical and/ or economic advantage in 38 39 deploying direct drive wind turbines instead of gearbox-driven

Manuscript received December 5, 2008; revised. This work was fully supported by the United Kingdom Engineering and Physical Sciences Research Council under the Prosen project Grant EP/C547594/1. Paper no. TEC-00483-2008.

D. McMillan is with the Institute for Energy and Environment, University of Strathclyde, Glasgow G1 1XQ, Scotland, U.K. (e-mail: dmcmillan@ eee.strath.ac.uk).

G. W. Ault is with the University of Strathclyde, Glasgow G1 1XQ, Scotland, U.K. (e-mail: dmcmillan@eee.strath.ac.uk).

Color versions of one or more of the figures in this paper are available online at http://ieeexplore.ieee.org.

Digital Object Identifier 10.1109/TEC.2009.2032596

machines. This analysis is based on quantitative modeling of

machines. This analysis is based on quantitative modeling of the operation, failure, and maintenance of wind turbine units as proposed in [5]. Such an operational comparison of different wind turbine concepts is not available in the existing literature. 43

45

1

A. Conventional Danish Concept—Gearbox Driven

The conventional Danish concept comprises a three-bladed 46 upwind rotor, which revolves on the horizontal axis (sometimes 47 called horizontal axis wind turbine, HAWT). The coupling be-48 tween rotor and electrical generator is indirect and is achieved 49 via a gearbox in order to increase the rotational speed to a level 50 that can drive a relatively small-diameter, lightweight induction 51 generator. A conceptual view of the energy conversion process 52 for such a typical modern wind turbine is outlined in Fig. 2. 53

The whole wind turbine assembly rotates into the prevalent wind direction on its vertical axis by means of an electromechanical yaw system. Once facing into the wind, control of the mechanical input power is achieved either by aerodynamic design of the rotor (stall control) or by actively changing the angle of attack of the rotor blades to the wind (pitch control) via electrical motors or hydraulics.

The electrical configuration of Danish concept WTs is influ-61 enced by mechanical aspects, as one main objective of the WT 62 mechanical design is to minimize the weight at the top of the 63 tower, where the nacelle (containing the generator) is located 64 in modern HAWTs. This means the generator has to be as light 65 as possible and must have a relatively small physical footprint. 66 For this reason, induction generators are employed: induction 67 generators have the added advantage of being more robust than 68



Fig. 2. Process diagram for gearbox-driven and direct drive wind turbines.

synchronous generators and tend to have fewer electrical faults.
However, due to the low rotational speed of the wind turbine
rotor, a gearbox has to be used to increase the rotation from tens
of revolutions per minute at the gearbox input to thousands at
the output. The primary reason for this is the low number of
induction generator poles.

The older Danish concept WTs that operated at fixed speed 75 76 employed squirrel cage induction generators; however, newer 77 variable speed technology has resulted in a switch to doubly fed induction generators (DFIG), which are now the dominant wind 78 turbine generator configuration. The reason for the dominance 79 of this configuration is that it represents a good compromise 80 between economy and performance. It is relatively economic 81 82 because it has only a partial electronic converter rated at $\sim 30\%$ of the generator output [6], not a full converter. 83

84 B. Direct Drive Concept

In a direct drive WT, the main rotor is coupled to the generator 85 input shaft, eliminating the need for a gearbox in the design. In 86 87 order to generate power at such a low rotation speed, the generator has to have many pole pairs, and usually a synchronous 88 generator is employed. This implies much greater dimensions 89 and weight as compared with an induction generator. In addi-90 tion, a fully rated electronic power converter is required, which 91 92 increases the cost of the system.

93 C. Types of Comparison

There are several examples in the published literature where a comparison is made between the two concepts. For example, Tavner *et al.* [3] focused on how the configuration of the WT generator and converter in different design concepts affected overall WT reliability. The data utilized by the authors had enough detail to enable a direct reliability comparison of 99 three WT concepts: fixed speed with gearbox, variable speed 100 with gearbox, and variable speed direct drive (no gearbox: syn-101 chronous generator). The main conclusion was that direct drive 102 systems are less reliable than models with a gearbox because 103 the potential increase in reliability due to elimination of gear-104 box failures is cancelled out by increased generator, inverter, 105 and electrical system failures. 106

Interestingly, the authors recognized that overall availability 107 would also be affected by component repair times: In this sense direct drive systems may have an advantage, as mean time to repair (MTTR) for a gearbox is likely to be very much more than MTTR for an electronics subassembly. As yet, no other research has addressed this operational comparison of the two concepts. 113

Echavarria et al. [4] analyzed a similar dataset, which pro-114 vides some highly relevant information regarding the reliability 115 of the two WT concepts. In particular, the data suggest that gen-116 erator failures in direct drive WTs are roughly two times the 117 gearbox-driven equivalent (0.22 failures per annum compared 118 with 0.12 suggested by Tavner et al. [7]). Similarly, power elec-119 tronics failures in direct drive synchronous machines are quan-120 tified as 1.03 failures per annum compared to 0.661 suggested 121 in [7] for the induction machine equivalent. 122

Polinder *et al.* [6] examine direct drive and gearbox-driven 123 WT concepts from the viewpoint of design and economic per-124 formance. The authors define a typical Danish concept WT with 125 a three-stage gearbox (3GDFIG) and a direct drive machine cou-126 pled to a synchronous generator (DDSG). Three other concepts 127 are also defined (DFIG with a single-stage gearbox, perma-128 nent magnet direct drive, and permanent magnet single-stage 129 gearbox) but these are not considered in this paper due to the 130 fact that they are not currently deployed in significant numbers. 131



Fig. 3. Markov chain of key wind turbine components. Bold arrows and boxes indicate direct drive system.

The authors again highlight the need for further work to better
understand the reliability and availability benefits of adopting
different WT design concepts—a requirement which this paper
aims to meet.

136

III. MODELING OF WIND TURBINE CONCEPTS

137 A. Physical Modeling of Wind Turbine Components

In order to build an accurate operational model, the key physical components of the WT must be identified and a suitable mathematical representation decided upon. It was reported in [5] that using a combination of failure rate data, downtime estimates, and expert opinion, the key components of a gearboxdriven WT could be identified as follows:

- 144 1) gearbox (GBX);
- 145 2) generator (GEN);
- 146 3) rotor blades (ROT); and
- 147 4) inverter, electronics, and control (ELE).

In terms of the mathematical representation, a Markov chain 148 solved via Monte Carlo simulation (MCS) was identified in pre-149 vious studies as a suitable model framework [5]. The Markov 150 chain representation has been very successfully applied to power 151 systems infrastructure deterioration and failure modeling, in-152 cluding wind turbines [8], [9]. The main problem with these an-153 alytically solved models is that the introduction of constraints, 154 such as weather-constrained maintenance, makes obtaining a so-155 lution rather difficult. Although not computationally efficient, 156 MCS methods overcome this difficulty and have been applied 157 to similar problems in the past [10], [11]. Since the problem 158 considered is essentially a planning problem, the time required 159 160 to get the solution is of little importance.

It has been assumed that three states are sufficient to capture 161 the deterioration and failure processes of the GBX, GEN, and 162 ROT. ELE failures are assumed to be instantaneous and, there-163 fore, require only binary representation. When all four com-164 ponents are modeled in a single Markov state-space, the total 165 number of states is 54. This is cut down to 28 by assuming that 166 degradation and failure events of different WT components can-167 not happen concurrently. Furthermore, for a GBX, GEN, or ROT 168 failure to occur, the system must transit through the deteriorated 169 (intermediate) state before outright failure. 170

The possible Markov states and transitions for the overall WT 171 system are visualized in Fig. 3. The three possible deterioration 172 levels are indicated as fully up (U), deteriorated (!), and down 173 (D). 174

All 28 states and transitions are possible for the case of 175 gearbox-driven WTs. The obvious physical difference when 176 modeling a direct drive WT is that there are no gearbox states. 177 Thus, the bold arrows in Fig. 3 refer to the transitions and states 178 that represent direct drive WTs, as a 12-state subset. 179

The arrows in Fig. 3 represent transition probabilities (e.g., 180 probability of transition from state a to state b is $p_{a,b}$), whose 181 magnitude must be estimated. These probabilities depend only 182 on the current state of the system (s) at current time t_k . 183 Equation 1 expresses this "memory-less" property of a Markov 184 chain 185

$$p_{a,b} = p(s_b, t_{k+1}|s_a, t_k)$$
 $k = 1, 2, 3, \dots$ (1)

Furthermore, the transition probabilities are constant in time: 186 This is the "time-stationary" property. The magnitude of all 187 transition probabilities from one state (*a*) to all others in the 188

 $p_{1,1} = 1 - p_{1,2}$ $p_{2,2} = 1 - p_{2,n}$ Estimated from sensitivity analysis $p_{2,n}$ Estimated from expert judgement
each component failure $p_{n,n} = 1$ Overall reliability is known
from 'target' failure rates

Fig. 4. Parameter estimation for Markov chain.

system must sum to unity. This is shown in (2)

$$\sum_{b} p_{a,b} = 1 \qquad a = 1 \dots n \tag{2}$$

where *n* is the total number of system states. For convenience, the transition probabilities for the whole system are expressed in a transition probability matrix (TPM). For the case of the system in Fig. 3, the TPM is shown in (3). Note that only the possible transitions (indicated in Fig. 3. with arrows) need to be estimated—all other probabilities are equal to zero

$$\text{TPM} = \begin{bmatrix} p_{1,1} & p_{1,2} & p_{1,28} \\ p_{2,1} & p_{2,2} & \ddots \\ \vdots & \vdots & \ddots & \vdots \\ p_{28,1} & \vdots & p_{28,28} \end{bmatrix}.$$
 (3)

The TPM values are estimated based on the partial information available, as illustrated in Fig. 4. A good estimate of the overall reliability ("target" failure rate) is known from the work of Tavner *et al.* [7].

Similarly, downtime estimates can be made for the outage of the key components [5]. The probability of an outright failure in a deteriorated condition $(p_{2,n})$ can be estimated based on expert opinion of times to failure. The remaining parameters can then be estimated by conducting sensitivity analyses (previously reported in [5]).

206 B. Data for Physical Model

The study of Tavner *et al.* [7] provided estimates of component failure rates based on populations of Danish and German WTs. The German population was larger (over 4,000 machines) and the population consisted of more modern WTs. Therefore, the German figures are used to fit the gearbox-drive WT physical model.

On the other hand, Echavarria *et al.* [4] suggest modifications
to the GEN and ELE failure rates for direct drive machines.
Taking this into account, the target failure rates for both WT
concepts can be visualized in Fig. 5.

Downtimes for the failure types are as follows: GBX—30 days, GEN—21 days, ELE—1 day, and ROT—30 days. They are based on domain knowledge elicited in [5]. Using this



Fig. 5. Reliability for gearbox-driven and direct drive wind turbines.

information, the TPM parameters were estimated based on the 220 iterative procedure devised in [5]. The procedure is based on 221 sensitivity analysis estimation of the unknown parameter $p_{1,2}$ 222 (see Fig. 4). 223

The Markov chain has been defined for both gearbox-driven 224 and direct drive concepts. In the following sections, other aspects of operational modeling—which are common to both 226 concepts—are discussed. 227

C. Energy Yield Modeling

There are two main components to the energy yield model. 229 These are the wind speed model and the power curve model. The 230 wind speed (WS) model previously used by the authors [5] was 231 based on a single parameter autoregressive process, or AR(1). 232 This is displayed in (4), where μ is the mean of a wind speed 233 time series, ϕ is the autoregressive parameter and the process is 234 driven by a Gaussian white noise function ε_t . 235

$$WS_t - \mu = \phi (WS_{t-1} - \mu) + \varepsilon_t .$$
(4)

228

The dataset used to fit the model was sourced from a supervisory control and data acquisition (SCADA) system of a 237 U.K. wind farm. Estimation of ϕ and ε_t was achieved by linear least squares, while classification of the model was based 239 on inspection of the autocorrelation and partial-autocorrelation 240 functions. 241

The power curve model is based on a manufacturers' 242 datasheet for a 2 MW WT [12], which is sampled and the theoretical equation for the power (P) in the wind (5) is matched to the data samples by modeling the coefficient of performance C_p 246

$$P = \frac{1}{2}\rho\pi r^2 v^3 (\times C_p).$$
⁽⁵⁾

In (5), ρ is air density (kg/m³), *r* is the rotor radius (m), and *v* 247 is air velocity through the WT rotor (m/s). The re-created power 248 curve is shown in Fig. 6 and has cut in, rated, and cut out wind 249 speeds of 4, 14, and 25 m/s, respectively. It is assumed that both 250 WT concepts adhere to the same power curve. 251



Fig. 6. Power curve for 2 MW wind turbine [12].

252 D. Maintenance Modeling

It is assumed that a six-monthly periodic maintenance plan 253 is adopted for both WT concepts. The assumptions in the main-254 tenance model are that maintenance actions restore the WT to 255 256 the fully up state (state 1 in Fig. 3) and that each maintenance visit involves a one-day outage. The model can easily accom-257 modate condition-based maintenance (CBM), but since mainte-258 nance paradigms are not the focus of this paper, this possibility 259 is neglected. 260

261 If a component failure occurs, a maintenance team is dispatched immediately. There is a probability that a component 262 replacement is necessary (replacement factor $\beta = 0.6$) or that it 263 can be repaired $(1-\beta = 0.4)$. This is based on an analysis by 264 Ribrant and Bertling [13] who highlighted that around 60% of 265 gearbox failures require a replacement rather than a repair ac-266 tion. Similar figures have not been published for the other WT 267 components; therefore, due to the lack of data, they are assumed 268 to have the same probabilities of repair and replacement as the 269 gearbox. 270

271 If the component can be repaired, it is restored instantaneously to a functional state. If a replacement is required, down-272 273 time lasts as follows: GBX-30 days, GEN-21 days, ELE-1 day, and ROT-30 days. This is based on the experience of a 274 wind farm operator. Furthermore, maintenance actions are con-275 strained by wind speed as in [5]. This means that nacelle-related 276 277 replacements need wind speed conditions of less than 10 m/s, while rotor maintenance cannot be conducted in wind speeds 278 over 7 m/s. 279

280 E. Costs and Revenue

It is of interest to compare the economic merits of the two WT configurations. Therefore, a cost model has been built which generates revenue from energy yield and incurs maintenance and replacement costs.

Polinder *et al.* provided costs for gearbox-driven and direct drive wind turbine components rated at 3 MW [6]. Figures can be derived for 2 MW machines of both types assuming that the cost varies linearly with the rating. These costs are provided in Table I. The rotor cost was not provided in [6] and so the value derived previously by the authors of this paper is adopted [5]. In the case of a component replacement, the full cost in Table I

 TABLE I

 COMPONENT COSTS FOR WIND TURBINE CONCEPTS

Component	Gearbox driven	Direct drive
Component	Cost £	
GBX	121,733	N/A
GEN	177,066	313,740
ROT	210,000	210,000
ELE	22,133	66,400

is incurred. In the case of a repair, it is assumed that only 10% 292 of this cost is incurred (repair cost factor $\alpha = 0.1$). 293

Besides the costs of the components themselves, the cost of 294 labor and equipment hire has been included. Andrawus [14] 295 showed that skilled labor for WT repairs costs around £50/h. It 296 has been assumed that three crew working an 8 h shift consti-297 tute one maintenance action. Therefore, the cost of labor (C_{LAB}) 298 is $\pounds 1200$ per action. Similarly, hire rates for telescopic cranes 299 $(C_{\rm EO})$ needed to perform nacelle component lifting operations 300 have been quantified by industry sources [15] as $\pounds 1500$ per 301 week. Lost revenue due to downtime is also taken into account 302 (R_{LOST}) —this is wind speed-dependent. These costs are com-303 bined with the component costs (see Table I) to calculate re-304 placement (6), repair, (7) and O&M cost (8) 305

$$C_{\text{replace}} = \beta \lambda \left(C_{\text{CAP}} + C_{\text{LAB}} + C_{\text{EQ}} + R_{\text{LOST}} \right)$$
(6)

$$C_{\text{repair}} = (1 - \beta) \lambda \times (\alpha C_{\text{CAP}} + C_{\text{LAB}} + C_{\text{EQ}} + R_{\text{LOST}})$$

$$C_{\rm O\&M} = C_{\rm replace} + C_{\rm repair} \,. \tag{8}$$

The revenue model for the WT is based on the energy yield in 306 each one-day simulation interval. Using equations (4) and (5), 307 this energy yield *Y* can be calculated as the power (see Fig. 6) 308 multiplied by a time interval Δt . The energy yielded in a year is 309 then calculated by summing the output over all individual days 310 in the year. 311

The revenue stream *R* can then be calculated by applying 312 equation (9). MP represents the market price for electricity and 313 renewable obligation certificates (ROCs). For this paper, MP_{elec} 314 and MP_{ROC} are set to £36 and £40 per MWh, respectively. 315 Although in reality, electricity and ROC prices fluctuate, the annual mean is adequately represented by the figures presented 317 as 318

$$R = Y \left(\mathrm{MP}_{\mathrm{elec}} + \mathrm{MP}_{\mathrm{ROC}} \right) - C_{\mathrm{O\&M}} \,. \tag{9}$$

It is important to note that any differences in yield between the 319 two WT concepts will be related to the reliability and downtime 320 (see Fig. 5 and Section III-D) of the two WT concepts, rather 321 than to the differences in the electrical design. This is because the 322 same power curve (Fig. 6) has been used for both WT concepts. 323

Polinder *et al.* showed that the theoretical difference in yield 324 between a 3 MW direct drive machine (DDSG) and a typical 325 3 MW DFIG (DFIG3G) is +150 MWh [6], if the detail of 326 the electrical machine design is taken into account. Assuming 327 this difference scales linearly with WT rating, it means that for 328 the 2 MW machines considered in this paper, the direct drive 329 machine yields roughly 100 MWh more per annum than the 330



Fig. 7. Availability comparison of wind turbine concepts.

DFIG. Applying (9) and neglecting the O&M cost, this translates to \pounds 7600 more revenue per annum for the direct drive machine. Therefore, the annual revenue for the direct drive machine has been boosted by \pounds 7600 per annum in the studies that follow.

335 F. Program Operation

The models outlined in this section were coded in FORTRAN
95, with the SCADA database interface for the wind speed
model written using f90SQL [16]. The statistical programming
language R was used to fit the wind speed model defined in (4).
The resultant capacity factor of the wind turbine based on the
simulated wind speed and power curve is just under 30%.

The confidence limit *L* of the simulation results can be measured by applying (10). Taking the Student-*t* distribution, and setting the level of confidence to 95%, it means that *L* can be specified, provided that the number of samples (*N*) and standard deviation (σ) of the quantity are known. *L* is shown in the results as confidence bands that specify the accuracy of the results

$$L = \pm \frac{2.045 \times \sigma}{\sqrt{N}}.$$
 (10)

IV. RESULTS

349 A. Operational Comparison of Concepts

348

Two comparisons are made in order to benchmark the operational merits of the two WT concepts: a technical comparison and an economic comparison. The first result in Fig. 7 compares the overall availability of the two concepts.

It can be seen that despite removing the gearbox from the design, the direct drive concept has similar overall availability to the gearbox-driven machine. Although the availability is marginally better for the case of the direct drive machine, the confidence limits show that this technical benefit is uncertain. It should be noted that grid availability is not included in this paper.

The second result, displayed in Fig. 8, shows the revenue generated (9) for both concepts. This shows that the gearboxdriven design has a much larger economic benefit than the direct drive concept.

The contribution to the revenue of increased energy yield due to avoidance of downtime is negligible in the case of Fig. 8 (direct drive machine avoids loss of \sim 6.35 MWh more energy



Fig. 8. Economic comparison of wind turbine concepts.



Fig. 9. Availability improvement of direct drive wind turbine as a function of generator reparability.

than gearbox-driven, economic benefit = $\pounds 482$ per annum). 368 Therefore, the large disparity in revenue ($\sim \pounds 44\ 000$ per annum) 369 must be due to incurred repair and replacement costs. The large 370 increase in cost and failure rate for the generator in particular 371 (see Table I and Fig. 5) appears to economically handicap the 372 direct drive concept. 373

B. Operational Impact of Generator Reparability 374

One possible explanation of the superior economic perfor-375 mance of the gearbox-drive concept is that a replacement factor 376 (β) of 0.6 per failure (see Section III-D) may represent a pes-377 simistic view of the "reparability" of a WT synchronous gener-378 ator. Indeed, it has been reported elsewhere [3] that the increase 379 in generator failure rate for the direct drive concept is related 380 to electrical failures rather than mechanical failures. Electrical 381 faults will be less likely to involve a complete component re-382 placement; therefore, the robustness of the conclusion drawn 383 from Fig. 8 is tested by modeling different levels of reparability 384 for the direct drive generator. 385

The replacement factor β was reduced from the base value of 386 0.6–0.1, as shown in Fig. 9. This figure shows that the operational availability of the direct drive concept WT can be significantly higher than the gearbox-driven WT, if a high proportion 389 of synchronous generator failures are minor electrical failures 390 rather than severe mechanical failures (e.g., bearing problems). 391



Fig. 10. Revenue increase of direct drive wind turbine as a function of generator reparability.



Fig. 11. Revenue increase of direct drive wind turbine as a function of component cost reduction.

The economic impact of this variation is illustrated in Fig. 10. 392 This figure shows that even for an optimistic scenario, the annual 393 revenue of the direct drive WT is still \sim £20 000 less than the 394 equivalent gearbox-driven WT. This aspect of WT component 395 reparability has not received much attention in the literature, but 396 397 Fig. 10 in particular shows that it is a significant factor when conducting operational modeling of WT concepts. More stud-398 ies of the type conducted by Ribrant and Bertling [13] will be 399 needed in order to better understand the reparability of different 400 WT components and their effect on operational metrics such as 401 402 availability and revenue. Analysis of WT failures in the context of repairs and replacements along with their probabilities 403 and costs are crucial for a deep understanding of wind farm 404 operational issues. 405

406 C. Operational Impact of Component Cost Reduction

In the WT marketplace, there is currently one company that 407 builds 2MW direct drive machines on an industrial scale [17]; 408 however, other large WT manufacturers have identified direct 409 drive machines as an avenue for future production [18]. With 410 more players in the market, it may be possible to significantly 411 reduce the direct drive component cost through a refinement 412 of mass production manufacturing processes. Therefore, it is 413 of interest to review the effect on the economic case for direct 414 drive machines if component costs are lowered. Such a review is 415 provided in Fig. 11, where the GEN and ELE component costs 416 are reduced in 10% steps to 50% of the original Table I values. 417

The final result shows that if substantial cost reductions in 418 direct drive technology are achieved in the future, this measure 419 may be enough to make the technology cost-competitive with 420 DFIGs. However, very large cost reductions of 50%+ will be 421 required. At current prices, the economic argument for a switch 422 to direct drive technology, for the onshore conditions evaluated, 423 appears to be weak. 424

V. CONCLUSION

An operational comparison of direct drive and gearbox-driven 426 wind turbines has been presented in this paper. The results sug-427 gest that there may be a technical advantage in deploying direct 428 drive machines over more established gearbox-driven designs 429 (see Fig. 9). In all cases, the economic analysis shows that 430 gearbox-driven machines are still preferable, unless manufactur-431 ing costs of direct drive technology can be significantly reduced 432 (see Fig. 11). 433

There are some issues that need to be better understood in 434 order to make more precise comparisons of these technologies. 435 One is that the repair probability of the components needs to be 436 investigated, in a manner similar to the one presented in [13] 437 but for all WT components. The failure rate increase for a syn-438 chronous generator relative to an induction generator (reported 439 in [3], [4]) will be made up mainly of electrical-related failures 440 rather than mechanical failures. It would be interesting to see 441 what proportion of direct drive WT generator failures are low 442 downtime (e.g., 1–3 days) as opposed to a mechanical failure of 443 a rotating component, which in some cases could take as long as 444 60 days to replace [14]. Such an analysis would aid understand-445 ing of WT failure modes and make operational comparisons 446 more accurate. 447

This study was carried out for fairly typical onshore condi-448 tions, but the conclusions may be linked to the site conditions. 449 Direct drive machines are perceived by some manufacturers as 450 primarily an offshore technology [18]. By modeling the offshore 451 wind resource, logistics, increased downtimes, and offshore ac-452 cess constraints, it may be possible to determine if direct drive 453 machines would become more economically attractive in off-454 shore conditions than the analysis presented in this paper shows. 455 The conclusions of the results in this paper and any further anal-456 ysis as described will be of value to both manufacturers and 457 operators of wind turbines. 458

ACKNOWLEDGMENT

The authors would like to thank Y. Patel of Scottish 460 Power/Iberdrola and M. Smith of Macom Technologies for their 461 valuable input to this paper. The authors would also like to thank 462 Dr. D. Hill and the IEEE reviewer for their valuable comments. 463

REFERENCES

- Scottish Power/Iberdrola press release (2008, Nov. 9). "Scottish 465 Power Renewables Windfarm Pushes Britain Past 3GW Wind Energy Landmark," [Online]. Available: http://www.(scottishpower.com/ PressReleases_1774.htm)
- British Wind Energy Association (2008, Nov. 15). "Operational wind farms," [Online]. Available: http://www.(bwea.com/ukwed/ operational.asp)

425

464

- [3] P. J. Tavner, G. J. W. V. Bussel, and F. Spinato, "Machine and converter reliabilities in wind turbines," in *Proc. 3rd Power Electron. Mach. Drives Conf. (PEMD 2006)*, Cork, Ireland, Mar., pp. 127–130.
- [4] E. Echavarria, B. Hahn, G. J. W. van Bussel, and T. Tomiyama, "Reliability
 of wind turbine technology through time," *Trans. ASME (J. Sol. Eng.)*,
 vol. 130, no. 3, pp. 0310051–0310058, Aug. 2008.
- [5] D. McMillan and G. W. Ault, "Condition monitoring benefit for wind turbines: Sensitivity to operational parameters," *IET Renewable Power Generation*, vol. 2, no. 1, pp. 60–72, Mar. 2008.
- [6] H. Polinder, F. F. A. Van Der Pijl, G. de Vilder, and P. J. Tavner, "Comparison of direct-drive and geared generator concepts for wind turbines," *IEEE Trans. Energy Convers.*, vol. 21, no. 3, pp. 725–733, Sep. 2006.
- P. J. Tavner, J. Xiang, and F. Spinato, "Reliability analysis for wind turbines," *Wind Energy*, vol. 10, no. 1, pp. 1–18, Jun. 2007.
- [8] R. P. Hoskins, G. Strbac, and A. T. Brint, "Modelling the degradation of condition indices," *IEE Proc. - Generation, Transmiss., Distrib.*, vol. 146, no. 4, pp. 386–392, Jul. 2002.
- [9] F. C. Sayas and R. N. Allan, "Generation availability assessment of wind farms," *IEE Proc.- Generation, Transmiss., Distrib.*, vol. 143, no. 5, pp. 507–518, Sep. 1996.
- 492 [10] M. Marseguerra, E. Zio, and L. Podofillini, "Condition-based maintenance
 493 optimization by means of genetic algorithms and Monte Carlo simulation,"
 494 *Rel. Eng. Syst. Safety*, vol. 77, no. 2, pp. 151–165, Jul. 2002.
- [11] N. B. Negra, O. Holmstrom, B. Bak-Jensen, and P. Sorensen, "Aspects
 of relevance in offshore wind farm reliability assessment," *IEEE Trans. Energy Convers.*, vol. 22, no. 1, pp. 159–166, Mar. 2007.
- 498 [12] Vestas (2008, May 1). "V80–2.0 MW Versatile Megawattage," Manufacturers Data Sheet. [Online]. Available: http://www.vestas.com/en/windpower-solutions/wind-turbines/2.0-mw.aspx
- J. Ribrant and L. Bertling, "Survey of failures in wind power systems with
 focus on Swedish wind power plants during 1997–2005," *IEEE Trans. Energy Convers.*, vol. 22, no. 1, pp. 167–173, Mar. 2007.
- [14] J. A. Andrawus, J. Watson, M. Kishk, and A. Adam, "The selection of
 a suitable maintenance strategy for wind turbines," *Wind Eng.*, vol. 30,
 no. 6, pp. 471–486, Dec. 2006.
- 507 [15] Anon "Expansion based on rental," *Cranes Access*, vol. 8, no. 7, pp. 24–
 508 25, Oct. 2006.
- 509 [16] F90SQL, [Online]. Available: http://www.(http://www.canaimasoft.com/
 510 f90sql/)
- 511 [17] Enercon GmbH E-82 2008, Dec. 1). [Online]. Available: http://
 512 www.(http://www.enercon.de/en/_home.htm)
- 513 [18] K. Larsen, "Making wind more efficient," *Renew. Energy Focus*, pp. 40–
 514 42, Nov./Dec. 2008.



David McMillan (M'06) received the B.Eng. de-
gree in electronics and electrical engineering from the
University of Glasgow, Glasgow, Scotland, U.K., in
2002, and the M.Sc. and Ph.D. degrees from the Uni-
versity of Strathclyde, Glasgow, in 2005 and 2008,
519
respectively.517
518

From 2002 to 2004, he was a Manufacturing Engineer with Methode Electronics, Burnley. He is currently a Research Fellow at the Institute for Energy and Environment, University of Strathclyde. His research interests include techno-economic analysis of 525

526

527

energy generation and infrastructure and storage technologies.



Graham W. Ault (S'96–M'00) received the B.Eng.528degree in electrical and mechanical engineering, and529the Ph.D. degree in electrical power systems from the530University of Strathclyde, Glasgow, Scotland, U.K.,531in 1993 and 2000, respectively.532

He is currently a Reader with the University of 533 Strathclyde, where he has been engaged in several 534 aspects of distributed and renewable generation, asset 535 management, and power system planning and development. He has been actively involved in national and 537 international initiatives to advance the development 538

of the renewable energy industry through the Institution of Engineering and Technology, IEEE, Congrès International des Réseaux Electriques de Distribution or International Conference on Electricity Distribution, and International Energy Agency. 542

	QUERIES	544
Q1:	Author: Please check the edit changes in this sentence for correctness.	545
Q2.	Author: Please check the edit changes in this sentence for correctness.	546
Q3.	Author: Please provide year in Ref. [16].	547
Q4.	Author: Please provide complete bibliographic details in Ref. [18].	548



1

2

З

4

01

18

Techno-Economic Comparison of Operational Aspects for Direct Drive and Gearbox-Driven Wind Turbines

David McMillan, Member, IEEE, and Graham W. Ault, Member, IEEE

5 Abstract—The majority of wind turbines currently in operation have the conventional Danish concept design-that is, the three-6 7 bladed rotor of such turbines is indirectly coupled with an electrical generator via a gearbox. Recent technological developments have 8 enabled direct drive wind turbines to become economically feasible. 9 Potentially, direct drive wind turbines may enjoy higher levels of 10 11 availability due to the removal of the gearbox from the design. However, this is only a theory: so far not substantiated by detailed 12 analytic calculation. By providing such a calculation, this paper 13 14 enables us to quantitatively evaluate technical and economic merits of direct drive and gearbox-driven wind turbines. 15

16 *Index Terms*—Markov chain, operational comparison, reliabil-17 ity, wind turbines.



I. INTRODUCTION

ORLDWIDE installed capacity of wind generation is 19 growing significantly and is likely to continue to in-20 crease in the future. The twin policy objectives of energy secu-21 rity and climate change mitigation have resulted in economic 22 incentives, which in turn, have driven investment in wind energy. 23 Taking the U.K. as an example, Fig. 1 shows how the installed 24 capacity has grown since 2005-by the end of 2008, the installed 25 capacity broke through the 3 GW barrier [1]. This 3 GW capac-26 ity consists of 2276 individual wind turbines (WTs) [2], the vast 27 majority of which are conventional Danish concept, gearbox-28 driven machines. However, recent technical strides have enabled 29 direct drive machines to become economically feasible. 30

Since a gearbox is not included in the direct drive concept, 31 it is clear that the reliability and availability of the WT will 32 improve-if it can be assumed that all other factors remain 33 unchanged. On the other hand, it has been reported in the paper 34 that failure rates of electrical components and generators of 35 direct drive wind turbines are significantly higher than those of 36 gearbox-driven equivalents [3], [4]. The purpose of this paper is 37 to establish if there is a technical and/ or economic advantage in 38 39 deploying direct drive wind turbines instead of gearbox-driven

Manuscript received December 5, 2008; revised. This work was fully supported by the United Kingdom Engineering and Physical Sciences Research Council under the Prosen project Grant EP/C547594/1. Paper no. TEC-00483-2008.

D. McMillan is with the Institute for Energy and Environment, University of Strathclyde, Glasgow G1 1XQ, Scotland, U.K. (e-mail: dmcmillan@ eee.strath.ac.uk).

G. W. Ault is with the University of Strathclyde, Glasgow G1 1XQ, Scotland, U.K. (e-mail: dmcmillan@eee.strath.ac.uk).

Color versions of one or more of the figures in this paper are available online at http://ieeexplore.ieee.org.

Digital Object Identifier 10.1109/TEC.2009.2032596

Fig. 1. Recent growth in U.K. wind generation capacity [2].

machines. This analysis is based on quantitative modeling of the operation, failure, and maintenance of wind turbine units as proposed in [5]. Such an operational comparison of different wind turbine concepts is not available in the existing literature. 43

II. COMPARISON OF CONCEPTS 44

45

A. Conventional Danish Concept—Gearbox Driven

The conventional Danish concept comprises a three-bladed 46 upwind rotor, which revolves on the horizontal axis (sometimes 47 called horizontal axis wind turbine, HAWT). The coupling be-48 tween rotor and electrical generator is indirect and is achieved 49 via a gearbox in order to increase the rotational speed to a level 50 that can drive a relatively small-diameter, lightweight induction 51 generator. A conceptual view of the energy conversion process 52 for such a typical modern wind turbine is outlined in Fig. 2. 53

The whole wind turbine assembly rotates into the prevalent wind direction on its vertical axis by means of an electromechanical yaw system. Once facing into the wind, control of the mechanical input power is achieved either by aerodynamic design of the rotor (stall control) or by actively changing the angle of attack of the rotor blades to the wind (pitch control) via electrical motors or hydraulics.

The electrical configuration of Danish concept WTs is influ-61 enced by mechanical aspects, as one main objective of the WT 62 mechanical design is to minimize the weight at the top of the 63 tower, where the nacelle (containing the generator) is located 64 in modern HAWTs. This means the generator has to be as light 65 as possible and must have a relatively small physical footprint. 66 For this reason, induction generators are employed: induction 67 generators have the added advantage of being more robust than 68



Fig. 2. Process diagram for gearbox-driven and direct drive wind turbines.

synchronous generators and tend to have fewer electrical faults.
However, due to the low rotational speed of the wind turbine
rotor, a gearbox has to be used to increase the rotation from tens
of revolutions per minute at the gearbox input to thousands at
the output. The primary reason for this is the low number of
induction generator poles.

The older Danish concept WTs that operated at fixed speed 75 76 employed squirrel cage induction generators; however, newer 77 variable speed technology has resulted in a switch to doubly fed induction generators (DFIG), which are now the dominant wind 78 turbine generator configuration. The reason for the dominance 79 of this configuration is that it represents a good compromise 80 between economy and performance. It is relatively economic 81 82 because it has only a partial electronic converter rated at $\sim 30\%$ of the generator output [6], not a full converter. 83

84 B. Direct Drive Concept

In a direct drive WT, the main rotor is coupled to the generator 85 86 input shaft, eliminating the need for a gearbox in the design. In 87 order to generate power at such a low rotation speed, the gen-88 erator has to have many pole pairs, and usually a synchronous generator is employed. This implies much greater dimensions 89 and weight as compared with an induction generator. In addi-90 tion, a fully rated electronic power converter is required, which 91 92 increases the cost of the system.

93 C. Types of Comparison

There are several examples in the published literature where a comparison is made between the two concepts. For example, Tavner *et al.* [3] focused on how the configuration of the WT generator and converter in different design concepts affected overall WT reliability. The data utilized by the authors had enough detail to enable a direct reliability comparison of 99 three WT concepts: fixed speed with gearbox, variable speed 100 with gearbox, and variable speed direct drive (no gearbox: syn-101 chronous generator). The main conclusion was that direct drive 102 systems are less reliable than models with a gearbox because 103 the potential increase in reliability due to elimination of gear-104 box failures is cancelled out by increased generator, inverter, 105 and electrical system failures. 106

Interestingly, the authors recognized that overall availability 107 would also be affected by component repair times: In this sense 108 direct drive systems may have an advantage, as mean time to 109 repair (MTTR) for a gearbox is likely to be very much more 110 than MTTR for an electronics subassembly. As yet, no other 111 research has addressed this operational comparison of the two concepts. 113

Echavarria et al. [4] analyzed a similar dataset, which pro-114 vides some highly relevant information regarding the reliability 115 of the two WT concepts. In particular, the data suggest that gen-116 erator failures in direct drive WTs are roughly two times the 117 gearbox-driven equivalent (0.22 failures per annum compared 118 with 0.12 suggested by Tavner et al. [7]). Similarly, power elec-119 tronics failures in direct drive synchronous machines are quan-120 tified as 1.03 failures per annum compared to 0.661 suggested 121 in [7] for the induction machine equivalent. 122

Polinder et al. [6] examine direct drive and gearbox-driven 123 WT concepts from the viewpoint of design and economic per-124 formance. The authors define a typical Danish concept WT with 125 a three-stage gearbox (3GDFIG) and a direct drive machine cou-126 pled to a synchronous generator (DDSG). Three other concepts 127 are also defined (DFIG with a single-stage gearbox, perma-128 nent magnet direct drive, and permanent magnet single-stage 129 gearbox) but these are not considered in this paper due to the 130 fact that they are not currently deployed in significant numbers. 131



Fig. 3. Markov chain of key wind turbine components. Bold arrows and boxes indicate direct drive system.

The authors again highlight the need for further work to better
understand the reliability and availability benefits of adopting
different WT design concepts—a requirement which this paper
aims to meet.

136

III. MODELING OF WIND TURBINE CONCEPTS

137 A. Physical Modeling of Wind Turbine Components

In order to build an accurate operational model, the key physical components of the WT must be identified and a suitable mathematical representation decided upon. It was reported in [5] that using a combination of failure rate data, downtime estimates, and expert opinion, the key components of a gearboxdriven WT could be identified as follows:

- 144 1) gearbox (GBX);
- 145 2) generator (GEN);
- 146 3) rotor blades (ROT); and
- 147 4) inverter, electronics, and control (ELE).

In terms of the mathematical representation, a Markov chain 148 solved via Monte Carlo simulation (MCS) was identified in pre-149 vious studies as a suitable model framework [5]. The Markov 150 chain representation has been very successfully applied to power 151 systems infrastructure deterioration and failure modeling, in-152 cluding wind turbines [8], [9]. The main problem with these an-153 alytically solved models is that the introduction of constraints, 154 such as weather-constrained maintenance, makes obtaining a so-155 lution rather difficult. Although not computationally efficient, 156 MCS methods overcome this difficulty and have been applied 157 to similar problems in the past [10], [11]. Since the problem 158 considered is essentially a planning problem, the time required 159 to get the solution is of little importance. 160

It has been assumed that three states are sufficient to capture 161 the deterioration and failure processes of the GBX, GEN, and 162 ROT. ELE failures are assumed to be instantaneous and, there-163 fore, require only binary representation. When all four com-164 ponents are modeled in a single Markov state-space, the total 165 number of states is 54. This is cut down to 28 by assuming that 166 degradation and failure events of different WT components can-167 not happen concurrently. Furthermore, for a GBX, GEN, or ROT 168 failure to occur, the system must transit through the deteriorated 169 (intermediate) state before outright failure. 170

The possible Markov states and transitions for the overall WT 171 system are visualized in Fig. 3. The three possible deterioration 172 levels are indicated as fully up (U), deteriorated (!), and down 173 (D). 174

All 28 states and transitions are possible for the case of 175 gearbox-driven WTs. The obvious physical difference when 176 modeling a direct drive WT is that there are no gearbox states. 177 Thus, the bold arrows in Fig. 3 refer to the transitions and states 178 that represent direct drive WTs, as a 12-state subset. 179

The arrows in Fig. 3 represent transition probabilities (e.g., 180 probability of transition from state a to state b is $p_{a,b}$), whose 181 magnitude must be estimated. These probabilities depend only 182 on the current state of the system (s) at current time t_k . 183 Equation 1 expresses this "memory-less" property of a Markov 184 chain 185

$$p_{a,b} = p(s_b, t_{k+1} | s_a, t_k)$$
 $k = 1, 2, 3, \dots$ (1)

Furthermore, the transition probabilities are constant in time: 186 This is the "time-stationary" property. The magnitude of all 187 transition probabilities from one state (*a*) to all others in the 188

 $p_{1,1} = 1 - p_{1,2}$ $p_{2,2} = 1 - p_{2,n}$ Estimated from sensitivity analysis $p_{2,n}$ Estimated from expert judgement
each component failure $p_{n,n} = 1$ Overall reliability is known
from 'target' failure rates

Fig. 4. Parameter estimation for Markov chain.

system must sum to unity. This is shown in (2)

$$\sum_{b} p_{a,b} = 1 \qquad a = 1 \dots n \tag{2}$$

where *n* is the total number of system states. For convenience, the transition probabilities for the whole system are expressed in a transition probability matrix (TPM). For the case of the system in Fig. 3, the TPM is shown in (3). Note that only the possible transitions (indicated in Fig. 3. with arrows) need to be estimated—all other probabilities are equal to zero

$$\text{TPM} = \begin{bmatrix} p_{1,1} & p_{1,2} & p_{1,28} \\ p_{2,1} & p_{2,2} & \ddots \\ \vdots & \vdots & \ddots & \vdots \\ p_{28,1} & \vdots & p_{28,28} \end{bmatrix}.$$
 (3)

The TPM values are estimated based on the partial information available, as illustrated in Fig. 4. A good estimate of the overall reliability ("target" failure rate) is known from the work of Tavner *et al.* [7].

Similarly, downtime estimates can be made for the outage of the key components [5]. The probability of an outright failure in a deteriorated condition $(p_{2,n})$ can be estimated based on expert opinion of times to failure. The remaining parameters can then be estimated by conducting sensitivity analyses (previously reported in [5]).

206 B. Data for Physical Model

The study of Tavner *et al.* [7] provided estimates of component failure rates based on populations of Danish and German WTs. The German population was larger (over 4,000 machines) and the population consisted of more modern WTs. Therefore, the German figures are used to fit the gearbox-drive WT physical model.

On the other hand, Echavarria *et al.* [4] suggest modifications
to the GEN and ELE failure rates for direct drive machines.
Taking this into account, the target failure rates for both WT
concepts can be visualized in Fig. 5.

Downtimes for the failure types are as follows: GBX—30 days, GEN—21 days, ELE—1 day, and ROT—30 days. They are based on domain knowledge elicited in [5]. Using this



Fig. 5. Reliability for gearbox-driven and direct drive wind turbines.

information, the TPM parameters were estimated based on the 220 iterative procedure devised in [5]. The procedure is based on 221 sensitivity analysis estimation of the unknown parameter $p_{1,2}$ 222 (see Fig. 4). 223

The Markov chain has been defined for both gearbox-driven 224 and direct drive concepts. In the following sections, other aspects of operational modeling—which are common to both 226 concepts—are discussed. 227

C. Energy Yield Modeling

A Annual Failure Rate (number of

failures per annum)

There are two main components to the energy yield model. 229 These are the wind speed model and the power curve model. The 230 wind speed (WS) model previously used by the authors [5] was 231 based on a single parameter autoregressive process, or AR(1). 232 This is displayed in (4), where μ is the mean of a wind speed 233 time series, ϕ is the autoregressive parameter and the process is 234 driven by a Gaussian white noise function ε_t . 235

$$WS_t - \mu = \phi (WS_{t-1} - \mu) + \varepsilon_t .$$
(4)

The dataset used to fit the model was sourced from a supervisory control and data acquisition (SCADA) system of a 237 U.K. wind farm. Estimation of ϕ and ε_t was achieved by linear least squares, while classification of the model was based 239 on inspection of the autocorrelation and partial-autocorrelation 240 functions. 241

The power curve model is based on a manufacturers' 242 datasheet for a 2 MW WT [12], which is sampled and the theoretical equation for the power (P) in the wind (5) is matched to the data samples by modeling the coefficient of performance C_p 246

$$P = \frac{1}{2}\rho\pi r^2 v^3 (\times C_p).$$
⁽⁵⁾

In (5), ρ is air density (kg/m³), *r* is the rotor radius (m), and *v* 247 is air velocity through the WT rotor (m/s). The re-created power 248 curve is shown in Fig. 6 and has cut in, rated, and cut out wind 249 speeds of 4, 14, and 25 m/s, respectively. It is assumed that both 250 WT concepts adhere to the same power curve. 251





Fig. 6. Power curve for 2 MW wind turbine [12].

252 D. Maintenance Modeling

It is assumed that a six-monthly periodic maintenance plan 253 is adopted for both WT concepts. The assumptions in the main-254 tenance model are that maintenance actions restore the WT to 255 256 the fully up state (state 1 in Fig. 3) and that each maintenance visit involves a one-day outage. The model can easily accom-257 modate condition-based maintenance (CBM), but since mainte-258 nance paradigms are not the focus of this paper, this possibility 259 is neglected. 260

261 If a component failure occurs, a maintenance team is dispatched immediately. There is a probability that a component 262 replacement is necessary (replacement factor $\beta = 0.6$) or that it 263 can be repaired $(1-\beta = 0.4)$. This is based on an analysis by 264 Ribrant and Bertling [13] who highlighted that around 60% of 265 gearbox failures require a replacement rather than a repair ac-266 tion. Similar figures have not been published for the other WT 267 components; therefore, due to the lack of data, they are assumed 268 to have the same probabilities of repair and replacement as the 269 gearbox. 270

271 If the component can be repaired, it is restored instantaneously to a functional state. If a replacement is required, down-272 time lasts as follows: GBX-30 days, GEN-21 days, ELE-1 273 day, and ROT-30 days. This is based on the experience of a 274 wind farm operator. Furthermore, maintenance actions are con-275 strained by wind speed as in [5]. This means that nacelle-related 276 277 replacements need wind speed conditions of less than 10 m/s, while rotor maintenance cannot be conducted in wind speeds 278 over 7 m/s. 279

280 E. Costs and Revenue

It is of interest to compare the economic merits of the two WT
configurations. Therefore, a cost model has been built which
generates revenue from energy yield and incurs maintenance
and replacement costs.

Polinder *et al.* provided costs for gearbox-driven and direct drive wind turbine components rated at 3 MW [6]. Figures can be derived for 2 MW machines of both types assuming that the cost varies linearly with the rating. These costs are provided in Table I. The rotor cost was not provided in [6] and so the value derived previously by the authors of this paper is adopted [5]. In the case of a component replacement, the full cost in Table I

 TABLE I

 COMPONENT COSTS FOR WIND TURBINE CONCEPTS

Component	Gearbox driven	Direct drive
Component	Cost £	
GBX	121,733	N/A
GEN	177,066	313,740
ROT	210,000	210,000
ELE	22,133	66,400

is incurred. In the case of a repair, it is assumed that only 10% 292 of this cost is incurred (repair cost factor $\alpha = 0.1$). 293

Besides the costs of the components themselves, the cost of 294 labor and equipment hire has been included. Andrawus [14] 295 showed that skilled labor for WT repairs costs around £50/h. It 296 has been assumed that three crew working an 8 h shift consti-297 tute one maintenance action. Therefore, the cost of labor (C_{LAB}) 298 is $\pounds 1200$ per action. Similarly, hire rates for telescopic cranes 299 $(C_{\rm EQ})$ needed to perform nacelle component lifting operations 300 have been quantified by industry sources [15] as $\pounds 1500$ per 301 week. Lost revenue due to downtime is also taken into account 302 (R_{LOST}) —this is wind speed-dependent. These costs are com-303 bined with the component costs (see Table I) to calculate re-304 placement (6), repair, (7) and O&M cost (8) 305

$$C_{\text{replace}} = \beta \lambda \left(C_{\text{CAP}} + C_{\text{LAB}} + C_{\text{EQ}} + R_{\text{LOST}} \right)$$
(6)

$$C_{\text{repair}} = (1 - \beta) \lambda \times (\alpha C_{\text{CAP}} + C_{\text{LAB}} + C_{\text{EQ}} + R_{\text{LOST}})$$

$$C_{\rm O\&M} = C_{\rm replace} + C_{\rm repair} \,. \tag{8}$$

The revenue model for the WT is based on the energy yield in 306 each one-day simulation interval. Using equations (4) and (5), 307 this energy yield *Y* can be calculated as the power (see Fig. 6) 308 multiplied by a time interval Δt . The energy yielded in a year is 309 then calculated by summing the output over all individual days 310 in the year. 311

The revenue stream *R* can then be calculated by applying 312 equation (9). MP represents the market price for electricity and 313 renewable obligation certificates (ROCs). For this paper, MP_{elec} 314 and MP_{ROC} are set to £36 and £40 per MWh, respectively. 315 Although in reality, electricity and ROC prices fluctuate, the annual mean is adequately represented by the figures presented 317 as 318

$$R = Y \left(\mathrm{MP}_{\mathrm{elec}} + \mathrm{MP}_{\mathrm{ROC}} \right) - C_{\mathrm{O\&M}} \,. \tag{9}$$

It is important to note that any differences in yield between the 319 two WT concepts will be related to the reliability and downtime 320 (see Fig. 5 and Section III-D) of the two WT concepts, rather 321 than to the differences in the electrical design. This is because the 322 same power curve (Fig. 6) has been used for both WT concepts. 323

Polinder *et al.* showed that the theoretical difference in yield 324 between a 3 MW direct drive machine (DDSG) and a typical 325 3 MW DFIG (DFIG3G) is +150 MWh [6], if the detail of 326 the electrical machine design is taken into account. Assuming 327 this difference scales linearly with WT rating, it means that for 328 the 2 MW machines considered in this paper, the direct drive 329 machine yields roughly 100 MWh more per annum than the 330



Fig. 7. Availability comparison of wind turbine concepts.

DFIG. Applying (9) and neglecting the O&M cost, this translates to \pounds 7600 more revenue per annum for the direct drive machine. Therefore, the annual revenue for the direct drive machine has been boosted by \pounds 7600 per annum in the studies that follow.

335 F. Program Operation

The models outlined in this section were coded in FORTRAN 95, with the SCADA database interface for the wind speed model written using f90SQL [16]. The statistical programming language R was used to fit the wind speed model defined in (4). The resultant capacity factor of the wind turbine based on the simulated wind speed and power curve is just under 30%.

The confidence limit *L* of the simulation results can be measured by applying (10). Taking the Student-*t* distribution, and setting the level of confidence to 95%, it means that *L* can be specified, provided that the number of samples (*N*) and standard deviation (σ) of the quantity are known. *L* is shown in the results as confidence bands that specify the accuracy of the results

$$L = \pm \frac{2.045 \times \sigma}{\sqrt{N}}.$$
 (10)

IV. RESULTS

349 A. Operational Comparison of Concepts

348

Two comparisons are made in order to benchmark the operational merits of the two WT concepts: a technical comparison and an economic comparison. The first result in Fig. 7 compares the overall availability of the two concepts.

It can be seen that despite removing the gearbox from the design, the direct drive concept has similar overall availability to the gearbox-driven machine. Although the availability is marginally better for the case of the direct drive machine, the confidence limits show that this technical benefit is uncertain. It should be noted that grid availability is not included in this paper.

The second result, displayed in Fig. 8, shows the revenue generated (9) for both concepts. This shows that the gearboxdriven design has a much larger economic benefit than the direct drive concept.

The contribution to the revenue of increased energy yield due to avoidance of downtime is negligible in the case of Fig. 8 (direct drive machine avoids loss of \sim 6.35 MWh more energy



Fig. 8. Economic comparison of wind turbine concepts.



Fig. 9. Availability improvement of direct drive wind turbine as a function of generator reparability.

than gearbox-driven, economic benefit = $\pounds 482$ per annum). 368 Therefore, the large disparity in revenue ($\sim \pounds 44\ 000$ per annum) 369 must be due to incurred repair and replacement costs. The large 370 increase in cost and failure rate for the generator in particular 371 (see Table I and Fig. 5) appears to economically handicap the 372 direct drive concept. 373

B. Operational Impact of Generator Reparability 374

One possible explanation of the superior economic perfor-375 mance of the gearbox-drive concept is that a replacement factor 376 (β) of 0.6 per failure (see Section III-D) may represent a pes-377 simistic view of the "reparability" of a WT synchronous gener-378 ator. Indeed, it has been reported elsewhere [3] that the increase 379 in generator failure rate for the direct drive concept is related 380 to electrical failures rather than mechanical failures. Electrical 381 faults will be less likely to involve a complete component re-382 placement; therefore, the robustness of the conclusion drawn 383 from Fig. 8 is tested by modeling different levels of reparability 384 for the direct drive generator. 385

The replacement factor β was reduced from the base value of 386 0.6–0.1, as shown in Fig. 9. This figure shows that the operational availability of the direct drive concept WT can be significantly higher than the gearbox-driven WT, if a high proportion 389 of synchronous generator failures are minor electrical failures 390 rather than severe mechanical failures (e.g., bearing problems). 391



Fig. 10. Revenue increase of direct drive wind turbine as a function of generator reparability.



Fig. 11. Revenue increase of direct drive wind turbine as a function of component cost reduction.

The economic impact of this variation is illustrated in Fig. 10. 392 This figure shows that even for an optimistic scenario, the annual 393 revenue of the direct drive WT is still $\sim \pounds 20\ 000$ less than the 394 equivalent gearbox-driven WT. This aspect of WT component 395 reparability has not received much attention in the literature, but 396 397 Fig. 10 in particular shows that it is a significant factor when conducting operational modeling of WT concepts. More stud-398 ies of the type conducted by Ribrant and Bertling [13] will be 399 needed in order to better understand the reparability of different 400 WT components and their effect on operational metrics such as 401 402 availability and revenue. Analysis of WT failures in the con-403 text of repairs and replacements along with their probabilities and costs are crucial for a deep understanding of wind farm 404 operational issues. 405

C. Operational Impact of Component Cost Reduction 406

In the WT marketplace, there is currently one company that 407 builds 2MW direct drive machines on an industrial scale [17]; 408 however, other large WT manufacturers have identified direct 409 drive machines as an avenue for future production [18]. With 410 more players in the market, it may be possible to significantly 411 reduce the direct drive component cost through a refinement 412 of mass production manufacturing processes. Therefore, it is 413 of interest to review the effect on the economic case for direct 414 drive machines if component costs are lowered. Such a review is 415 provided in Fig. 11, where the GEN and ELE component costs 416 are reduced in 10% steps to 50% of the original Table I values. 417

The final result shows that if substantial cost reductions in 418 direct drive technology are achieved in the future, this measure 419 may be enough to make the technology cost-competitive with 420 DFIGs. However, very large cost reductions of 50%+ will be 421 required. At current prices, the economic argument for a switch 422 to direct drive technology, for the onshore conditions evaluated, 423 appears to be weak. 424

V. CONCLUSION

An operational comparison of direct drive and gearbox-driven 426 wind turbines has been presented in this paper. The results sug-427 gest that there may be a technical advantage in deploying direct 428 drive machines over more established gearbox-driven designs 429 (see Fig. 9). In all cases, the economic analysis shows that 430 gearbox-driven machines are still preferable, unless manufactur-431 ing costs of direct drive technology can be significantly reduced 432 (see Fig. 11). 433

There are some issues that need to be better understood in 434 order to make more precise comparisons of these technologies. 435 One is that the repair probability of the components needs to be 436 investigated, in a manner similar to the one presented in [13] 437 but for all WT components. The failure rate increase for a syn-438 chronous generator relative to an induction generator (reported 439 in [3], [4]) will be made up mainly of electrical-related failures 440 rather than mechanical failures. It would be interesting to see 441 what proportion of direct drive WT generator failures are low 442 downtime (e.g., 1–3 days) as opposed to a mechanical failure of 443 a rotating component, which in some cases could take as long as 444 60 days to replace [14]. Such an analysis would aid understand-445 ing of WT failure modes and make operational comparisons 446 more accurate. 447

This study was carried out for fairly typical onshore condi-448 tions, but the conclusions may be linked to the site conditions. 449 Direct drive machines are perceived by some manufacturers as 450 primarily an offshore technology [18]. By modeling the offshore 451 wind resource, logistics, increased downtimes, and offshore ac-452 cess constraints, it may be possible to determine if direct drive 453 machines would become more economically attractive in off-454 shore conditions than the analysis presented in this paper shows. 455 The conclusions of the results in this paper and any further anal-456 ysis as described will be of value to both manufacturers and 457 operators of wind turbines. 458

ACKNOWLEDGMENT

The authors would like to thank Y. Patel of Scottish 460 Power/Iberdrola and M. Smith of Macom Technologies for their 461 valuable input to this paper. The authors would also like to thank 462 Dr. D. Hill and the IEEE reviewer for their valuable comments. 463

REFERENCES

- [1] Scottish Power/Iberdrola press release (2008, Nov. 9). "Scottish 465 Power Renewables Windfarm Pushes Britain Past 3GW Wind En-466 ergy Landmark," [Online]. Available: http://www.(scottishpower.com/ 467 PressReleases_1774.htm) 468
- "Opera-[2] British Wind Energy Association (2008, Nov. 15). 469 tional wind farms," [Online]. Available: http://www.(bwea.com/ukwed/ 470 operational.asp) 471

7

425

464

- [3] P. J. Tavner, G. J. W. V. Bussel, and F. Spinato, "Machine and converter 472 473 reliabilities in wind turbines," in Proc. 3rd Power Electron. Mach. Drives 474 Conf. (PEMD 2006), Cork, Ireland, Mar., pp. 127-130.
- [4] E. Echavarria, B. Hahn, G. J. W. van Bussel, and T. Tomiyama, "Reliability 475 476 of wind turbine technology through time," Trans. ASME (J. Sol. Eng.), vol. 130, no. 3, pp. 0310051-0310058, Aug. 2008.
- 478 [5] D. McMillan and G. W. Ault, "Condition monitoring benefit for wind 479 turbines: Sensitivity to operational parameters," IET Renewable Power Generation, vol. 2, no. 1, pp. 60-72, Mar. 2008. 480
- 481 [6] H. Polinder, F. F. A. Van Der Pijl, G. de Vilder, and P. J. Tavner, "Comparison of direct-drive and geared generator concepts for wind turbines." 482 483 IEEE Trans. Energy Convers., vol. 21, no. 3, pp. 725-733, Sep. 2006.
- 484 P. J. Tavner, J. Xiang, and F. Spinato, "Reliability analysis for wind tur-[7] bines," Wind Energy, vol. 10, no. 1, pp. 1-18, Jun. 2007. 485
- 486 [8] R. P. Hoskins, G. Strbac, and A. T. Brint, "Modelling the degradation of condition indices," *IEE Proc.- Generation, Transmiss., Distrib.*, vol. 146, no. 4, pp. 386–392, Jul. 2002. 487 488
- 489 [9] F. C. Sayas and R. N. Allan, "Generation availability assessment of wind 490 farms," IEE Proc.- Generation, Transmiss., Distrib., vol. 143, no. 5, 491 pp. 507-518, Sep. 1996.
- 492 [10] M. Marseguerra, E. Zio, and L. Podofillini, "Condition-based maintenance 493 optimization by means of genetic algorithms and Monte Carlo simulation," 494 Rel. Eng. Syst. Safety, vol. 77, no. 2, pp. 151-165, Jul. 2002.
- [11] N. B. Negra, O. Holmstrom, B. Bak-Jensen, and P. Sorensen, "Aspects 495 496 of relevance in offshore wind farm reliability assessment," IEEE Trans. Energy Convers., vol. 22, no. 1, pp. 159-166, Mar. 2007. 497
- Vestas (2008, May 1). "V80–2.0 MW Versatile Megawattage," Manufac-498 [12] 499 turers Data Sheet. [Online]. Available: http://www.vestas.com/en/windpower-solutions/wind-turbines/2.0-mw.aspx 500
- [13] 501 J. Ribrant and L. Bertling, "Survey of failures in wind power systems with 502 focus on Swedish wind power plants during 1997-2005," IEEE Trans. 503 Energy Convers., vol. 22, no. 1, pp. 167-173, Mar. 2007.
- 504 [14] J. A. Andrawus, J. Watson, M. Kishk, and A. Adam, "The selection of a suitable maintenance strategy for wind turbines," Wind Eng., vol. 30, 505 no. 6, pp. 471-486, Dec. 2006. 506
- 507 [15] Anon "Expansion based on rental," Cranes Access, vol. 8, no. 7, pp. 24-508 25. Oct. 2006.
- 509 [16] F90SQL, [Online]. Available: http://www.(http://www.canaimasoft.com/ 510 f90sql/)
- 511 Enercon GmbH E-82 2008, Dec. 1). [Online]. Available: http:// [17] www.(http://www.enercon.de/en/_home.htm) 512
- K. Larsen, "Making wind more efficient," Renew. Energy Focus, pp. 40-513 [18] 514 42, Nov./Dec. 2008.



David McMillan (M'06) received the B.Eng. de-515 gree in electronics and electrical engineering from the 516 University of Glasgow, Glasgow, Scotland, U.K., in 517 2002, and the M.Sc. and Ph.D. degrees from the Uni-518 versity of Strathclyde, Glasgow, in 2005 and 2008, 519 respectively. 520

From 2002 to 2004, he was a Manufacturing En-521 gineer with Methode Electronics, Burnley. He is cur-522 rently a Research Fellow at the Institute for Energy 523 and Environment, University of Strathclyde. His re-524 search interests include techno-economic analysis of 525

526

527

energy generation and infrastructure and storage technologies.

Graham W. Ault (S'96–M'00) received the B.Eng. 528 degree in electrical and mechanical engineering, and 529 the Ph.D. degree in electrical power systems from the 530 University of Strathclyde, Glasgow, Scotland, U.K., 531 in 1993 and 2000, respectively. 532

He is currently a Reader with the University of 533 Strathclyde, where he has been engaged in several 534 aspects of distributed and renewable generation, asset 535 management, and power system planning and devel-536 opment. He has been actively involved in national and 537 international initiatives to advance the development 538

of the renewable energy industry through the Institution of Engineering and 539 Technology, IEEE, Congrès International des Réseaux Electriques de Distribu-540 tion or International Conference on Electricity Distribution, and International 541 Energy Agency. 542 543

	QUERIES	544
Q1:	Author: Please check the edit changes in this sentence for correctness.	545
Q2.	Author: Please check the edit changes in this sentence for correctness.	546
Q3.	Author: Please provide year in Ref. [16].	547
Q4.	Author: Please provide complete bibliographic details in Ref. [18].	548

